



# MADE EASY

India's Best Institute for IES, GATE & PSUs

## ESE 2019 : Mains Test Series

UPSC ENGINEERING SERVICES EXAMINATION

### Civil Engineering

#### Test-3: Strength of Materials

#### Transportation Engineering-1 + Surveying & Geology-1

#### Geo-technical & Foundation Engg-2 + Environmental Engg-2

Name : .....

Roll No : C E 1 9 M B D L A 8 3 8

#### Test Centres

- Delhi  Bhopal  Noida  Jaipur  Indore   
 Lucknow  Pune  Kolkata  Bhubaneswar  Patna   
 Hyderabad

#### Student's Signature

#### Instructions for Candidates

- Do furnish the appropriate details in the answer sheet (viz. Name & Roll No).
- Answer must be written in English only.
- Use only black/blue pen.
- The space limit for every part of the question is specified in this Question Cum Answer Booklet. Candidate should write the answer in the space provided.
- Any page or portion of the page left blank in the Question Cum Answer Booklet must be clearly struck off.
- Last two pages of this booklet are provided for rough work. Strike off these two pages after completion of the examination.

#### FOR OFFICE USE

Question No.	Marks Obtained
Section-A	
Q.1	60 - 3 = 57
Q.2	59 - 1 = 58
Q.3	
Q.4	60
Section-B	
Q.5	34
Q.6	32
Q.7	
Q.8	
<b>Total Marks Obtained</b>	<b>240 - 4 = 236</b>

Signature of Evaluator

Cross Checked by

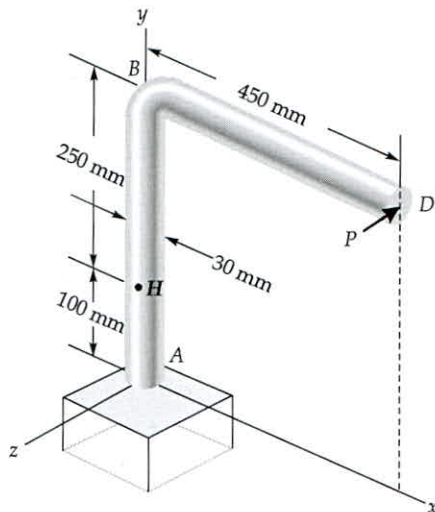
*Good presentation  
Attempt & improve theory portion also.*



**Section A : Strength of Materials**

1 (a) A single horizontal force  $P$  of magnitude 600 N is applied to end  $D$  of lever  $ABD$ . The diameter of lever  $ABD$  is 30 mm. Determine :

- (i) The normal and shearing stress on an element located at point  $H$  having sides parallel to  $x$  and  $y$  axis.
- (ii) The directions of principal planes and principal stresses at point  $H$ .



[12 marks]

Shear force at  $H$   
 $= 600 \text{ N}$

Bending moment  $= 600 \times 250$   
 $M_H = 150000 \text{ N}\cdot\text{mm}$

Torsional moment  
 $= 600 \times 450 = 270000 \text{ N}\cdot\text{mm}$

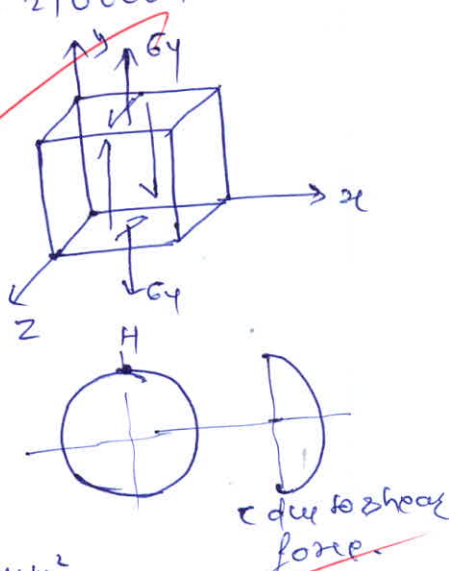
Element at  $H$

$\tau$  due to shear force  
 $= 0 \text{ N/mm}^2$

$\tau$  due to Torsional moment

$$= \frac{16T}{\pi d^3}$$

$$= \frac{16 \times 270,000}{\pi \times (30)^3} = 50.93 \text{ N/mm}^2$$

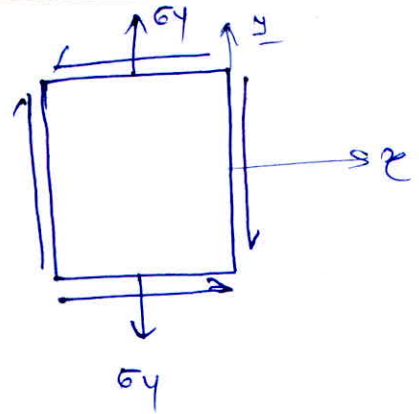
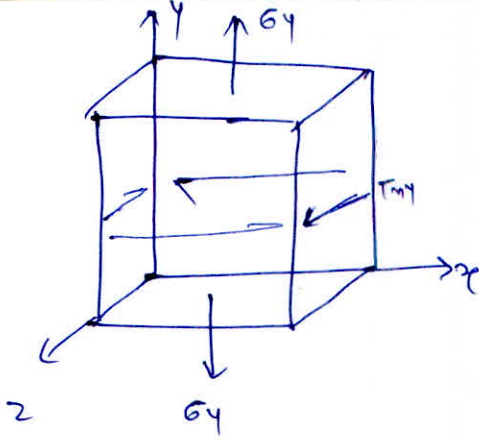


bending stress at  $x$

$$= \frac{M}{Z} = \frac{150000}{\frac{\pi \times 30^3}{32}} = 56.588 \text{ N/mm}^2$$

Shear stress = 50.93 N/mm<sup>2</sup>

(11)



$$\tau_{xy} = 50.93 \text{ N/mm}^2$$

$$\sigma_y = 56.588 \text{ N/mm}^2$$

principle stresses  $\sigma_{p1/p2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$

$$= \frac{\sigma + 56.588}{2} \pm \sqrt{\left(\frac{56.588}{2}\right)^2 + (50.93)^2}$$

$$\sigma_{p1} = 86.55 \text{ N/mm}^2$$

$$\sigma_{p2} = -29.96 \text{ N/mm}^2$$

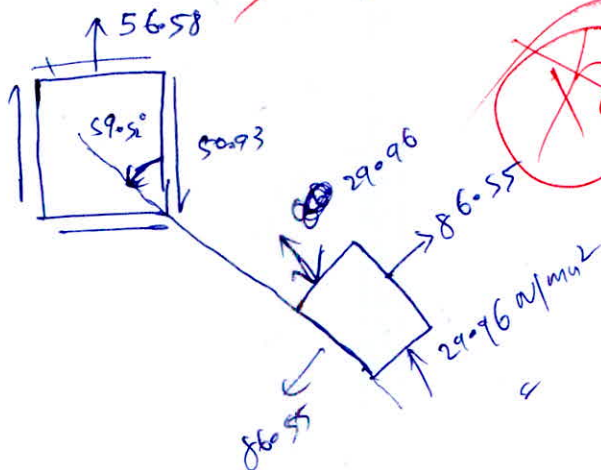
direction of principle planes:

$$\tan 2\theta = \frac{2\tau_{xy}}{\sigma_x - \sigma_y}$$

$$\Rightarrow \frac{2(-50.93)}{56.588 - 0}$$

$$\left[ \theta = 59.52^\circ \right] \text{ anticlockwise from } y \text{ axis.}$$

$$\text{or } 104.52^\circ$$



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- (b) A steel specimen is subjected to the following principal stresses: (i)  $120 \text{ N/mm}^2$  tensile (ii)  $60 \text{ N/mm}^2$  tensile and (iii)  $30 \text{ N/mm}^2$  compressive. The proportionality limit for the steel specimen is  $250 \text{ N/mm}^2$ . Find the factor of safety according to
- (i) Maximum shear stress theory.      (ii) Maximum principal strain theory.  
(iii) Maximum strain energy theory.

