



MADE EASY

India's Best Institute for IES, GATE & PSUs

ESE 2024 : Mains Test Series

UPSC ENGINEERING SERVICES EXAMINATION

Mechanical Engineering

Test-1 : Thermodynamics [All Topics]

Strength of Materials & Mechanics [All Topics]

Name :

Roll No :

Test Centres

Delhi Bhopal Jaipur
Pune Kolkata Hyderabad

Student's Signature

Instructions for Candidates

- Do furnish the appropriate details in the answer sheet (viz. Name & Roll No).
- There are Eight questions divided in TWO sections.
- Candidate has to attempt FIVE questions in all in English only.
- Question no. 1 and 5 are compulsory and out of the remaining THREE are to be attempted choosing at least ONE question from each section.
- Use only black/blue pen.
- The space limit for every part of the question is specified in this Question Cum Answer Booklet. Candidate should write the answer in the space provided.
- Any page or portion of the page left blank in the Question Cum Answer Booklet must be clearly struck off.
- There are few rough work sheets at the end of this booklet. Strike off these pages after completion of the examination.

FOR OFFICE USE

Question No.	Marks Obtained
Section-A	
Q.1	44
Q.2	45
Q.3	—
Q.4	48
Section-B	
Q.5	1
Q.6	8
Q.7	—
Q.8	—
Total Marks Obtained	145

Signature of Evaluator

Cross Checked by

.....

IMPORTANT INSTRUCTIONS

CANDIDATES SHOULD READ THE UNDERMENTIONED INSTRUCTIONS CAREFULLY. VIOLATION OF ANY OF THE INSTRUCTIONS MAY LEAD TO PENALTY.

DONT'S

1. Do not write your name or registration number anywhere inside this Question-cum-Answer Booklet (QCAB).
2. Do not write anything other than the actual answers to the questions anywhere inside your QCAB.
3. Do not tear off any leaves from your QCAB, if you find any page missing do not fail to notify the supervisor/invigilator.
4. Do not leave behind your QCAB on your table unattended, it should be handed over to the invigilator after conclusion of the exam.

DO'S

1. Read the Instructions on the cover page and strictly follow them.
2. Write your registration number and other particulars, in the space provided on the cover of QCAB.
3. Write legibly and neatly.
4. For rough notes or calculation, the last two blank pages of this booklet should be used. The rough notes should be crossed through afterwards.
5. If you wish to cancel any work, draw your pen through it or write "Cancelled" across it, otherwise it may be evaluated.
6. Handover your QCAB personally to the invigilator before leaving the examination hall.

Remarks:-

- Mention the units properly to avoid silly mistakes.
- Improve the presentation of writing the solutions.

Section : A

- Q.1 (a) The exhaust gas at 720°C from a boiler is used to heat water. The rate of gas flow is 1450 kg/min and the rate of water flow is 2000 kg/min . The water enters the heat exchanger at 42°C and the gases leave the exchanger at 160°C . Assume that the mean specific heat of gas and water are 1.088 kJ/kgK and 4.27 kJ/kgK respectively.

The atmospheric temperature is 27°C . Determine the loss of available energy resulting from heat transfer. Assuming Exhaust gas to behave as ideal gas :-

for Exhaust gas, $T_1 = 720^{\circ}\text{C}$, $m_1 = 1450\text{ kg/min}$, $T_2 = 160^{\circ}\text{C}$ [12 marks]
 $C_{p1} = 1.088\text{ kJ/kgK}$.

for water, $T_3 = 42^{\circ}\text{C}$, $m_2 = 2000\text{ kg/min}$, $(C_p)_w = 4.27\text{ kJ/kgK}$

By Energy Conservation (assuming all heat from gas goes to water)

$$(m C_p \Delta T)_{\text{gas}} = (m C_p \Delta T)_{\text{water}} \Rightarrow 1450 \times 1.088 [720 - 160] = 2000 \times 4.27 \times [T_4 - 42]$$

outlet temp of water, $T_4 = 145.45^{\circ}\text{C}$

Ambient condition = $T_0 = 300\text{K}$.

change in availability of gases from going from 720°C to 160°C .

$$= (h_1 - T_0 s_1) - (h_2 - T_0 s_2)$$

$$= (h_1 - h_2) - T_0 (s_2 - s_1)$$

$$\phi_1 - \phi_2 = \frac{1450}{60} \left[1.088 [720 - 160] + 300 \times \frac{1450}{60} \left[1.033 \ln \frac{433}{393} - R \ln \frac{p_2}{p_1} \right] \right]$$

assuming heat transfer takes place at constant pressure,

$$\frac{h_2 p_2}{p_1} = 0$$

$$\phi_1 - \phi_2 = 8.18 \text{ MJ}$$

for water
change in availability $\Rightarrow (h_4 - T_0 s_4) - (h_3 - T_0 s_3)$

$$= (h_4 - h_3) - T_0 (s_4 - s_3)$$

$$= \frac{2000}{60} \left[4.27 [145.45 - 42] - 300 \times \left[4.27 \ln \frac{418.45}{315} \right] \right]$$

$$= 2.567 \text{ MJ}$$

∴, loss of available Energy due to heat transfer =
the (loss in available energy of gas) - [Gain in the available energy of water]

$$= 8.18 - 2.567$$

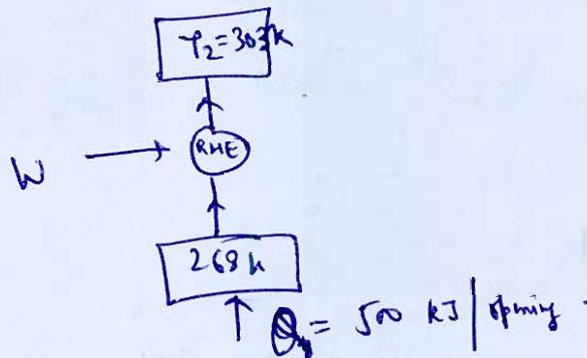
$$= \boxed{5.913 \text{ MJ.}}$$

(6)

- Q.1 (b) A household refrigerator maintains a space at a temperature of -5°C . Every time the door is opened, warm material is placed inside, introducing an average 500 kJ of heat, but making only a small change in temperature of the refrigerator. The door is opened 30 times a day and the refrigerator operates at 25% of ideal COP. The cost of work is ₹2.5 per kWh. What is the yearly bill of this refrigerator? The atmospheric temperature is at 30°C .

Temp of Refrigerated space = $-5^{\circ}\text{C} = 268 \text{ K}$.

[12 marks]



$$\therefore (\text{COP})_{\text{ideal}} = \frac{T_L}{T_H - T_L} = \frac{268}{35} = 7.657$$

$$\text{Actual COP} = (0.25) (\text{COP})_{\text{ideal}}$$

$$\boxed{(\text{COP})_{\text{actual}} = 1.914}$$

As the Refrigerator is opened 30 times a day,
 \therefore amount of heat input / day = $500 \times 30 = 15000 \text{ kJ/day}$

\therefore for a ~~month~~ year, heat input, $Q_1 = 15000 \times 365 \text{ kJ}$.

$$\therefore \left(\text{Coefficient} = \frac{Q_1}{W_{\text{input}}} \right) \Rightarrow W_{\text{input}} = \frac{15000 \times 365}{1.914}$$

$$W_{\text{input}} = 2860.501 \text{ MJ}$$

As the Cost of Electricity is Rs 2.5 for 1 kWh.
 i.e. 3600 kJ

$$\therefore \frac{3600 \text{ kJ}}{1} \rightarrow \text{Rs } 2.5$$

$$2860.501 \times 10^3 \rightarrow$$

$$\frac{2.5 \times 2860.501 \times 10^3}{3600}$$

$$\text{Total yearly cost} = \text{Rs } 1986.45$$

- Q.1 (c) A cylinder contains 0.15 m^3 of air at 1 bar and 80°C . It is compressed to 0.02 m^3 , during a polytropic process the final pressure being 12 bar. Calculate the increase in internal energy and heat transferred during compression. [12 marks]

Initial Condition	Compression	Final Condition	[12 marks]
$V_1 = 0.15 \text{ m}^3$ $P_1 = 1 \text{ bar}$ $T_1 = 353 \text{ K}$	\longrightarrow	$V_2 = 0.02 \text{ m}^3$ $P_2 = 12 \text{ bar}$ T_2	

Let n be the index of polytropic compression,

$$\therefore P V^n = C$$

$$\text{i.e. } \Rightarrow P_1 V_1^n = P_2 V_2^n$$

$$(1) [0.15]^n = (12) (0.02)^n$$

$$(7.5)^n = 12 \Rightarrow n \ln 7.5 = \ln 12$$

$$\Rightarrow n = 1.233$$

Using $PV = nRT$. (for ideal gas assumption),
 $\frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2} \Rightarrow T_2 = \left(\frac{P_2}{P_1}\right) \left(\frac{V_2}{V_1}\right) T_1 = \left(\frac{12}{1}\right) \left(\frac{0.02}{0.15}\right) (353)$

$$T_2 = 564.8 \text{ K}$$

~~Increase in internal energy~~ also, $P_1 V_1 = nRT_1$,
 $\Rightarrow M = \frac{n M_1}{RT_1} = \frac{(1200)(0.15)}{0.287 \times 353}$

$$M = 0.148 \text{ Kg}$$

Increase in Internal Energy

$$\begin{aligned} \Delta U &= m C_v (T_2 - T_1) \\ &= 0.148 \times 0.718 (564.8 - 353) \end{aligned}$$

$$\Delta U = 22.506 \text{ kJ}$$

~~Work transfer during the process~~

Heat transfer during the compression \Rightarrow

$$\begin{aligned} \text{Work transfer during the compression} &= \frac{P_2 V_2 - P_1 V_1}{n-1} \\ &= \frac{1200 \times 0.02 - (100)(0.15)}{0.233} = 38.626 \text{ kJ} \end{aligned}$$