Take Poisson's ratio = 0.3

[12 marks]

(1) Maximum stress

$$= \max \left[ \frac{120-60}{2}, \frac{120-(-30)}{2}, \frac{60-(-30)}{2} \right]$$

$$= \max [30, 75, 45]$$

$$\tau_{\max} = 75 \text{ N/mm}^2$$

as per maximum stress theory.

$$75 \leq \left( \frac{250}{2 \cdot \text{FOS}} \right)$$

$$\text{FOS} \leq \frac{125}{75}$$

$$\boxed{\text{FOS} \leq 1.67}$$

(1) Maximum principle strain theory.

$$\frac{\sigma_1}{E} - \nu \frac{\sigma_2}{E} - \frac{\nu \sigma_3}{E} \leq \left( \frac{\sigma_y}{E \cdot \text{FOS}} \right)$$

$$\frac{120}{E} - \frac{60 \times 0.3}{E} - \frac{0.3 \times (-30)}{E} \leq \frac{250}{E \cdot \text{FOS}}$$

$$\frac{120 - 18 + 9}{E} \leq \frac{250}{E \cdot \text{FOS}}$$

$$\text{FOS} \leq \frac{250}{111}$$

$$\boxed{\text{FOS} \leq 2.252}$$

(iii)

As per maximum strain energy theory

$$\frac{1}{2E} \left[ G_1^2 + G_2^2 + G_3^2 - 2\mu(G_1G_2 + G_2G_3 + G_3G_1) \right] \leq \frac{1}{2E} \left( \frac{\sigma_y}{\text{FOS}} \right)^2$$

$$\Rightarrow (120)^2 + 60^2 + 30^2 - 2(0.3) \left[ (120 \times 60) + (60 \times -30) + (-30 \times 120) \right] \leq \left( \frac{250}{\text{FOS}} \right)^2$$

$$\Rightarrow 17820 \leq \left( \frac{250}{\text{FOS}} \right)^2$$

$$\text{FOS} \leq \frac{250}{\sqrt{17820}}$$

$$\boxed{\text{FOS} \leq 1.972}$$

- 1.1 (c) A uniformly tapering vertical post of height  $H$  having a diameter  $D$  at the base and a diameter  $d$  at the top is fixed at its base. A horizontal force  $P$  is applied at the top of the post. Determine the maximum bending stress for the post and state where it occurs.

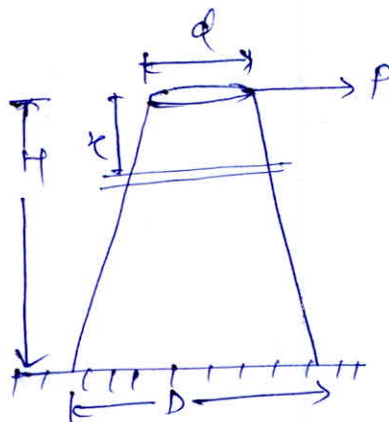
[12 marks]

Dia at  $x$  from top

$$d_x = d + \left(\frac{D-d}{H}\right) \cdot x$$

put  $\frac{D-d}{H} = k$

$$d_x = d + kx$$



Bending stress at  $x$

is given by.

$$\sigma_x = \frac{M \cdot r}{I}$$

$$\sigma_x = \frac{P \cdot x}{\frac{\pi (D_x)^3}{32}} = \frac{32 P \cdot x}{\pi (d + kx)^3}$$

for  $\sigma_x$  to be maximum

$$\frac{d\sigma_x}{dx} = 0$$

$$\frac{d}{dx} \left[ \frac{32 P x}{\pi (d + kx)^3} \right] = 0$$

$$\frac{d}{dx} \frac{P x}{(d + kx)^3} = 0$$

$$\frac{(d + kx)^3 - x \cdot 3(d + kx)^2 \cdot k}{(d + kx)^6} = 0$$

$$(d + kx)^3 - 3kx(d + kx)^2 = 0$$

$$(d + kx)^2 [d + kx - 3kx] = 0$$

$$d - 2kx = 0$$

$$d = 2 \cdot k \cdot x$$

$$x = \frac{d}{2k}$$

$$= \frac{d}{2 \left(\frac{D-d}{H}\right)} = \frac{d \cdot H}{2(D-d)}$$

Bending maximum at stress

$$x = \frac{d \cdot H}{2(D-d)}$$

Maximum Bending stress

$$\sigma_x = \frac{32 P \cdot x}{\pi (d+kx)^3}$$

$$\Rightarrow d + kx = d + \left(\frac{D-d}{H}\right) \times \frac{d \cdot H}{2(D-d)}$$

$$\Rightarrow d + \frac{d}{2} = \left(\frac{3d}{2}\right)$$

$$\sigma_x = \frac{32 P \cdot \left[\frac{dH}{2(D-d)}\right]}{\pi \left(\frac{3d}{2}\right)^3}$$

$$\sigma_x = \frac{32 P \cdot dH \cdot 8}{2(D-d) \times \pi \times 27 d^3}$$

$$\sigma_x = \frac{128 P H}{27 (D-d) \pi d^2}$$

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- Q.1 (d) A 10 mm diameter mild steel bar of length 1.50 metre is stressed by a weight of 120 N dropping freely through 20 mm before commencing to stretch the bar. Find the maximum instantaneous stress and the elongation produced in the bar.

[Take  $E = 2 \times 10^5 \text{ N/mm}^2$ ]

[12 marks]

$\therefore$  weight 120 N falling through height of 20 mm.

From energy conservation principle:

$$mg(H + \Delta H) = \frac{\sigma^2}{2E} \cdot A \cdot l$$

$$\Rightarrow 120 \left[ \frac{20}{1000} + \frac{\sigma \times 1.5}{E} \right] = \frac{\sigma^2}{2 \times (2 \times 10^{11})} \times \frac{\pi}{4} \times \left( \frac{10}{1000} \right)^2 \times 1.5$$

$$\Rightarrow \sigma = 91811150.17 \text{ N/m}^2$$

$$\boxed{\sigma = 91.81 \text{ N/mm}^2}$$

Elongation produced

$$= \frac{\sigma \cdot l}{E}$$

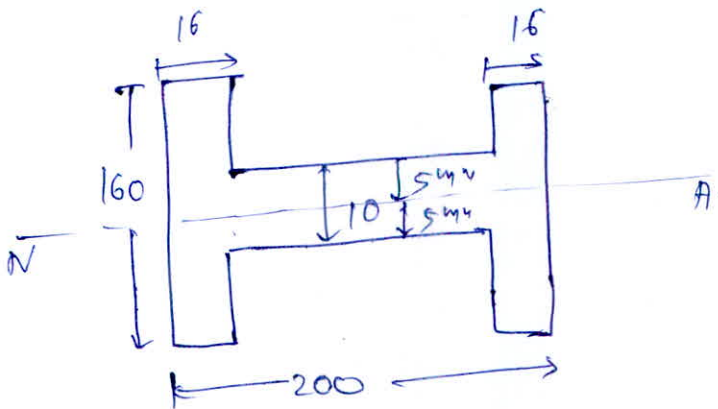
$$\Rightarrow \frac{91.81 \times 1.5 \times 10^3}{2 \times 10^5} \text{ mm}$$

$$= \boxed{0.688 \text{ mm}}$$

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Q.1 (e) A steel beam of I-section, 200 mm deep and 160 mm wide has 16 mm thick flanges and 10 mm thick web. The beam is subjected to a shear force of 200 kN. Draw the shear stress distribution, if the web of the beam is kept horizontal.

[12 marks]



Beam subjected to shear force = 200 kN.

take  $y$  distance from NA.

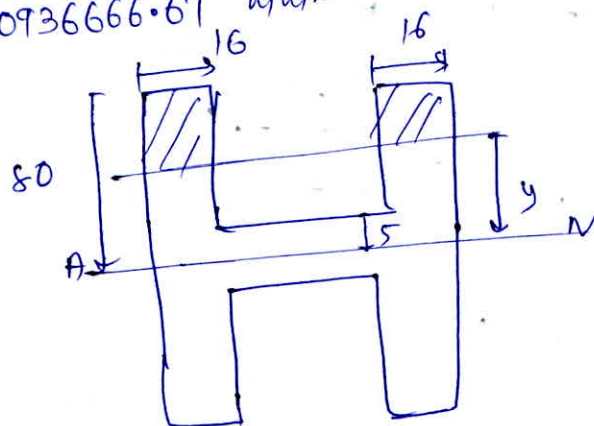
such that  $y > 50$  mm

then.

$$\tau = \frac{VAy}{Ib}$$

$$I = \frac{200 \times 160^3}{12} - 2 \left[ \frac{168 \times 75^3}{12} + \frac{168 \times 75 \times (75-50)^2}{75} \right]$$

$$I = 10936666.67 \text{ mm}^4$$



$$A \cdot \bar{y} = \frac{(80-y) \times 16 \times 2}{2} \times \left[ \frac{80+y}{2} \right]$$

$$= 16(80^2 - y^2)$$

$$\left[ \tau_y = \frac{200 \times 10^3 \times 16(80^2 - y^2)}{10936666.67 \times 32} \right]$$

at  $y = 5 \text{ mm}$ .

$$\tau = 58.29 \text{ N/mm}^2$$

If  $y < 5 \text{ mm}$

then

A ·  $\bar{y}$

$$\Rightarrow \frac{75 \times 16 \times 2}{2} \times \left[ \frac{75+5}{2} \right]$$

$$+ (5-y) \times 200 \times \left[ \frac{5+y}{2} \right]$$

$$= 102000 + 100(5^2 - y^2)$$

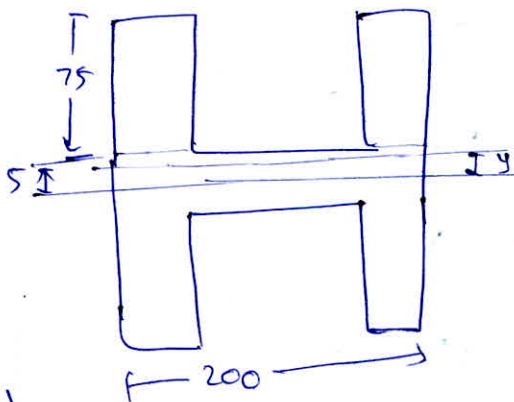
$$\tau_y = \frac{200 \times 10^3 \times [102000 + 100(5^2 - y^2)]}{10936666.67 \times 200}$$

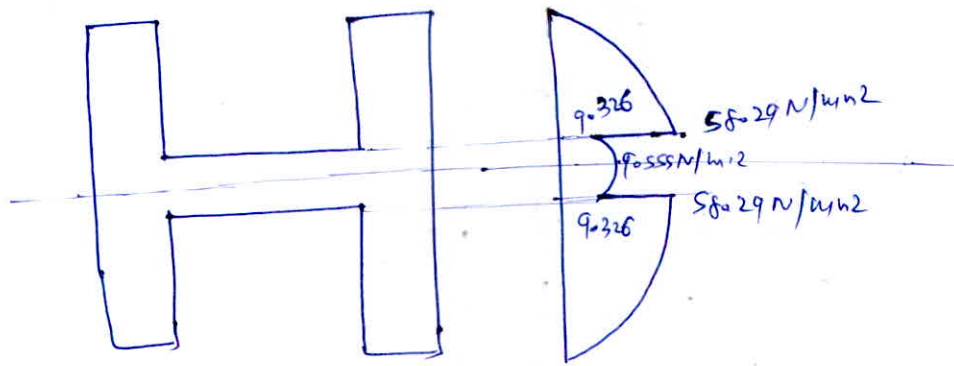
at  $y = 0$

$$\tau = 9.555 \text{ N/mm}^2$$

at  $y = 5 \text{ mm}$

$$\tau = 9.326 \text{ N/mm}^2$$

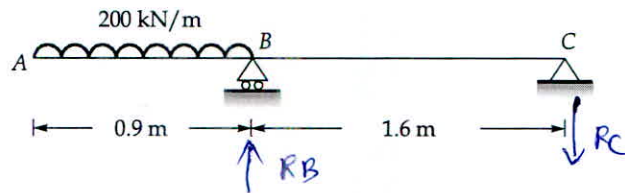




Variation of Shear stress

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Q.2 (a) For the beam shown below, find the deflection at end A, using moment area method. [Take EI = constant]



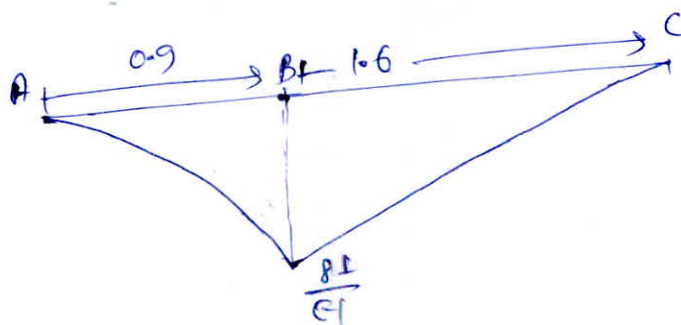
[20 marks]

$$R_B + R_C = 180 \text{ kN}$$

$$R_C = \frac{-1}{1.6} \left[ 200 \times 0.9 \times \frac{0.9}{2} \right]$$

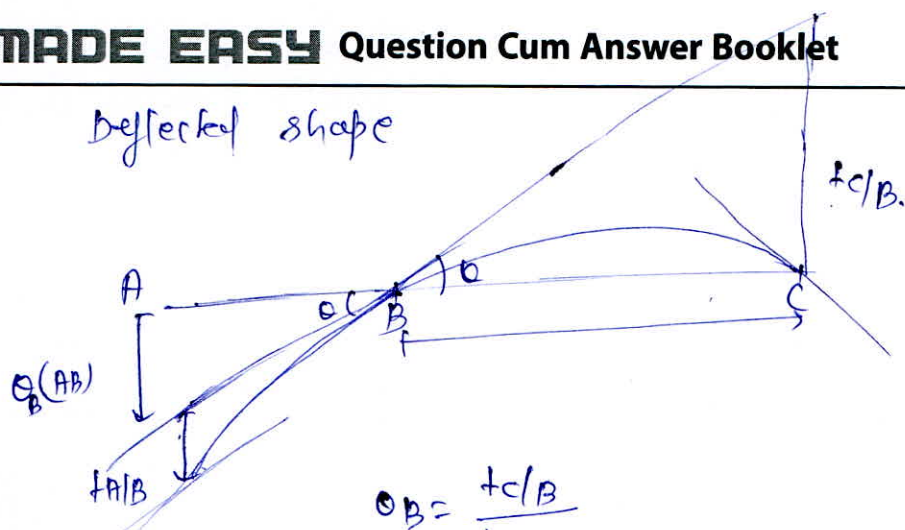
$$\left[ \begin{aligned} R_C &= -50.625 \text{ kN} \\ R_B &= \frac{184.5}{8} \text{ kN} \end{aligned} \right]$$

Moment diagram: ( $\frac{m}{EI}$ )





Deflected shape



$$\theta_B = \frac{\delta_{C/B}}{L_{BC}}$$

deflection at A =  $\theta_B(AB) + \delta_{A/B}$

$$\delta_{C/B} = \left(\frac{1}{2} \times \frac{51}{EI} \times 1.6\right) \times \frac{2}{3} \times 1.6$$

$$= \frac{1728}{25EI}$$

$$\theta_B = \frac{1728}{25EI \times 1.6}$$

$$\theta_B = \frac{216}{5EI}$$

$$\delta_{A/B} = \frac{1}{3} \times \left(\frac{51}{EI} \times 0.9\right) \times \frac{0.9 \times 3}{4}$$

$$= \frac{6561}{400EI}$$

$\delta$  at A

$$= \theta_B(\text{length of AB}) + \delta_{A/B}$$

$$= \frac{216}{5EI} \times (0.9) + \frac{6561}{400EI}$$

$$= \frac{972}{25EI} + \frac{6561}{400EI}$$

$$= \frac{22113}{400EI}$$

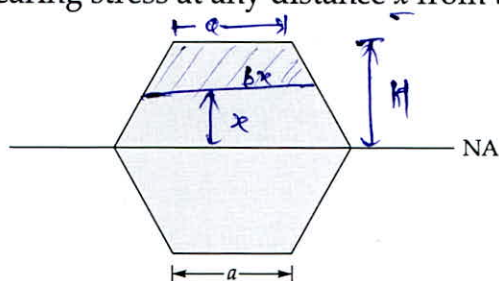
$$\delta_A = \frac{55.2825}{EI}$$

$$20 - 1 = 19$$

deflection direction should be mentioned in final answer.



- 2 (b) A beam section is a regular hexagon of side 'a' and is placed so that one diagonal is horizontal as shown below. If the beam section is subjected to a shear force S, obtain an expression for the shearing stress at any distance x from the horizontal diagonal.