By 1st law of thermodynamics

$$\Delta Q = \Delta U + \Delta W$$

$$= 22.506 - 38.626$$

$$\Delta Q = -16.12 \text{ kJ}$$

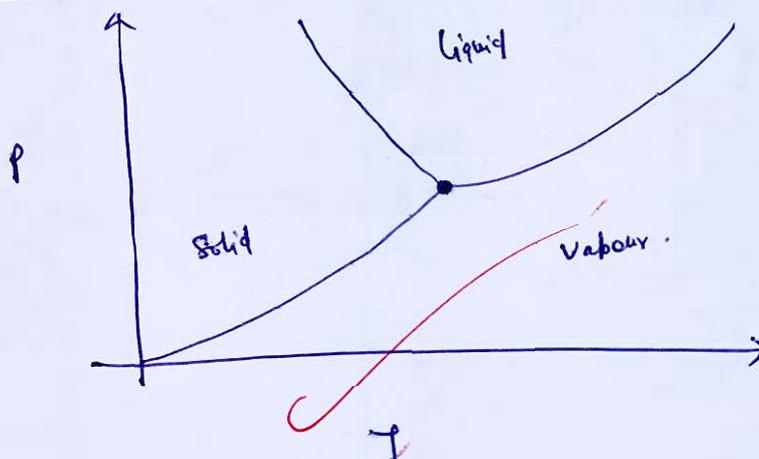
Rejection

(11)

- Q.1 (d) Sketch a P-T phase diagram for water and indicate the solid, liquid and vapour regions on it. Explain how this diagram differs from the phase diagram of other substances.

P-T phase diagram for water

[12 marks]



Water has a peculiar behaviour, that upon melting, from solid to liquid, its volume decreases. Unlike any other substance, where the volume increases upon melting.

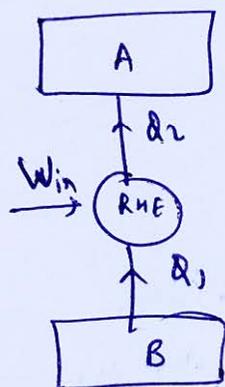
So, for water, $\left. \frac{\partial P}{\partial T} \right|_{\text{solid} \rightarrow \text{liquid}}$ is negative,

whereas the slope of P-T for other substances are positive.

(4)

- Q.1 (e) Two identical bodies of same heat capacity are at the same initial temperature T_1 . A reversible refrigerator operates between these two bodies, until one body is cooled to the temperature T_2 . If the bodies are at constant temperatures and do not undergo any change of phase, prove that the minimum amount of work needed by refrigerator is

$$W_{in} = C_p \left[\frac{T_1^2}{T_2} + T_2 - 2T_1 \right]$$



initial temp of both bodies = T_1 [12 marks]
Heat capacity, $C_A = C_B = C$ (given)

as final temp of body B = T_2 .
Let W_{in} be the amount of work consumed by the refrigerator.

for the minimum work, the Heat Engine should work Reversibly, and hence $(\Delta S)_{universe} = 0$.

$$\therefore (\Delta S)_A + (\Delta S)_{RHE} + (\Delta S)_B = 0$$

as Heat Engine is a cyclic process, $(\Delta S)_{RHE} = 0$.

$$\therefore, (\Delta S)_A + (\Delta S)_B = 0 \quad \text{--- (1)}$$

Amount of heat taken out from B to cool it from T_1 to T_2

$$Q_1 = C [T_1 - T_2]$$

Amount of heat gain by Body A = $W + Q_1$

Let T_f be the final temp of 'A',

$$\therefore, C [T_f - T_1] = W_{in} + C [T_1 - T_2]$$

$$T_f - T_1 = \frac{W_{in}}{C} + (T_1 - T_2)$$

$$T_f = \frac{W_{in}}{C} + 2T_1 - T_2 \quad \text{--- (2)}$$

$$\text{from (1), } (\Delta S)_A + (\Delta S)_B = 0$$

$$C \ln \frac{T_f}{T_1} - C \ln \frac{T_1}{T_2} = 0$$

$$\text{or } T_f = \frac{T_1 T_2}{T_2} \quad \text{--- (3)}$$

from (2) and (3)

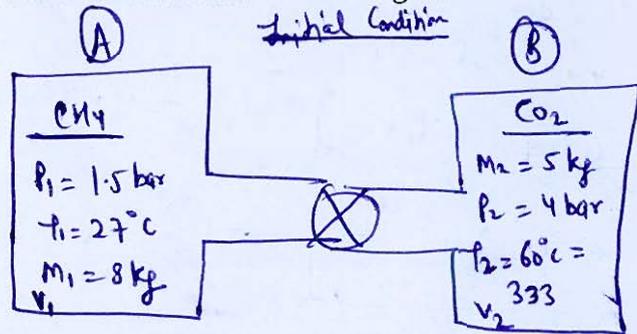
$$\frac{W_{in}}{C} + 2T_1 - T_2 = \frac{T_1 T_2}{T_2}$$

$$\frac{W_{in}}{C} = \frac{T_1 T_2}{T_2} + T_2 - 2T_1$$

$$(W_{in})_{min} = C \left[\frac{T_1 T_2}{T_2} + T_2 - 2T_1 \right]$$

11

- Q.2 (a) A tank containing 8 kg of methane at 1.5 bar and 27°C is connected to another tank containing 5 kg of CO₂ at 4 bar and 60°C through a valve. The valve is opened and the gases mix adiabatically. Determine the final pressure and temperature of the mixture and the irreversibility of the process, when environment temperature is 27°C. Given constant pressure specific heat of methane = 2.1 kJ/kgK and that of CO₂ = 0.85 kJ/kgK. Treat both the constituents as ideal gas.



[20 marks]

Using Ideal Gas
Relation, $PV = nRT$

Volume of tank A, V_1 ⇒

$$R_{CH_4} = \frac{R}{M_{CH_4}} = \frac{8.314}{16}, \quad P_1 V_1 = M_1 R T_1 \quad \Rightarrow \quad V_1 = (8) \left(\frac{8.314}{16} \right) \frac{300}{1.5 \times 10^5}$$

$$V_1 = 8.314 \text{ m}^3$$

Volume of tank B, V_2 ⇒

$$R_{CO_2} = \frac{R}{M_{CO_2}} = \frac{8.314}{44}$$

$$V_2 = \frac{M_2 R T_2}{P_2} = (5) \left(\frac{8.314}{44} \right) \frac{333}{4 \times 10^5}$$

$$V_2 = 0.786 \text{ m}^3$$

no. of moles of CH₄, $n_1 = \frac{M_1}{M_1} = \frac{8}{16} = 0.5 \text{ kg-mol}$

no. of moles of CO₂, $n_2 = \frac{m_2}{M_2} = \frac{5}{44} = 0.1136 \text{ kg-mol}$

upon mixing, let final pressure and temp be P_f, T_f .

As mixing is adiabatic, so, by energy conservation,
 $U_1 + U_2 = U_f$

~~Heat loss by CO_2 = Heat gain by Methane~~

~~Ans~~ As the system is closed system,

$$\underbrace{U_1}_{\text{Methane}} + \underbrace{U_2}_{CO_2} = \underbrace{U_f}_{\text{Mixed condition}} \quad \text{--- (1)}$$

$$(C_p)_{CH_4} = 2.1 \text{ kJ/kgK}, \quad (C_p - C_v = R)_{\text{methane}}$$

$$C_v = 2.1 - \frac{8.314}{16} \Rightarrow$$

$$(C_v)_{CH_4} = 1.58 \text{ kJ/kgK}$$

$$(C_p)_{CO_2} = 0.85 \text{ kJ/kgK} \Rightarrow C_v = (C_p - R)_{CO_2} = 0.85 - \frac{8.314}{44}$$

$$(C_v)_{CO_2} = 0.66 \text{ kJ/kgK}$$

So, from (1) $m_1 (C_v)_{CH_4} T_1 + m_2 (C_v)_{CO_2} T_2 = m_1 (C_v)_{CH_4} T_f + m_2 (C_v)_{CO_2} T_f$

$$T_f = \frac{(8)(1.58)[300] + (5)(0.66)[333]}{(8)(1.58) + (5)(0.66)}$$

$$T_f = 306.83 \text{ K} \rightarrow \text{Answer}$$

Now using $PV = n\bar{R}T$

$$(P_f) V_f = (n_1 + n_2) \bar{R} T_f$$

$$\text{here } V_f = V_1 + V_2 = 8.314 + 0.786 = 9.1 \text{ m}^3$$

$$\text{So, } (P_f) [9.1] = (n_1 + n_2) \bar{R} (T_f)$$

$$(P_f) (9.1) = (0.5 + 0.1136) (8.314) (306.83)$$

$$P_f = 1.72 \text{ bar} \rightarrow \text{Answer}$$

Irreversibility of the process :-

$$\Delta S_{\text{universe}} = \Delta S_{\text{system}} + \Delta S_{\text{surroundings}} \quad \left[\text{as no heat is transferred to the surroundings} \right]$$

↓
taking tanks as system.