**[20 marks]**

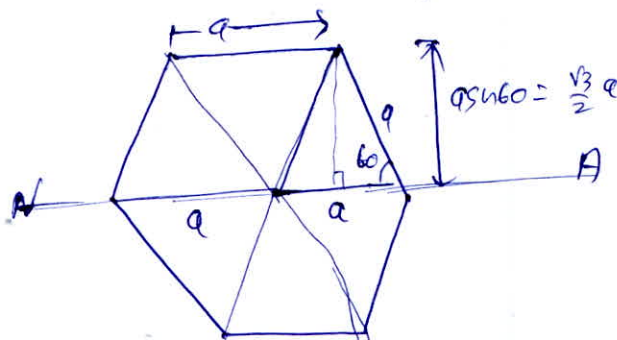
length of diagonal =  $2a$ .

at x distance from neutral axis.

calculate length of bx.

$$H = a \sin 60$$

$$= a \left( \frac{\sqrt{3}}{2} \right)$$



$$bx = 2a - \left( \frac{2a - a}{H} \right) x \cdot x$$

$$bx = 2a - \frac{a}{H} \cdot x$$

$$bx = 2a - \frac{H \times 2 - x}{\sqrt{3}a}$$

$$bx = 2a - \frac{a}{H} \cdot x$$

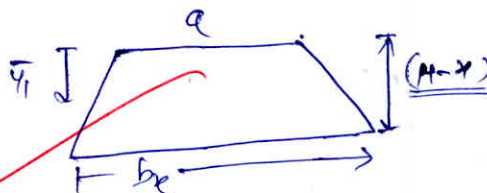
$$bx = \left( 2a - \frac{2x}{\sqrt{3}} \right)$$

Area

$$= \frac{1}{2} [a + bx] x (H - x)$$

$$\Rightarrow \frac{(a + bx)(H - x)}{2}$$

$$\Rightarrow \frac{\left[ a + \left( 2a - \frac{a}{H} x \right) \right] (H - x)}{2} = \frac{\left( 3a - \frac{a}{H} x \right) (H - x)}{2}$$



$\bar{y}$  from NA given by

$$\Rightarrow H - \bar{y}$$

$$\bar{y} = \left( \frac{2bx + q}{a + bx} \right) \left[ \frac{H-x}{3} \right]$$

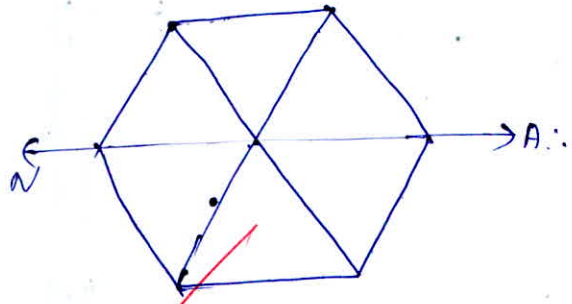
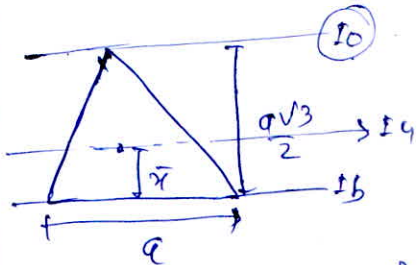
$$= \frac{2x(2a - \frac{qx}{H}) + q}{(2a - \frac{qx}{H} + q)} \times \left[ \frac{H-x}{3} \right]$$

$$\Rightarrow \frac{4a - \frac{2qx}{H} + q}{3a - \frac{qx}{H}} \times \left( \frac{H-x}{3} \right)$$

$$= \left( \frac{5a - \frac{2qx}{H}}{3a - \frac{qx}{H}} \right) \left( \frac{H-x}{3} \right)$$

$$H - \bar{y}_1 = H - \left[ \frac{5 - \frac{2x}{H}}{3 - \frac{qx}{H}} \left( \frac{H-x}{3} \right) \right]$$

Calculation of  $I_1$  :-



$$I_b = I_G + A \cdot \bar{y}^2$$

$$= \frac{a^4 \sqrt{3}}{36} + \frac{1}{2} a^2 \cdot \frac{3}{4} a^2$$

$$\Rightarrow \frac{a^4 \left( \frac{a\sqrt{3}}{2} \right)^3}{36} + \frac{1}{2} a^4 \left( \frac{\sqrt{3}a}{2} \right) \left( \frac{\sqrt{3}a}{2} \right)^2 = \frac{1}{12} a^4 \left( \frac{a\sqrt{3}}{2} \right)^3$$

$$= \frac{3\sqrt{3}a^4}{96}$$

$$I_0 = \frac{a^4 \left( \frac{a\sqrt{3}}{2} \right)^3}{36} + \frac{1}{2} a^4 \left( \frac{\sqrt{3}a}{2} \right) \left( \frac{\sqrt{3}a}{2} \right)^2$$

$$\Rightarrow \frac{3\sqrt{3}a^4}{288} + \frac{3\sqrt{3}a^4}{36} = \frac{3\sqrt{3}a^4}{32}$$



∴ of Regular Hexagon

$$= 4I_b + 2I_o$$

$$4 \times \frac{3\sqrt{3} a^4}{96} + 2 \times \frac{3\sqrt{3} a^4}{32} = \boxed{\frac{15\sqrt{3} a^4}{48}}$$

$$A \cdot \bar{y} = \left[ \frac{(3a - \frac{a \cdot y}{H})(H-y)}{2} \times \left[ H - \left( \frac{5 - \frac{2x}{H}}{3 - \frac{2x}{H}} \left( \frac{H-y}{3} \right) \right) \right] \right]$$

where  $H = \frac{\sqrt{3}}{2} a$

$$\tau \text{ (shear stress)} = \frac{V A \cdot \bar{y}}{I \cdot b}$$

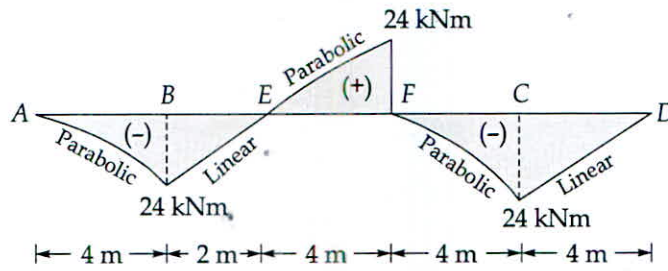
$$= \frac{V \times \left[ \frac{(3a - \frac{a \cdot y}{H})(H-y)}{2} \right] \times \left[ H - \frac{5 - \frac{2x}{H}}{3 - \frac{2x}{H}} \left( \frac{H-y}{3} \right) \right]}{\frac{15\sqrt{3} a^4}{48} \times (2a - \frac{a \cdot y}{H})}$$

where  $H = \frac{\sqrt{3}}{2} a$

Simplify the expression.

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Q.2 (c) A beam ABCD is supported at B and C and has over-hangs AB and CD. Its bending moment diagram is shown below. Determine the loading diagram and the shear force diagram of the beam.



[20 marks]

Shear force  $v = \frac{dM}{dx}$

loading  $w = \frac{dv}{dx} = \frac{d^2M}{dx^2}$

In AB:

moment =  $w \times \frac{x^2}{2} = 24$

$w = 3 \text{ kN/m}$  downward.

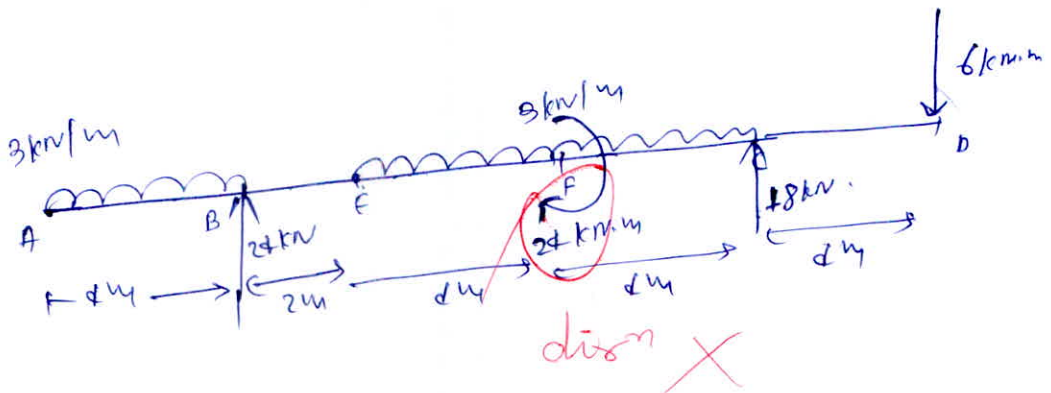
In BE  $w = \frac{d^2M}{dx^2} = 0$  there is no load present within BE.

In EF there is load of  $3 \text{ kN/m}$  acting and at point F there is point moment of magnitude  $24 \text{ kNm}$ .

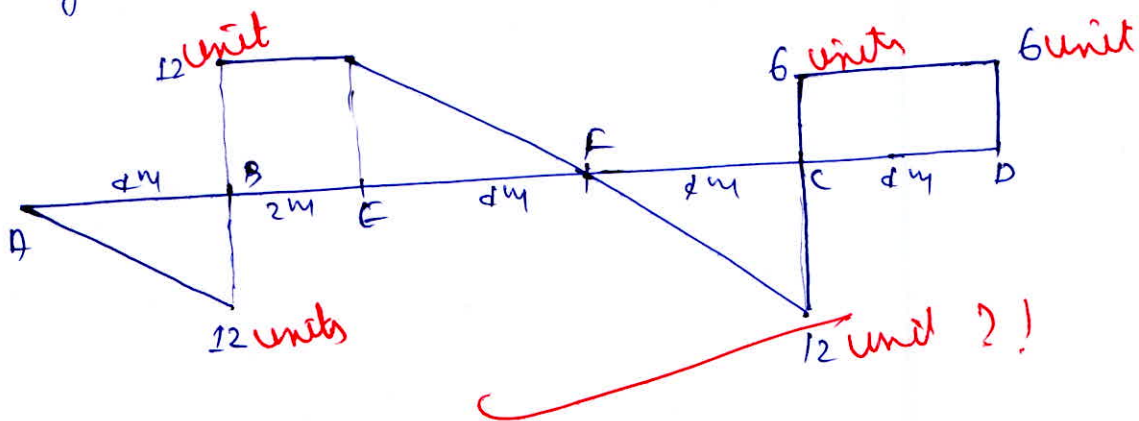
In FC  $3 \text{ kN/m}$  load acting downward.

at point C:  $6 \text{ kN}$  point load acting.

loading diagram



Shear force diagram.

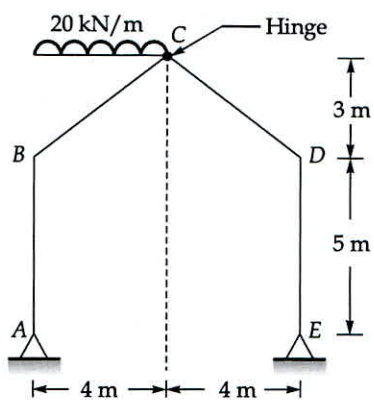


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3 (a) Draw the bending moment diagram for the frame shown below.

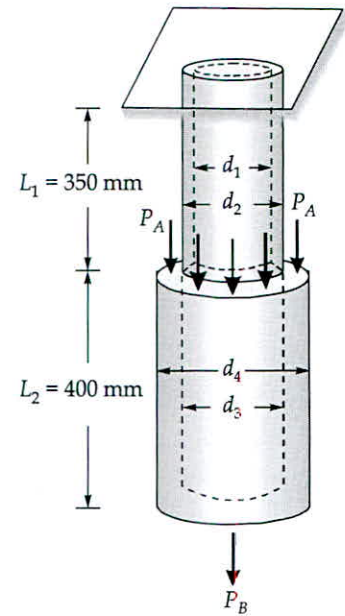


[20 marks]





- Q.3 (b) A hollow circular nylon pipe as shown in figure supports a load  $P_A = 7800$  N, which is uniformly distributed around a cap plate at the top of lower pipe. A second load  $P_B$  is applied at the bottom. The inner and outer diameters of the upper and lower parts of the pipe are  $d_1 = 51$  mm,  $d_2 = 60$  mm,  $d_3 = 57$  mm and  $d_4 = 63$  mm respectively. The upper pipe has a length  $L_1 = 350$  mm and lower pipe has a length  $L_2 = 400$  mm. Neglect the self weight of the pipes.



- (i) Find  $P_B$  so that the tensile stress in the upper pipe is 14.5 MPa. Also determine the resulting stress in lower pipe?
- (ii) If  $P_A$  remains unchanged, find the new value of  $P_B$  so that upper and lower pipes have same tensile stress.
- (iii) Find the tensile strains in the upper and lower pipe segments for the loads in part (ii) if the elongation of the upper pipe is 3.56 mm and downward displacement of bottom pipe is 7.63 mm?

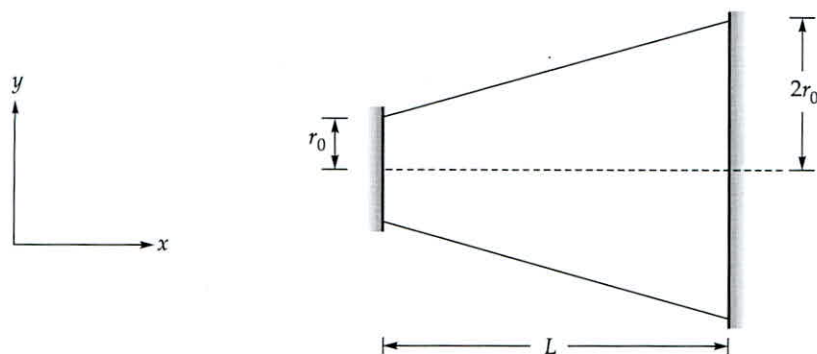
[20 marks]



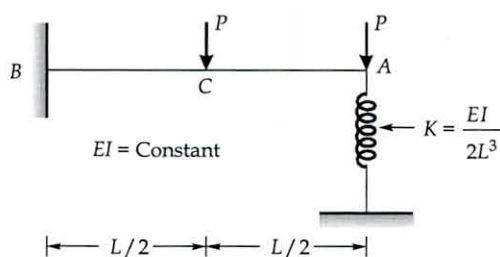




- 3 (c) (i) A bar as shown in figure below is in a shape of a solid, truncated cone of circular cross-section and is situated between two rigid supports. The temperature of the entire bar is then raised by  $\Delta T$ . Assume that the cross-sections perpendicular to longitudinal axis of symmetry remain plane and neglect localised end effect due to the end supports. Determine the normal stress at any point in the bar.



- (ii) Determine the bending moment and shear force at support  $B$  in the uniform beam  $AB$  with flexural rigidity  $EI$  shown in the figure. Take spring constant  $K = \frac{EI}{2L^3}$ .



[10 + 10 marks]



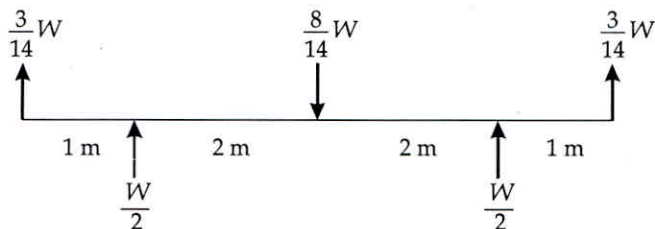






Q.4 (a)

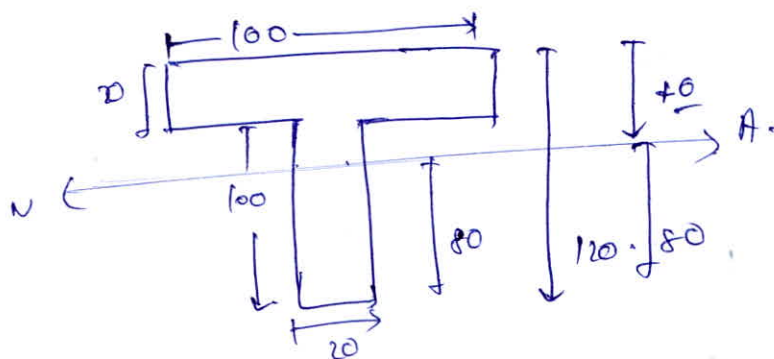
- (i) A beam of T-section 6 m long supports the load system as shown below. The beam has a flange width of 100 mm and an overall depth of 120 mm. The flange and the web are 20 mm thick. The section is placed with flange at the bottom. Find the safe value of W if the stresses in compression and tension shall not exceed 90 N/mm<sup>2</sup> and 50 N/mm<sup>2</sup> respectively.



- (ii) If a tension test bar is found to taper uniformly from (D - a) diameter to (D + a) diameter, prove that the error involved in using the mean diameter to calculate the Young's modulus is  $\left(\frac{10a}{D}\right)^2$  percent.