$$\Delta S_{\text{system}} = S_f - (S_1 + S_2) = \Delta S_{\text{CH}_4} + \Delta S_{\text{CO}_2}$$

$$P_{f1}, \text{ Partial pressure of Methane upon mixing} = \left(\frac{n_1}{n_1 + n_2} \right) P_f = \left(\frac{0.5}{0.5 + 1.136} \right) 1.2 = 1.401 \text{ bar}$$

$$P_{f2}, \text{ Partial pressure of CO}_2 \text{ upon mixing} = \left(\frac{n_2}{n_1 + n_2} \right) P_f = \left(\frac{1.136}{0.5 + 1.136} \right) 1.2 = 0.318 \text{ bar}$$

$$\Delta S_{\text{CH}_4} = M_1 \left[C_p \ln \frac{T_f}{T_1} - R \ln \frac{P_{f1}}{P_1} \right]$$

$$\Delta S_{\text{CH}_4} = [8] \left[2.1 \ln \frac{306.83}{300} - \frac{8.314}{16} \ln \frac{1.401}{1.5} \right] = 0.662 \text{ kJ/k}$$

$$\Delta S_{\text{CH}_4} = 0.662 \text{ kJ/k}$$

$$\Delta S_{\text{CO}_2} = M_2 \left[C_p \ln \frac{T_f}{T_2} - R \ln \frac{P_{f2}}{P_2} \right]$$

$$= [5] \left[0.85 \ln \frac{306.83}{333} - \frac{8.314}{44} \ln \frac{0.318}{4} \right]$$

$$\Delta S_{\text{CO}_2} = 2.044 \text{ kJ/k}$$

$$\Delta S_{\text{universe}} = \Delta S_{\text{CH}_4} + \Delta S_{\text{CO}_2} = 0.662 + 2.044$$

$$\Delta S_{\text{universe}} = 2.666 \text{ kJ/k}$$

$$\dot{I} = T_0 \Delta S_{\text{universe}}$$

$$= (300) [2.666] = 799.8 \approx 800 \text{ kJ}$$

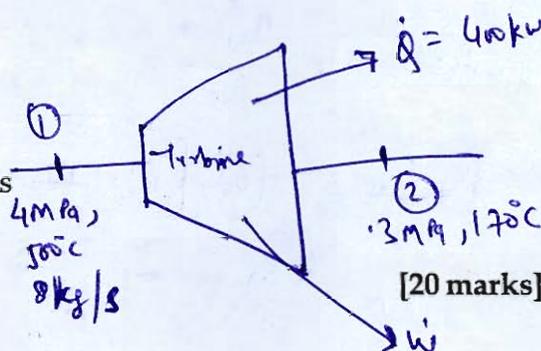
$$\dot{I} = 799.8 \approx 800 \text{ kJ}$$

17

Q.2 (b) The steam enters at steam turbine at 4 MPa and 500°C at a rate of 8 kg/s and exits at 0.3 MPa and 170°C. The steam is losing heat to the surroundings at 100 kPa and 25°C at a rate of 400 kW. The changes in kinetic and potential energies are negligible. Calculate

- (i) actual power output
- (ii) the reversible work
- (iii) second law efficiency
- (iv) the availability of steam at inlet conditions

[Refer steam table attached]



~~To look to~~
Atmospheric Condition $\Rightarrow T_0 = 298 \text{ K}$
 $P_0 = 100 \text{ kPa}$.

for the condition of steam at ① (Inlet)

at 4 MPa, $T_{sat} = 250.354^\circ\text{C} < 500^\circ\text{C}$

\therefore the steam is superheated at inlet to the turbine, \therefore from steam table,

$$h_1 = 3446 \text{ kJ/kg}, \quad S_1 = 7.0922 \text{ kJ/kgK}$$

for the condition of steam at ② (outlet)

at 0.3 MPa, $T_{sat} = 133.522^\circ\text{C} < 170^\circ\text{C}$, \therefore the steam is superheated at outlet also,

\therefore for 0.3 MPa, 170°C,

$$h_2 = 2803.2 \text{ kJ/kg}$$

$$S_2 = 7.1773 \text{ kJ/kgK}$$

① Actual Power output :-

By energy conservation, (neglecting change in KE and PE).

$$\dot{m}_1(h_1) - \dot{Q} - \dot{W} = \dot{m}_2 h_2$$

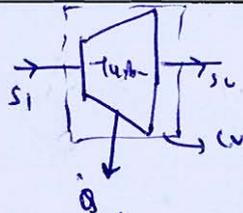
$$\dot{W} = \dot{m}_1(h_1 - h_2) - \dot{Q}$$

$$= (8) [3446 - 2803.2] - 400$$

$$\dot{W}_{\text{actual}} = 4.7384 \text{ MW}$$

② Reversible work (w_{max})

Irreversibility in the process, $i = T_0 (\Delta S)_{\text{universe}} = T_0 (\Delta S)_{\text{gen}}$.



$$\frac{ds_{cv}}{dt} = S_1 + S_{gen} - S_2 - \frac{Q}{T_0}$$

(Assuming heat loss from turbine occurs at $T_0 = 298 \text{ K}$).

for steady state, $\frac{ds_{cv}}{dt} = 0$, $S_{gen} = S_2 + \frac{Q}{T_0} - S_1$

$$S_{gen} = (8) [7.1773 - 7.0322] + \frac{400}{298}$$

$$S_{gen} = 2.023 \text{ kJ/K}$$

$$I = T_0 S_{gen} = (298) (2.023) = 602.854 \text{ kJ}$$

∴, Reversible work, $W_{rev} = W_{act} + I$
 $= 4738.4 + 602.854$

$$W_{rev} = 5341.254 \text{ kJ}$$

$$W_{rev} = 5.341 \text{ MJ}$$

10

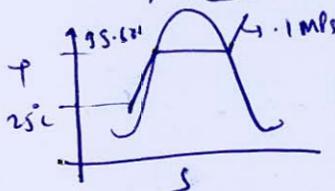
(iii) η_{II}

$$\eta_{II} = \frac{W_{actual}}{W_{rev}} = \frac{4.738}{5.341} = 88.709\%$$

(iv) Availability of steam at initial condition

at atmospheric condition, 0.1 MPa , $T_{sat} = 99.617^\circ \text{C}$, so, at the atmospheric condition, steam is ~~subcooled~~ subcooled.

∴, ~~at atmospheric condition, $h_0 = h_f$~~ $h_0 = h_f$



∴, condition of steam at 0.1 MPa , 25°C can be taken as condition of saturated water at 25°C

∴, $h_0 = h_f = 104.83 \text{ kJ/kg}$, $s_0 = s_f = 0.36722 \text{ kJ/kgK}$

∴, Availability of steam at inlet $= m [(h_1 - T_0 s_1) - (h_0 - T_0 s_0)]$

$$= [(h_1 - h_0) + T_0 (s_0 - s_1)] m$$

$$= [3446 - 104.83] + 298 [0.36722 - 7.0322] (8)$$

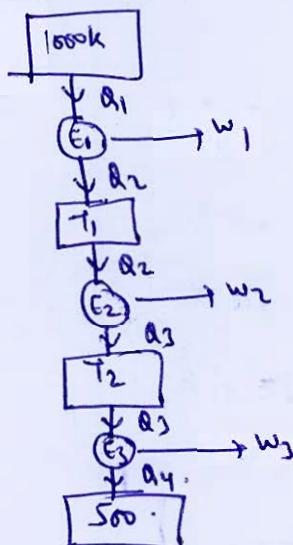
$$= 10.697 \text{ MJ [answer]}$$

- Q.2 (c) Three Carnot engines E_1 , E_2 and E_3 operate in series between two heat reservoirs, which are at temperatures of 1000 K and 500 K. Calculate the temperature of the intermediate reservoirs if the amount of work produced by these engines in the proportion of 7 : 5 : 4.

[20 marks]

Given

$$W_1 : W_2 : W_3 = 7 : 5 : 4$$



for Carnot Engine 1,

$$\eta_{E1} = 1 - \frac{T_1}{1000}$$

$$\therefore 1 - \frac{T_1}{1000} = \frac{W_1}{Q_1}$$

$$W_1 = \left(1 - \frac{T_1}{1000}\right) Q_1$$

$$\text{now } Q_2 = Q_1 - W_1 = Q_1 - \left(1 - \frac{T_1}{1000}\right) Q_1 = Q_1 \frac{T_1}{1000}$$

$$Q_2 = Q_1 \frac{T_1}{1000}$$

for Carnot Engine 2

$$\eta_{E2} = 1 - \frac{T_2}{T_1} = \frac{W_2}{Q_2}$$

$$W_2 = \left(1 - \frac{T_2}{T_1}\right) \left(Q_2 \frac{T_1}{1000}\right) = \left(1 - \frac{T_2}{T_1}\right) \left(Q_1 \frac{T_1}{1000}\right)$$

$$\text{also, } Q_3 = Q_2 - W_2 = Q_2 - \left(1 - \frac{T_2}{T_1}\right) Q_2 = Q_2 \frac{T_2}{T_1}$$

$$\therefore Q_3 = Q_1 \left(\frac{T_1}{1000}\right) \left(\frac{T_2}{T_1}\right) \Rightarrow Q_3 = Q_1 \frac{T_2}{1000}$$