[10 + 10 marks]

(i)

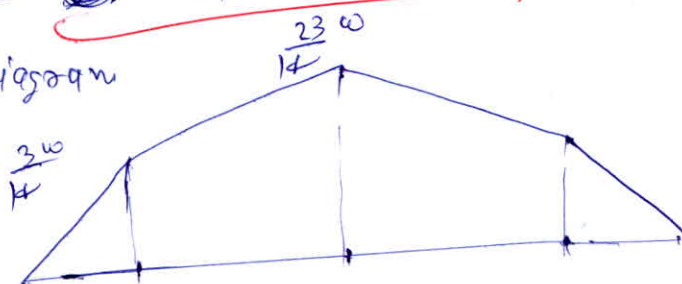


$$\bar{x} = \frac{(100 \times 70 \times 50) + (100 \times 20 \times 110)}{(100 \times 20) + (100 \times 20)} = 80 \text{ mm}$$

$$I = \frac{20 \times 100^3}{12} + 20 \times 100 \times 30^2 + \frac{100 \times 20^3}{12} + 100 \times 20 \times 50^2$$

$$= 333330333 \text{ mm}^4$$

BMD diagram



Maximum stress BM =  $\left(\frac{23W}{14}\right)$

In tension =  $\frac{23W \times 10^6 \cdot y}{H \times I} \leq 50$

10

$$\frac{230 \times 10^6 \times 100}{4 \times 333333.33} \leq 50$$

$$\omega \leq \frac{2246 \text{ kN}}{4}$$

$$\omega \leq 2.0289$$

as per compression criteria.

$$\frac{230 \times 10^6}{4 \times 333333.33} \leq 90$$

$$\omega \leq 7.068 \text{ kN}$$

so  $\omega$  should be less than above 2

$$\omega = 2.028 \text{ kN}$$

(11)

In 1st case

$$d_1 = \frac{4PL}{\pi (D+d)(D+d) E}$$

$$D_1 = \frac{4PL}{\pi E (D^2 - d^2)}$$

$$E_1 = \frac{4PL}{\pi D (D^2 - d^2)}$$

from mean dia

$$D = \frac{PL}{\frac{\pi}{4} \left[ \frac{(D+d)(D-d)}{2} \right]^2 \cdot E}$$

$$D = \frac{4PL}{\pi D \cdot E} \quad E = \frac{4PL}{\pi D^2 E}$$

$$\text{error} = \frac{4PL}{\pi D (D-d)} - \frac{4PL}{\pi D^2}$$

$$= \frac{4PL}{\pi D} \left[ \frac{D - D + d^2}{(D-d) \cdot D} \right] = \frac{4PL \cdot d^2}{\pi D (D-d) D^2}$$

$$\sqrt{10000}$$

$$= \frac{\cancel{d}^2 d^2 \times 100}{\cancel{d}^2 (b^2 - d^2) b^2 \times \cancel{d}^2} \times 100$$

$$\Rightarrow \frac{d^2 \times 100}{b^2}$$

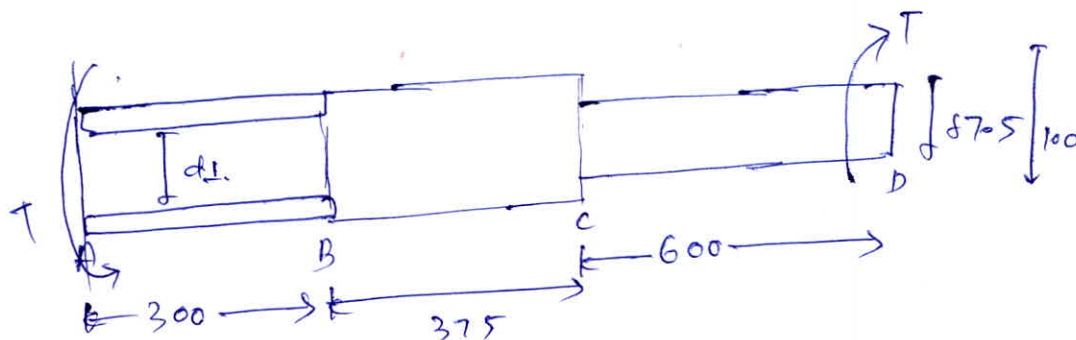
$$\sqrt{10000} = \left(\frac{100}{b}\right)^2$$

$$10$$



Q.4 (b) A steel shaft ABCD has a total length of 1275 mm and is made up as follows: AB = 300 mm, BC = 375 mm and CD = 600 mm. AB is hollow, its outside diameter being 100 mm and inside diameter  $d_1$  mm. BC and CD are solid having diameters of 100 mm and 87.5 mm respectively. If equal and opposite torques are applied at the ends of the shaft, then find the maximum permissible value of  $d_1$  for the maximum shearing stress in AB not to exceed that in CD. If the torque applied to the shaft is 9000 Nm, what is the total angle of twist? Take  $G = 8 \times 10^4 \text{ N/mm}^2$ .

[20 marks]



Maximum shearing stress in AB

$$= \frac{T \cdot r}{\frac{\pi}{32} [(100^4 - d_1^4)] \times \frac{100}{2}}$$

$$\Rightarrow \frac{16T \times 100}{\pi (100^4 - d_1^4)} = \tau_{cb}$$

$$\tau_{cb} = \frac{16T}{\pi (87.5)^3}$$

$$\frac{16T \times 100}{\pi (100^4 - d_1^4)} = \frac{16T}{\pi \times 87.5^3}$$

$$d_1 = 75.79 \text{ mm}$$

If the Torque = 9000 Nm.

Total angle of twist.

$$\phi = \frac{9000 \times 10^3 \times 600}{G \left[ \frac{\pi}{32} \times 87.5^4 \right]} + \frac{9000 \times 10^3 \times 375}{G \left[ \frac{\pi}{32} \times 100^4 \right]} + \frac{9000 \times 10^3 \times 300}{G \left[ \frac{\pi}{32} (100^4 - 75.79^4) \right]}$$



$$\phi = \frac{938.343}{4} + \frac{343.7746}{4} + \frac{410.4463}{4}$$

$$= \frac{1692.563961}{8 \times 10^4}$$

$$= 0.021157 \text{ rad}$$

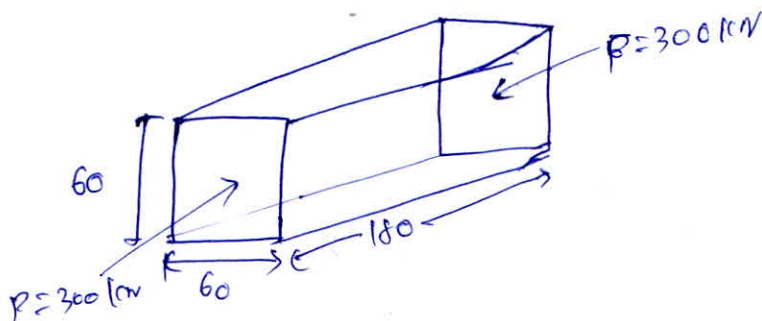
$$\phi = 1.212^\circ$$

no

Q.4 (c) A steel bar of square section 60 mm × 60 mm and 180 mm long is subjected to an axial compressive load of 300 kN. The lateral strain is prevented by the application of uniform external pressure. If  $\mu = 0.3$  and  $E = 2 \times 10^5 \text{ N/mm}^2$ , find the alteration in the length of the bar.

If however, only one-half the lateral strain is prevented then what would be the alteration in the length of bar?

[20 marks]



lateral strain is prevented by uniform pressure.

$$\mu = 0.3, \quad E = 2 \times 10^5 \text{ N/mm}^2$$

$$\therefore \epsilon_{\text{lat}} = 0$$

$$\Rightarrow \sigma_1 = \sigma_2 = P \text{ - lateral stress then,}$$

$$\frac{P_1}{E} - \frac{\mu P}{E} - \frac{\mu \times P}{E} = 0$$

$$\Rightarrow P(1 - \mu) = \mu \times P$$

$$P = \frac{\mu}{1 - \mu} \times \left( \frac{300 \times 10^3}{60 \times 60} \right)$$

$$P = \frac{0.3}{0.7} \times \frac{250}{3} = \underline{\underline{-35.714 \text{ N/mm}^2}} \text{ (lateral pressure).}$$

alteration in length =  $\epsilon_{\text{longitudinal}} \times \text{length}$

$$\epsilon_{\text{L}} \Rightarrow \left( \frac{300 \times 10^3}{60 \times 60} \right) \times \frac{1}{E} - \frac{\mu \cdot P}{E} - \frac{\mu \cdot P}{E}$$

$$P = \frac{250}{7} \text{ N/mm}^2$$

$$\epsilon_{\text{L}} = \frac{-250}{3E} + \frac{75}{7E} + \frac{75}{7E}$$

$$\epsilon_{\text{L}} = \frac{-1300}{21E}$$

$$\Delta L = \frac{-1300}{21 \times 2 \times 10^5} \times 180$$

$$\Delta L = \underline{\underline{-0.0557 \text{ mm}}}$$

compressed

(11)

Sj

$$\frac{P}{E} - \frac{4P}{E} - \frac{45}{E} + \frac{45}{2E} = 0$$

$$\frac{P}{E} - \frac{4P}{E} = \frac{45}{2E}$$

$$P = \frac{4}{(4-1)2} \times \left( \frac{360 \times 10^3}{60 \times 60} \right) = \frac{-125 \text{ N/mm}^2}{7}$$

$$\Delta l = \left( \frac{-300 \times 10^3}{3600 E} + \frac{0.3 \times 125}{7E} + \frac{0.3 \times 125}{7E} \right) \times l$$

$$\Delta l = \left( \frac{-1525}{21E} \right) \times 180$$

$$= \frac{-1525 \times 180}{21 \times 21 \times 10^9}$$

$$\Delta l = -0.065357 \text{ mm}$$

compressed

20

**Section B : Transportation Engg-1 + Surveying and Geology-1  
Geo-technical & Foundation Engg-2 + Environmental Engg-2**

- 5 (a) A footing  $3 \text{ m} \times 2 \text{ m}$  in size transmits a pressure of  $140 \text{ kN/m}^2$  on a soil having  $E = 5 \times 10^4 \text{ kN/m}^2$  and  $\mu = 0.50$ . Find the immediate settlement for the footing at the centre. Assuming it to be (i) Flexible footing (ii) Rigid footing  
For  $L/B = 1.5$ , Influence factor = 1.36 for flexible and 1.06 for rigid footing.