~~for Carnot Engine 3, $\eta_{E3} = 1 - \frac{500}{T_2} = \frac{W_3}{Q_3}$~~

~~$$W_3 = \left(1 - \frac{500}{T_2}\right) Q_1 \left(\frac{T_2}{1000}\right)$$~~

for Circuit type 3 $\eta_{ES} = 1 - \frac{500}{t_2} = \frac{w_3}{Q_3}$

$$w_3 = \left(1 - \frac{500}{t_2}\right) Q_3$$

$$w_3 = \left(1 - \frac{500}{t_2}\right) \left[Q_1 \frac{t_2}{1000}\right]$$

$$\text{So, } w_3 = \left(1 - \frac{t_1}{1000}\right) Q_1 \quad , \quad w_2 = \left(1 - \frac{t_2}{t_1}\right) \frac{Q_1 t_1}{1000}$$

$$\text{As } \frac{w_1}{w_2} = \frac{7}{5}$$

$$\text{So, } \frac{1 - \frac{t_1}{1000}}{\left(1 - \frac{t_2}{t_1}\right) \frac{t_1}{1000}} = \frac{7}{5}$$

$$5 - \frac{5t_1}{1000} = 7 \left[\frac{t_1}{1000} - \frac{t_2}{1000} \right]$$

$$5 = \frac{12t_1}{1000} - \frac{7t_2}{1000}$$

$$12t_1 - 7t_2 = 5000 \quad \text{--- (1)}$$

$$\text{also, } \frac{w_2}{w_3} = \frac{5}{4}$$

$$\frac{\left(1 - \frac{t_2}{t_1}\right) \frac{t_1}{1000}}{\left(1 - \frac{500}{t_2}\right) \frac{t_2}{1000}} = \frac{5}{4}$$

$$\frac{T_1 - T_2}{T_2 - 500} = \frac{5}{4}$$

$$4T_1 - 4T_2 = 5T_2 - 2500$$

$$4T_1 - 9T_2 = -2500 \quad \text{--- (1)}$$

So, $T_2 = \frac{2500 + 4T_1}{9}$, putting this in (1)