[12 marks]

$$\text{pressure} = 140 \text{ kN/m}^2$$

(i) Flexible footing.

$$s_f = \frac{q_B (1-\mu^2) I_f}{E}$$

$$= \frac{140 \times 2 \times 10^3 \times (1-0.5^2) \times 1.36}{5 \times 10^4}$$

$$= \boxed{5.712 \text{ mm}}$$

(ii) For Rigid footing.

$$s_f = \frac{q_B (1-\mu^2) \cdot I_f}{E}$$

$$= \frac{140 \times 2 \times 10^3 \times (1-0.5^2) \times 1.06}{5 \times 10^4}$$

$$= \boxed{4.452 \text{ mm}}$$

12

Q.5 (b) A column footing of  $1.8 \text{ m} \times 1.8 \text{ m}$  is to be placed  $1.5 \text{ m}$  below ground level in a dry cohesionless soil. The unit weight of soil is  $21 \text{ kN/m}^3$  and angle of internal friction,  $\phi = 36^\circ$ . The footing is required to carry a total load of  $1350 \text{ kN}$  including column load, weight of footing and weight of soil surcharge. Determine the factor of safety against bearing capacity failure assuming:

- Ground water table well below the base of footing, and
- Ground water table at ground level

Given for  $\phi = 36^\circ$ ,  $N_c = 63.53$ ,  $N_q = 47.16$ ,  $N_\gamma = 51.7$

[Assume,  $\gamma_{\text{bulk}} = \gamma_{\text{saturated}} = 21 \text{ kN/m}^3$ ]

[12 marks]

(i) Footing load

$$= 1350 \text{ kN}$$

footing square

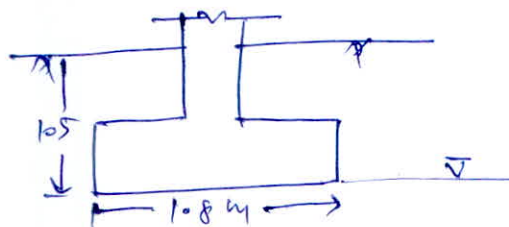
$$c = 0,$$

$$q = \gamma D F = 21 \times 1.5 = 31.5 \text{ kN/m}^2$$

$$\therefore \frac{1350}{1.8^2} = \frac{0 + 31.5(N_c) + q(N_q - 1) + 0.4 \gamma B N_\gamma}{\text{FOS}} + q$$

$$\Rightarrow \frac{1350}{1.8^2} = \frac{0 + 31.5(63.53) + 31.5(47.16 - 1) + 0.4 \times 21 \times 1.8 \times 51.7}{\text{FOS}} + 31.5$$

$$\Rightarrow \frac{1350}{1.8^2} = \frac{18700.576}{\text{FOS}} + 31.5$$



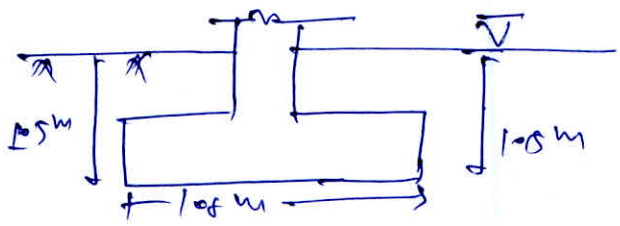


~~$FOS = 4.8563$~~

(11)

$$q = (21 - 9.81) \times 1.5$$

$$= 16.785 \text{ kN/m}^2$$



footing load

$$= \frac{1.3 \text{ MC} + q(Nq+1) + \text{oad} \times B W \times + q}{FOS}$$

$$\Rightarrow \frac{1350}{1.82} = \frac{0 + \frac{16.785 (16 + 16) + (0.04 \times (21 - 9.81) \times 1.8 \times 1.7)}{16.785}}{FOS}$$

$$\Rightarrow \frac{1250}{3} = \frac{1191.33}{FOS} + 16.785$$

~~$FOS = 2.979$~~

6

- Q.5 (c) (i) Define the processes involved in MBBR (Moving Bed Biofilm Reactor) used for secondary wastewater treatment?
- (ii) One hundred cubic meters per day ( $100 \text{ m}^3/\text{d}$ ) of mixed sludge at 4 percent solids is to be thickened to 8.0 percent solids. What is the approximate volume of the sludge after thickening and also comment on the result?

[6 + 6 marks]

(ii) Volume of sludge =  $100 \text{ m}^3/\text{d}$  at solid = 4%.

Let assume volume of sludge at 8% solid =  $V_2$

$$\therefore V_2(100 - P_1) = V_2(100 - P_2)$$

where  $P_1$  and  $P_2$  = water content.

then

$$100[100 - 96] = V_2[100 - 92]$$

$$100 \times 4 = 8V_2$$

$$\boxed{V_2 \approx 50 \text{ m}^3}$$

Comment?



Do not  
write in  
this margin

Q.5 (d) A 4-lane National Highway is passing through a built up area. Design the following geometric features for a horizontal circular curve of radius 350 m for this highway considering design speed as 80 kmph and the length of wheel base of largest truck as 6.0 m:

- (i) Superelevation
- (ii) Length of transition curve

Also suggest the most suitable shape of curve.

[12 marks]

(f) Horizontal curve of Radius = 350 m.  
 design speed = 80 kmph.  
 length of wheel base = 6.0 m.

$$e = \frac{(0.75V)^2}{gR}$$

$$= \frac{(0.75 \times \frac{80}{3.6})^2}{9.81 \times 350} = 0.0809 > \underline{0.07}$$

Hence restrict the superelevation to 0.07

check

$$e + f = \frac{V^2}{gR}$$

$$= \frac{(80/3.6)^2}{9.81 \times 350} = 0.1438$$

$$f = 0.1438 - 0.07$$

$$= 0.0738 < 0.15 \text{ Hence } \underline{\underline{OK}}$$

provide superelevation of 7%

(ii) If Radius is  $> 300$  meter then no need to provide extra widening.

$$w_p = 0,$$

width of pavement for 4 lane

$$= 3.5 \times 4 = \underline{\underline{14M}}$$

length of transition curve

as per comfort criteria:

$$e = \frac{V^3}{CR}$$

$$e = \frac{30^3}{75 \times 350} = \underline{\underline{0.516}} \text{ is between } \underline{\underline{0.5-0.8}}$$

$$L = \frac{(e/0.06)^3}{0.516 \times 350} = \underline{\underline{60.76 \text{ m}}}$$

as per Rate of superelevation for built up area

$$N = 100,$$

assume pavement rotated about ~~types edge~~ central of pavement then.

length of transition curve

$$= \left[ \frac{14 \times 0.07}{2} \right] \times 100$$

$$= \underline{\underline{49 \text{ m}}}$$

$$e = 0.07$$

12

AS per IRC

$$L_s = \frac{2.7 V^2}{R} = \frac{2.7 \times (30)^2}{350}$$

$$= \underline{\underline{49.37 \text{ m}}}$$



Here transition length is maximum of above three

$$L_s = 60.76 \text{ m}$$

∴ Most suitable shape for transition length is spiral because in spiral curve length  $\propto \frac{L_s}{\text{radius}}$

$$L_s = \text{constant}$$

- Q.5 (e) A railway embankment, 500 m long, has a width at formation level of 9 m with side slopes of 2 to 1. The ground levels at every 100 m distance along the centreline are as follows:

Distance, (m)	0	100	200	300	400	500
Ground level, (m)	107.8	106.3	110.5	111.0	110.7	112.2

The embankment has a rising gradient of 1.2 m per 100 m and the formation level is 110.5 m at zero chainage. Assume the ground to be level across the centerline, compute the volume of earthwork using trapezoidal method.





- Q.6 (a)** (i) For a railway track 7 m high embankment is required. The clay to be used for the embankment was found to have  $c = 20 \text{ kN/m}^2$  and unit weight  $= 19 \text{ kN/m}^3$ . Compute the critical maximum side slope angle for the embankment if a hard rocky stratum was found 3.5 m below the ground level. Assume  $\phi$  for the clay equal to zero. The following values are given from Taylor's chart for depth factor  $D = 1.5$  :

$S_n$	0.181	0.174	0.164	0.150
$\beta$	$53^\circ$	$45^\circ$	$30^\circ$	$20^\circ$

- (ii) Using Terzaghi's method, determine the ultimate bearing capacity of a square footing of size 1.5 m with its base at a depth of 1 m below the ground level, resting on a dry sand stratum.