Solving ~~(1) and (2)~~

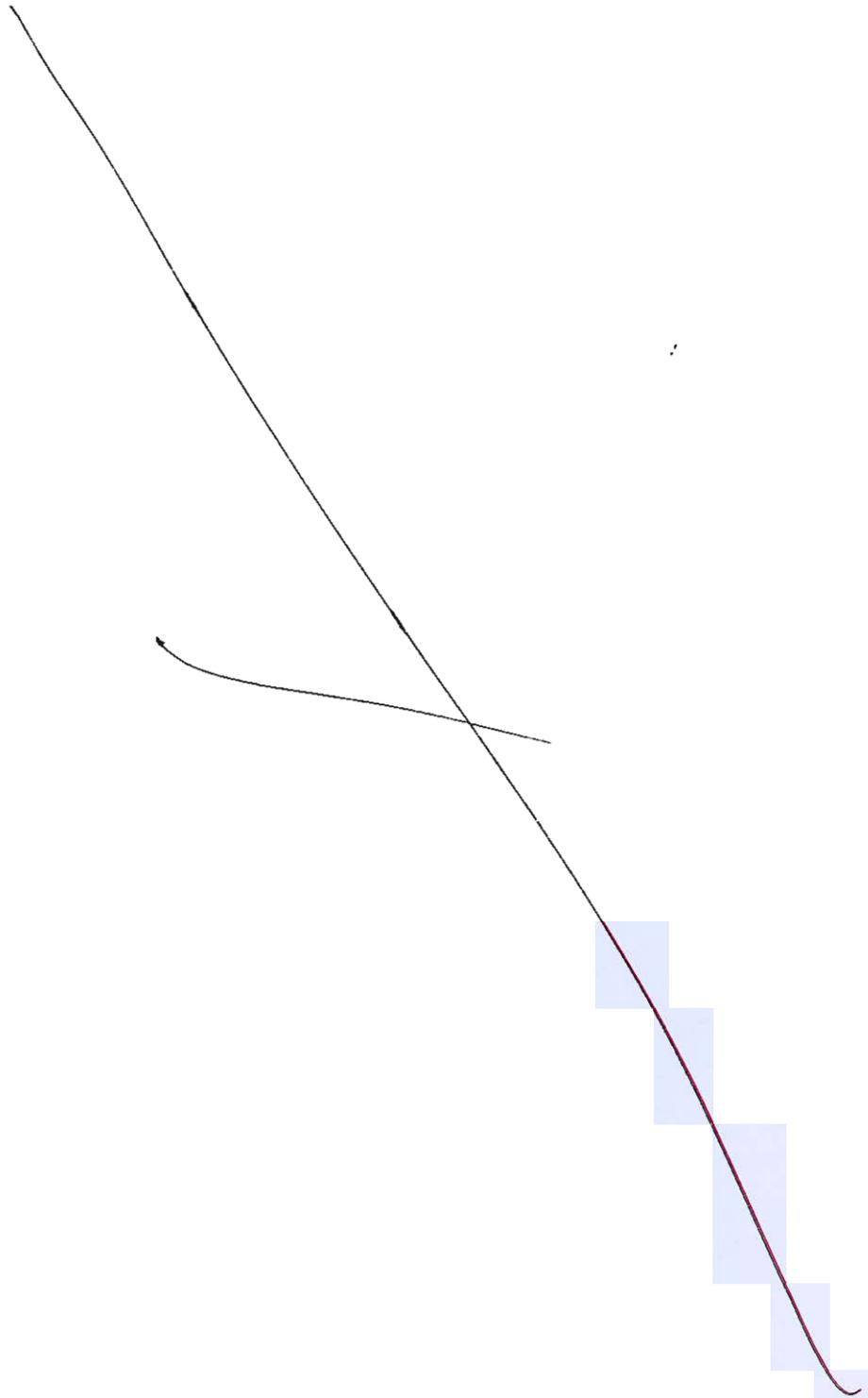
$$T_1 = 781.25 \text{ k}$$

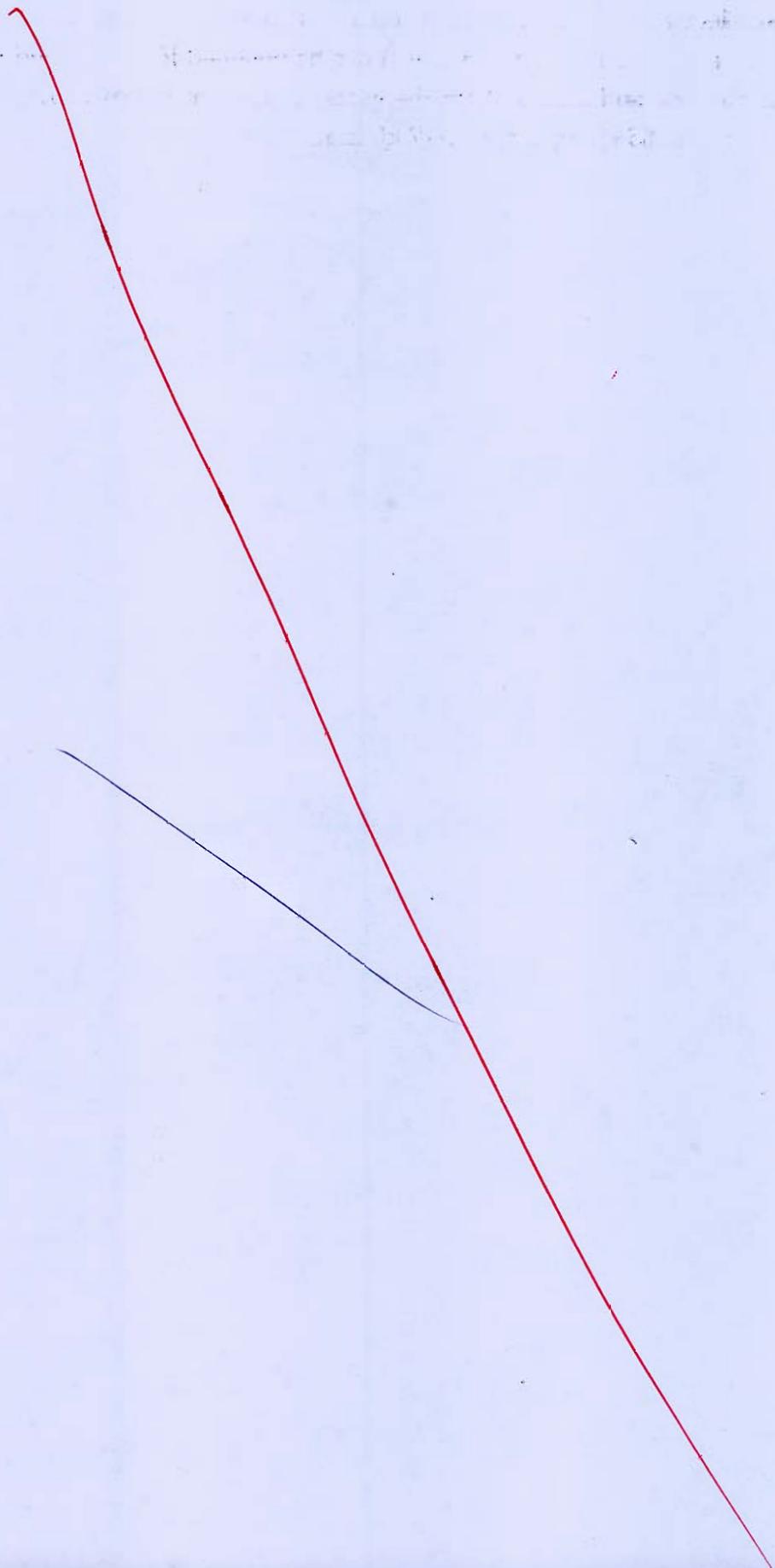
$$T_2 = 625 \text{ k}$$

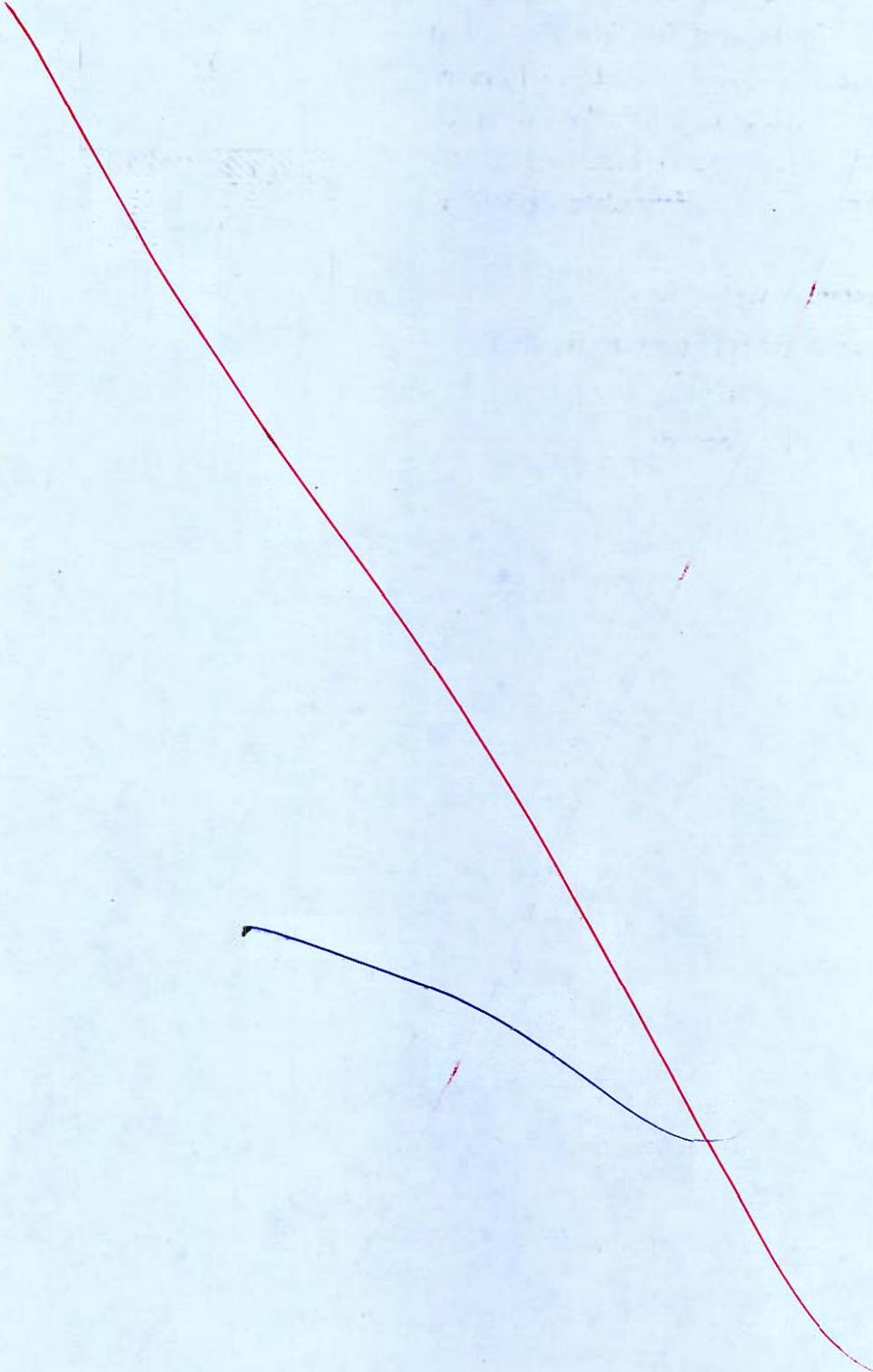
18

- Q.3 (a) In a process industry hot gases are delivered by different units. One unit delivers a gas A at 1 bar and 1200 K at a rate of 1.5 kg/sec while a second unit delivers a gas B at 1 bar and 900 K at a rate of 2 kg/sec. These hot gases are usually cooled to 300 K in heat exchangers. The ambient atmosphere is at 300 K. An engineer plans to use the hot gases as source and ambient atmosphere as sink to operate a heat engine and thus obtain some power. Calculate the maximum power that can be obtained if gases A and B are used as separate sources, and assume both the gases A and B are ideal with $\gamma = 1.4$, $R = 0.287 \text{ kJ/kgK}$, $c_p = 1.005 \text{ kJ/kgK}$, $c_v = 0.717 \text{ kJ/kgK}$.

[20 marks]





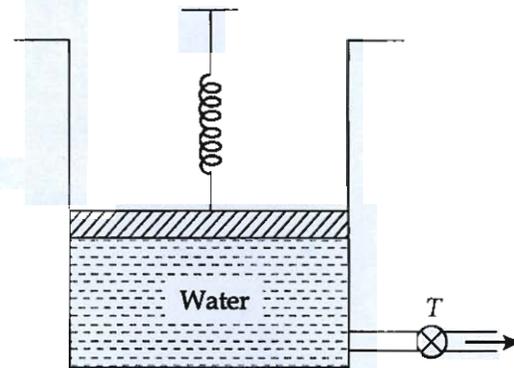


Q.3 (b) A cylinder containing 30 kg of saturated liquid water at 80°C has the piston restrained by a linear spring as shown in figure below.

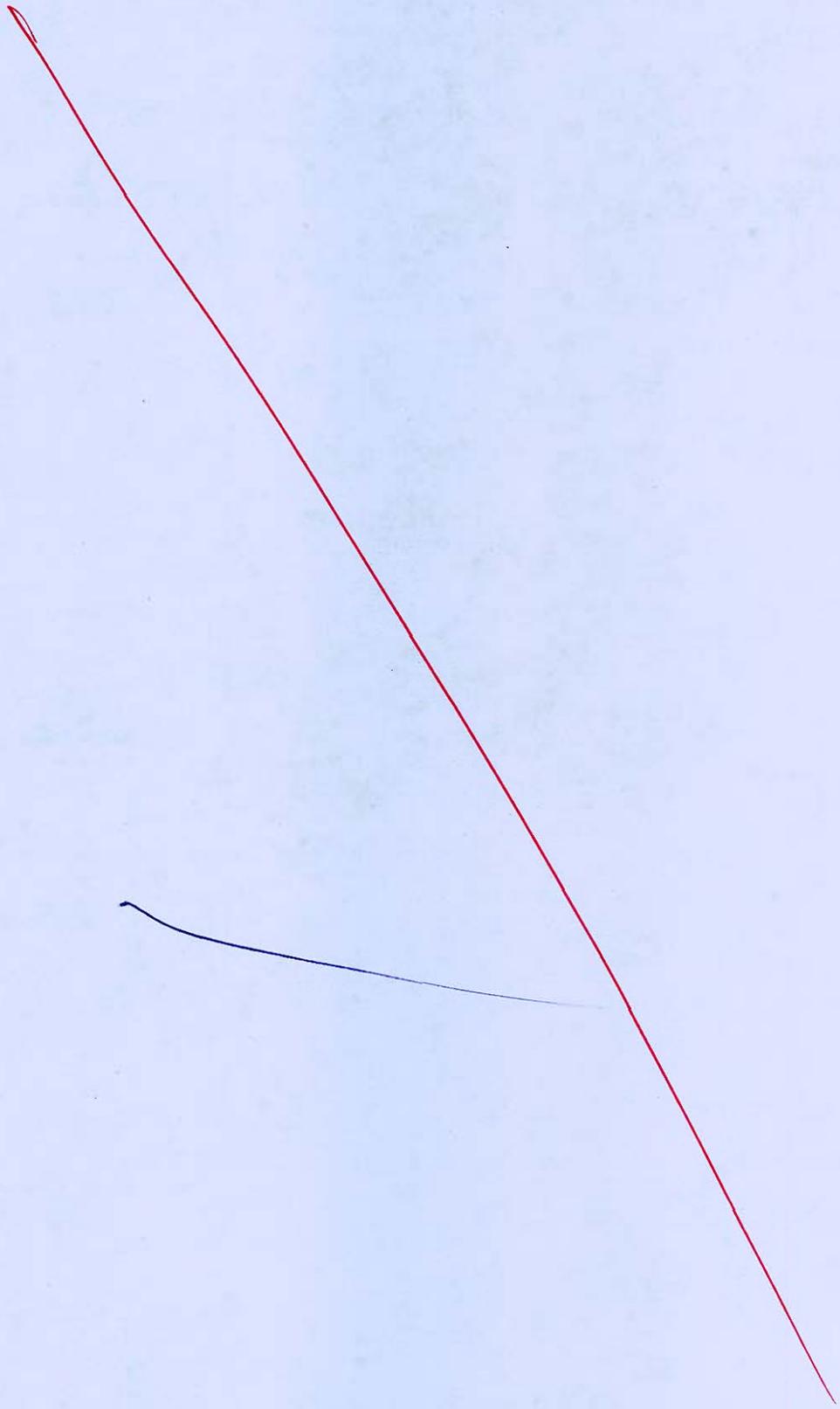
The valve fitted near the bottom is opened to allow 10 kg of saturated liquid to flow out at 60°C . During this process, heat transfer from the surrounding is allowed and at the end of the process, the residual liquid is saturated at 40°C . The area of cross-section of the piston is 500 cm^2 . Estimate the following:

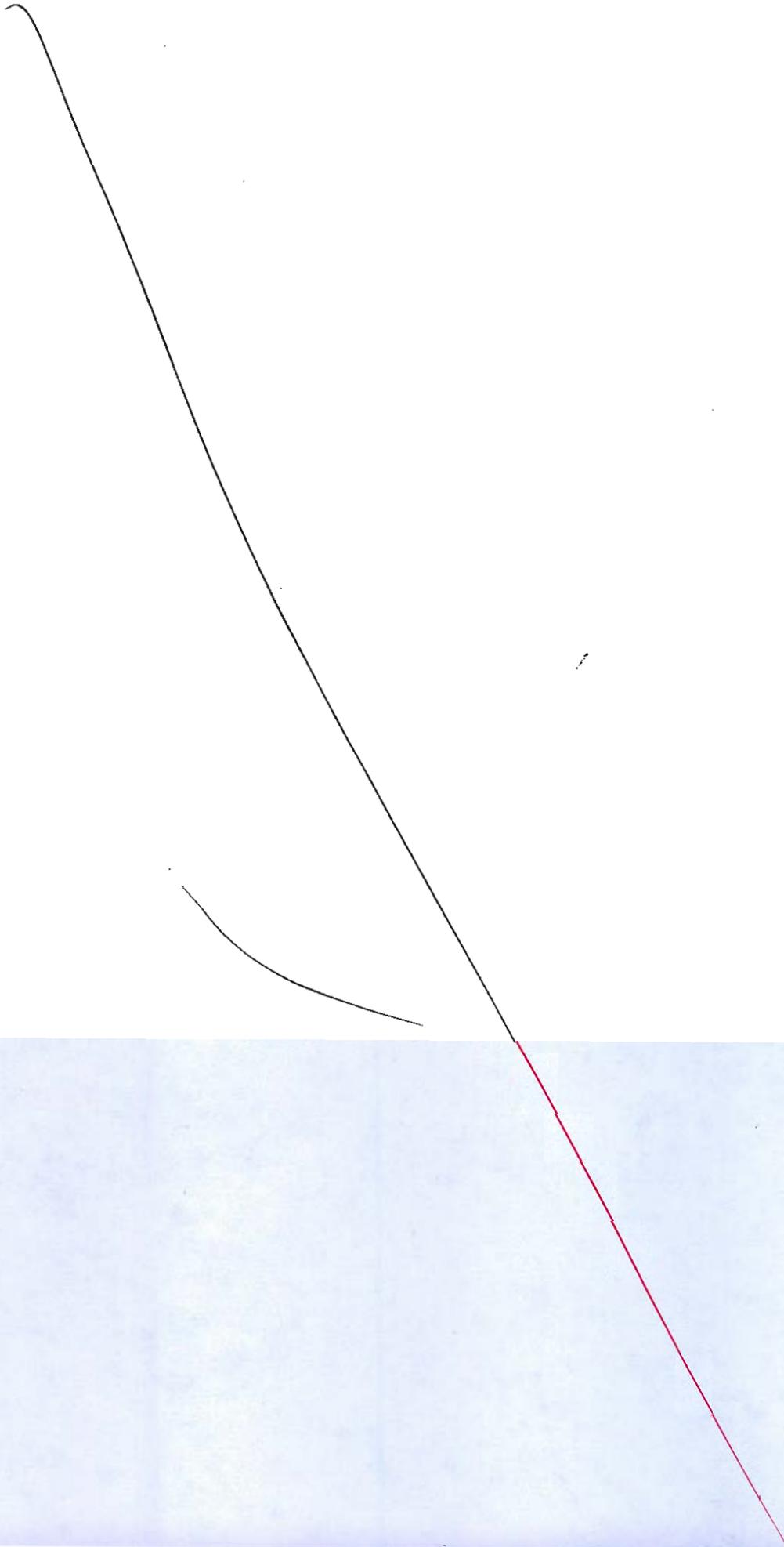
- (i) Final state of water,
- (ii) The magnitude of heat transfer, and
- (ii) the spring constant

[Refer steam table attached]



[20 marks]

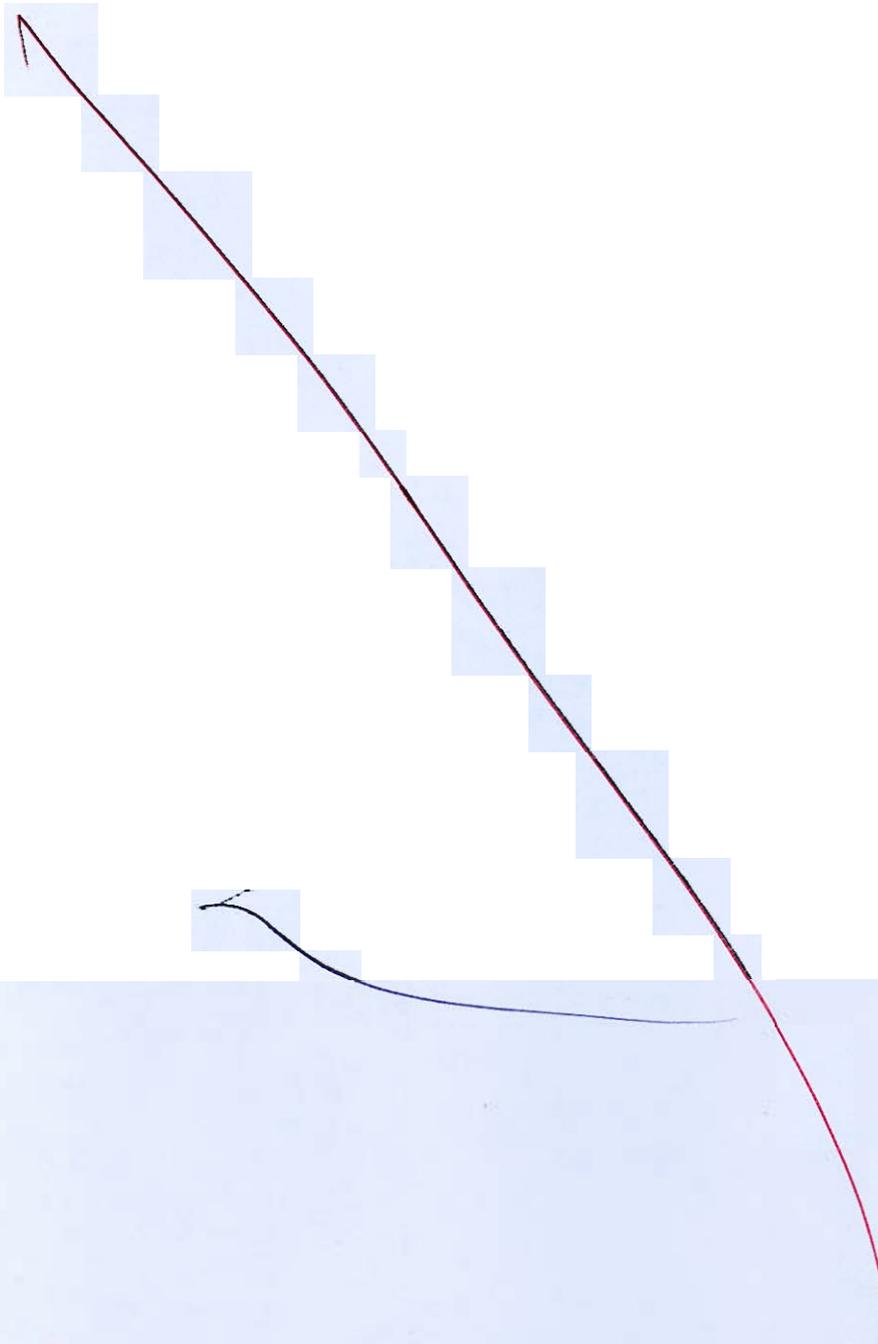


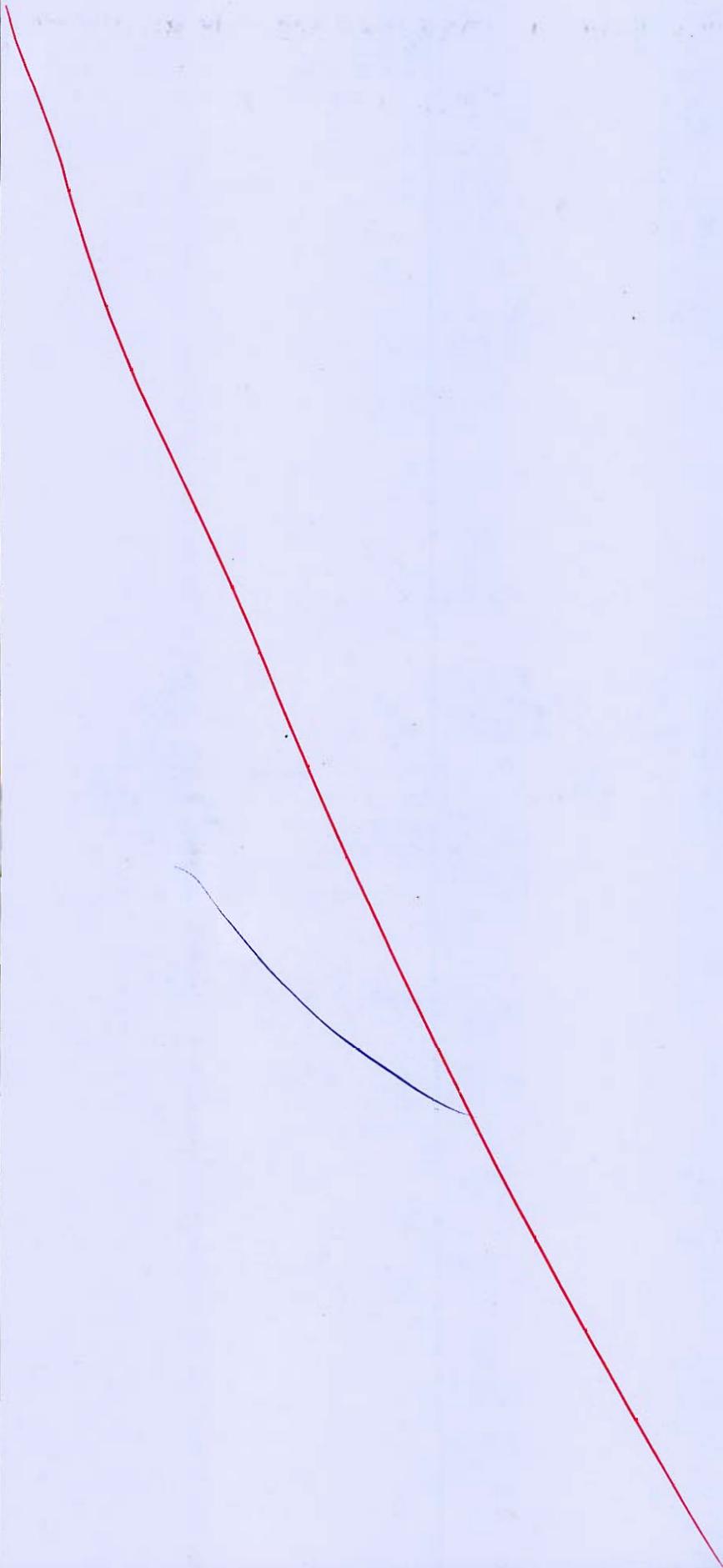


- Q.3 (c) A rigid cylinder with a volume of 3.5 m^3 contains air at 200 kPa and 27°C . The heat is transferred to air from a constant temperature heat source at 1800 K and air in the cylinder is heated to 900 K . The atmosphere is at 1 bar and 17°C . Calculate the initial and final availability of air, maximum useful work and **irreversibility**. Neglect changes in kinetic and potential energies.

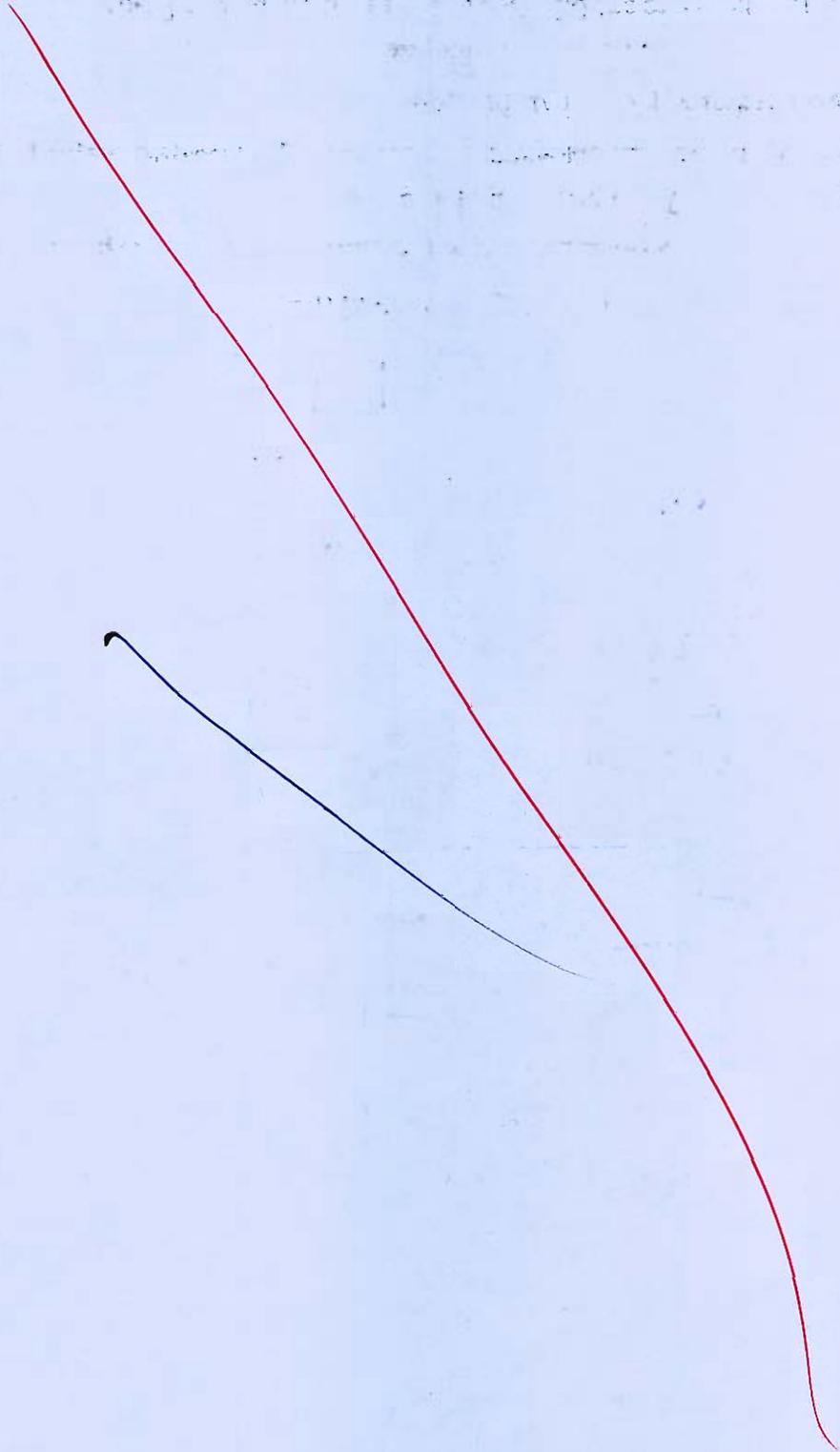
(For Air take, $c_p = 1.005 \text{ kJ/kgK}$, $c_v = 0.717 \text{ kJ/kgK}$, $R = 0.287 \text{ kJ/kgK}$)

[20 marks]





[Faint, illegible text, likely bleed-through from the reverse side of the page]

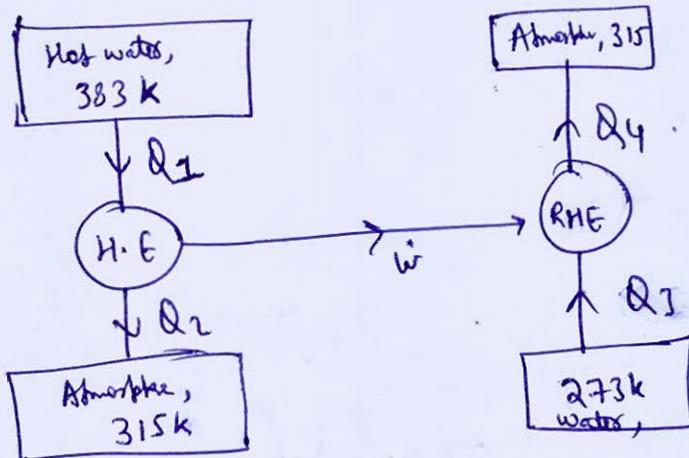


Q.4 (a) It is proposed to produce 1500 kg of ice per hour from liquid water at 0°C in summer when the ambient atmospheric temperature is 42°C . It is planned to use a heat engine to operate the refrigeration plant. Hot water at 110°C which is produced by solar heating, may be used as a source to supply energy as heat to the heat engine and the engine uses the ambient atmospheric air as sink. Calculate

- (i) the power required by the refrigeration plant
- (ii) the ratio of the energy extracted as heat from the freezing water to the energy absorbed as heat by the heat engine and
- (iii) the rate at which energy is rejected to the ambient atmosphere by both the devices.

The enthalpy of fusion of water at 0°C is 333.43 kJ/kg .

[20 marks]



① Power required by the refrigeration plant :-

Heat to be taken out from water at 0°C to form ice = $(m_{\text{ice}}) L_{\text{fusion}}$

$$Q_3 = \left(\frac{1500}{3600} \right) (333.43) = 138.929 \text{ kW}$$

For a Reversible heat engine, $(\text{Cor})_{\text{max}} = (\text{Cor})_{\text{min}} = \frac{T_2}{T_1 - T_2}$

$$\frac{273}{42} = \frac{Q_3}{W_{\text{input}}}$$

$$W_{\text{input}} = 138.929 \left(\frac{42}{273} \right) \Rightarrow W_{\text{in}} = 21.37 \text{ kW}$$

②

$$\frac{Q_3}{Q_2} = ?$$

for Heat Engine,

$$\eta_{\text{max}} = \eta_{\text{Carnot}} = 1 - \frac{T_2}{T_1} = 1 - \frac{315}{383}$$

$$\eta_{\text{HE}} = 0.1775$$

$$2, \quad 1775 = \frac{Q_2}{\dot{w}} \Rightarrow Q_2 = \dot{w} \cdot 1775$$

$$\text{as } \dot{w} = 21.38 \Rightarrow Q_1 = \frac{21.38}{1775} \Rightarrow Q_1 = 120.39 \text{ kw}$$

$$\text{So, } \frac{Q_3}{Q_2} = \frac{138.928}{120.39} \Rightarrow \frac{Q_3}{Q_1} = 1.154$$

$$(iii) \quad Q_2 + Q_4 = ?$$

$$Q_2 = \dot{Q}_1 - \dot{w} = 120.39 - 21.38 = 99.02 \text{ kw.}$$

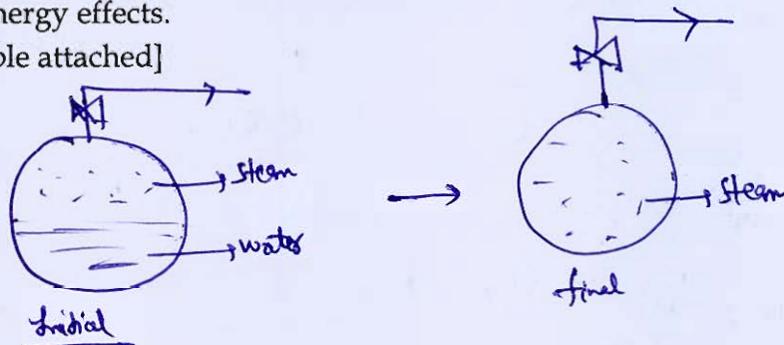
$$Q_4 = \dot{Q}_3 + \dot{w} = 138.928 + 21.38 = 160.298 \text{ kw}$$

$$2, \quad Q_2 + Q_4 = 259.319 \text{ kw}$$

19

- Q.4 (b) A tank having a volume of 0.95 m^3 initially contains water as a two phase liquid-vapour mixture at 300°C and a quality of 0.8 . Saturated water vapour at 300°C is slowly withdrawn through a pressure regulating valve at the top of the tank as energy is transferred by heat to maintain the pressure constant in the tank. This continues until the tank is filled with saturated vapour at 300°C . Determine the amount of heat transfer. Neglect all kinetic and potential energy effects.