Take  $\gamma_d = 17 \text{ kN/m}^3$ ,  $\phi' = 38^\circ$ ,  $c' = 0$ ,  $N_q = 60$ ,  $N_\gamma = 75$ .

[10 + 10 marks]

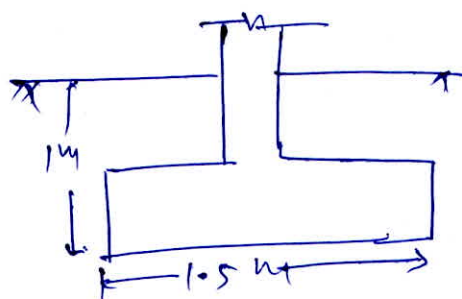
(1)  $c = 20 \text{ kN/m}^2$   $\gamma = 19 \text{ kN/m}^3$   
 $\phi = 0$ ,  $D = 1.5 \text{ m}$ .

(11)

$$q = 17 \times 1$$

$$= 17 \text{ kN/m}^2$$

Ultimate load  
square footing



$$= 1.3 c N_c + q N_q + 0.4 \times 17 \times 1.5 \times 75$$

$$= 1.3 \times 0 + 17 \times 60 + 0.4 \times 17 \times 1.5 \times 75$$

$$= 0 + 1020 + 765$$

$$= 1785 \text{ kN/m}^2$$

$$Q_u = 1785 \text{ kN/m}^2$$

10



(b) Consider the following data for a completely mixed activated sludge system to treat wastewater from a community of 60000 persons:

Sewage flow,  $Q = 9000 \text{ m}^3/\text{day}$

$\text{BOD}_5 = 360 \text{ mg/l}$  (raw)

Assume 30% BOD removal in primary settling and 90% in biological treatment.

Winter temperature of mixed liquor =  $10^\circ\text{C}$

Yield,  $y = 0.6$

$k_d = 0.07/\text{day}$  ( $\text{BOD}_5$  basis at  $15^\circ\text{C}$ )

$\text{MLSS} = 4000 \text{ mg/l}$ ,  $\frac{\text{VSS}}{\text{SS}} = 0.8$

Adopt sludge age ( $\theta_c$ ) = 10 days

Determine F/M ratio and oxygen requirement uptake per day for this completely mixed activated sludge system.

[20 marks]

$Q = 9000 \text{ m}^3/\text{day}$

$\text{BOD}_5 = 360 \text{ mg/l}$

BOD in after primary settling =  $252 \text{ mg/l}$

BOD effluent =  $25.2 \text{ mg/l}$

$\text{MLSS} = 4000$

$\text{MLVSS} = 4000 \times 0.8 = 3200 \text{ mg/l}$

$\theta_c = 10 \text{ days}$

$$\frac{F}{M} = \frac{(9000 \times 10^3) \times 252 \times 10^{-6}}{V \cdot X}$$

$$\therefore \frac{1}{\theta_c} + k_d = \frac{S_0(S_0 - S) \cdot Y}{V \cdot X}$$

$$\Rightarrow \frac{1}{10} + 0.07 = \frac{9000 \times 10^3 \times (252 - 25.2) \times 10^{-6} \times 0.6}{V \cdot X}$$

$$V \cdot X = 7204.235$$

10

$$\frac{F}{M} = \frac{9 \times 252}{7204.235} = 0.3148$$

$$\frac{F}{M} = 0.3148$$









- Q.6 (c) (i) What are the advantages of photogrammetric techniques in highway location and design? What are the various objectives of highway planning?  
 (ii) Following five alternate road plan development proposals with particulars as mentioned below are available:

Proposal	Number of towns and villages served along with population range					Total industrial products in thousand tonnes
	<2000	2001-5000	5001-10000	10001-20000	> 20000	
A	80	10	25	5	1	60
B	115	120	30	10	2	370
C	340	230	25	20	4	350
D	150	200	100	35	6	750
E	200	90	70	60	3	500

If the total road length of proposals A, B, C, D and E are respectively 200 km, 380 km, 605 km, 700 km and 400 km, calculate the utility rate per unit length of each road proposal and indicate the priority based on saturation system. Assume the utility units as follows:

For population :

Range	Unit
< 2000	: 0.25
2001 to 5000	: 0.50
5001 to 10000	: 1.00
10001 to 20000	: 2.00
> 20000	: 3.00

For products :

One unit for 1000 tonnes.

[8 + 12 marks]

(P1) For proposal A  
 utility per unit length  

$$= \frac{(80 \times 0.25) + (10 \times 0.5) + (25 \times 1.0) + (5 \times 2) + (1 \times 3) + 60}{200}$$

$$= 0.615$$

For Road B  
 utility per unit length =  $\frac{(115 \times 0.25) + (120 \times 0.5) + (30 \times 1) + 20 + 2 \times 370}{380}$   

$$= 1.34$$

for Road c

$$\begin{aligned} \text{Utility Factor} &= \frac{15 + 115 + 25 + 40 + 12 + 350}{605} \\ &= 1.036 \end{aligned}$$

for Road b

$$\begin{aligned} \text{Utility Factor} &= \frac{37.5 + 100 + 100 + 70 + 28 + 750}{700} \\ &= 1.536 \end{aligned}$$

Road E

$$\text{Utility Factor} = \frac{790}{400} = 1.975$$

Priority = E > D > B > C > A.

12





- Q.7 (a)
- (i) What are conditions which necessitate taking up of a realignment project of a highway? Discuss the general principles in the realignment of a highway and explain how the work is carried out.
  - (ii) Determine the extra width required for a road of carriageway 7.5 m on a horizontal curve of radius 300 m. The longest wheel base of vehicle using the road may be taken as 6.1 m. Design speed is 80 km/hr.

[16 + 4 marks]







(b) Design a septic tank with the following data :

- (i) Number of users = 200
- (ii) Rate of water supply = 150 l/head/day
- (iii) Detention period = 18 hours
- (iv) Percolating capacity of filter media of soak well = 1250 litres/m<sup>3</sup>
- (v) Rate of sludge accumulation = 40 litres/person/year

Also find the diameter of soak well. Assume reasonable data, if required.

**[20 marks]**





- Q.7 (c) Two sets of tacheometric readings were taken from an instrument station A (RL of A = 100 m) to a staff station B as shown below.

Instruments	P	Q
Multiplying constant	100	95
Additive constant	0.30	0.45
Height of instrument	1.40 m	1.45 m
Staff held	Vertical	Normal

Instruments	Instruments station	Staff station	Vertical angle	Stadia readings
P	A	B	5°44'	1.090, 1.440, 1.795
Q	A	B	5°44'	?

**Determine:**

- (i) The distance between instrument station and staff station.
- (ii) The R.L. of staff station B.
- (iii) Stadia readings with instrument Q.

[20 marks]







a) The following whole circle bearings were observed in running a closed traverse:

Line	F.B.	B.B.
AB	71°05'	250°20'
BC	110°20'	292°35'
CD	161°35'	341°45'
DE	220°50'	40°05'
EA	300°50'	121°10'

Determine the correct magnetic bearings of the lines.

**[20 marks]**





- Q.8 (b)
- (i) What are the desirable characteristics of grouting material in soils? List some of grouting methods adopted in practice.
  - (ii) A square pile group of 16 piles penetrates through a filled up soil of 3 m depth. The pile diameter is 250 mm and pile spacing is 0.75 m. The unit cohesion of the material is  $18 \text{ kN/m}^2$  and the unit weight of soil is  $15 \text{ kN/m}^3$ . Draw plan and sectional elevation of the pile group and compute the negative skin friction on the group. [Take  $\alpha = 0.7$ ]

[6 + 14 marks]





- Q.8 (c) (i) The driver of a vehicle requires 15 m less to stop after he applies the brakes while travelling up a grade than a driver travelling at the same initial speed down the same grade. Consider the coefficient of friction between tyres and pavement as 0.35 and initial speed to be 90 kmph. What is the percent grade?
- (ii) Compute the moisture deficit in a landfill for each each cubic meter of waste if the parameters are :
- Density of waste at time of deposit =  $800 \text{ kg/m}^3$ .
- Field capacity = 60% by weight.
- Water content of waste being deposited = 30% by weight.
- Also discuss the result.

[10 + 10 marks]





OOOO

**Space for Rough Work**

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**Space for Rough Work**

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