[Refer steam table attached]



[20 marks]

$$V_{\text{tank}} = V_1 = 0.95 \text{ m}^3, x_1 = 0.8$$

from steam table, at 300°C , $v_f = 0.00140423 \text{ m}^3/\text{kg}$, $v_g = 0.021660 \text{ m}^3/\text{kg}$

$$v_1 = v_f + x_1 v_g = 0.00140423 + (0.8)(0.021660 - 0.00140423)$$

initial specific vol, $v_1 = 0.01766 \text{ m}^3/\text{kg}$

∴ let total mass be M ∴ $(M)(0.01766) = V_1$

$$M = \frac{0.95}{0.01766} \Rightarrow M = 53.97 \text{ kg}$$

also, $x = \frac{M_v}{M} \Rightarrow M_v = (0.8) 53.97 = 43.17 \text{ kg}$

∴ $M_{\text{steam}} = M_v = 43.17 \text{ kg}$

Initial Mass of steam and water in tank
∴ $M_{\text{water}} = M - M_v = 10.79 \text{ kg}$

for final condition, at 300°C , $v_g = 0.021660 \text{ m}^3/\text{kg}$

$$M_{v2} = \frac{V_1}{v_g} = \frac{0.95}{0.021660} \Rightarrow M_{v2} = 43.85 \text{ kg}$$

By Energy Conservation

$$U_1 - h_o + Q = U_2$$

\downarrow initial internal energy \downarrow enthalpy of steam taken out \downarrow final internal energy

initial internal energy

enthalpy of steam taken out

final internal energy

$$\text{Mass of steam taken out} = 53.97 - 43.85 = 10.12 \text{ kg.}$$

$$u_1 = u_f + x u_{fg}$$

$$= 1332.9 + (0.8) [2563.6 - 1332.9]$$

$$u_1 = 2312.46 \text{ kJ/kg}$$

$$h_o = h_g \Big|_{300^\circ\text{C}} \Rightarrow h_o = 2749.6 \text{ kJ/kg}$$

$$u_2 = u_g \Big|_{300^\circ\text{C}} = 2563.6 \text{ kJ/kg.}$$

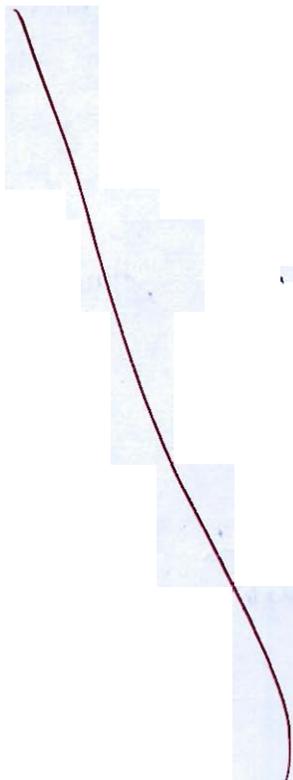
$$\therefore Q = (u_2 - u_1) + h_o$$

$$= (43.85) [2563.6] - (53.97) [2312.46]$$

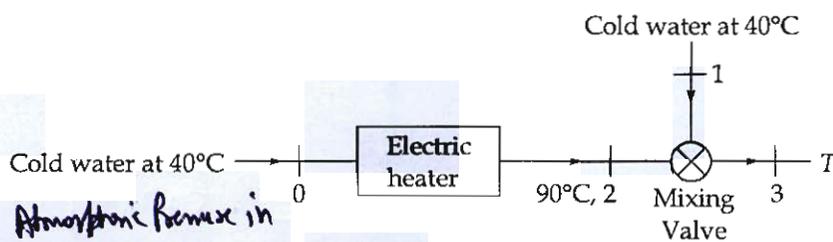
$$+ (10.12) [2749.6]$$

20

$$Q = 15.16 \text{ MJ} \rightarrow \text{Heat addition}$$

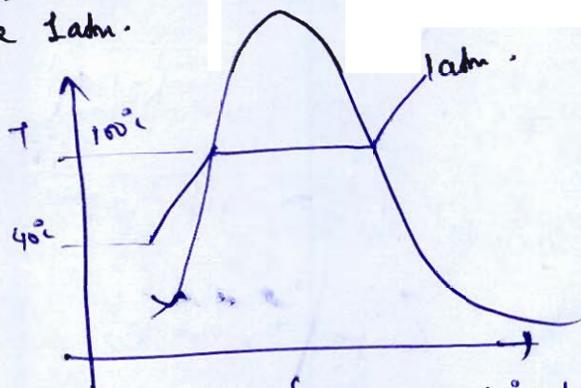


Q.4 (c) In a laboratory, hot water stream at any desired temperature T can be produced by mixing cold water at 40°C (State 1) with hot water stream leaving the electric heater at 90°C (State 2) as shown in figure. Determine the entropy generation rate for producing a hot water stream of 1.5 kg/min at 60°C (State 3) by this arrangement. Neglect the small effect of pressure on subcooled water properties.
 [Refer steam table attached]



Let the Atmospheric Pressure in laboratory be 1 atm .

[20 marks]



As at 1 atm , $T_{\text{sat}} = 100^\circ\text{C} > 40^\circ\text{C}$ and 90°C , both water are in subcooled conditions,
 As it is given do neglect effect of pressure on subcooled water properties,

Properties of water at 40°C = Properties of Saturated water at 40°C
 from steam table
 $h_0 = h_1 = 167.53 \text{ kJ/kg}$

$$s_0 = s_1 = 0.57240 \text{ kJ/kgK}$$

Properties of water at 90°C = Properties of Saturated water at 90°C

$$h_2 = 377.04 \text{ kJ/kg}$$

$$s_2 = 1.1928 \text{ kJ/kgK}$$

for Across Electric heater, let m_1 be the mass flow rate of water at condition 1 in kg/min



for steady state, $s_0 + s_{gen} = s_2$

$$s_{gen} |_{\text{Electric heater}} = s_{gen,1} = s_2 - s_1$$

$$= m_1 [1.1928 - 0.57240]$$

$$s_{gen,1} = 0.6205 m_1 \text{ kJ/minK} \quad \text{--- (1)}$$

let m_2 be the mass flow rate of water at condition 2, (1)

upon mixing, let entropy gen be $s_{gen,2}$

$$m_1 s_1 + m_2 s_2 + s_{gen,2} = s_3$$

for S_2 , the properties of water = properties of saturated water at 60°C .

$$h_3 = 251.18 \text{ kJ/kg}, \quad s_3 = 0.83129 \text{ kJ/kgK}$$

$$S_2, \quad S_{\text{gen},2} = (1.5) [0.83129] - \dot{m}_1 (0.57240) - \dot{m}_0 (1.1929)$$

$$S_{\text{gen},2} = 1.2469 - \dot{m}_1 (0.57240) - \dot{m}_0 (1.1929) \text{ kJ/min} \quad (11)$$

$$\text{Now, } \dot{m}_1 + \dot{m}_0 = 1.59 \quad (11)$$

$$\text{also, } \dot{m}_1 h_1 + \dot{m}_0 h_2 = (1.59) h_3$$

$$(\dot{m}_1) (162.53) + (\dot{m}_0) (377.04) = 1.59 (251.18)$$

$$\dot{m}_1 + 2.25 \dot{m}_0 = 2.38 \quad (12)$$

Solving (11) & (12)

$$\dot{m}_1 = 0.95 \text{ kg/min}$$

$$\dot{m}_0 = 0.64 \text{ kg/min}$$

$$S_2, \text{ total entropy generation} = S_{\text{gen},1} + S_{\text{gen},2}$$

$$= 0.6205 \dot{m}_0 + 1.2469 - \dot{m}_1 (0.57240) - \dot{m}_0 (1.1929)$$

$$= 0.6205 (0.64) + 1.2469 - 0.95 (0.57240) - 0.64 (1.1929)$$

$$(D_s)_{\text{gen}} = 0.336 \text{ kJ/min-K} \rightarrow \text{Answer}$$

$$(D_s)_{\text{gen}} = 20.202 \text{ kW/K}$$

(8)

Saturated Water and Steam (Temperature-based)

T °C	P _{sat} MPa	Volume, m ³ /kg v _f	Energy, kJ/kg u _f	Enthalpy, kJ/kg h _f	Entropy, kJ/(kg K) s _f	Volume, m ³ /kg v _g	Energy, kJ/kg u _g	Enthalpy, kJ/kg h _g	Entropy, kJ/(kg K) s _g
0.01	0.0006117	0.00100021	295.991	0	0	237.19	0.00	2300.9	2500.9
1	0.0006571	0.00100015	296.339	4.18	0.01526	237.62	4.18	2502.7	2504.6
2	0.0007069	0.00100011	297.758	8.39	0.03061	239.06	8.39	2504.6	2506.2
3	0.0007581	0.00100008	299.000	12.60	0.04589	240.51	12.60	2506.3	2507.9
4	0.0008135	0.00100007	301.116	16.81	0.06110	242.02	16.81	2508.2	2509.1
5	0.0008732	0.00100008	303.633	21.02	0.07625	243.59	21.02	2510.1	2510.9
6	0.0009354	0.00100011	306.633	25.22	0.09134	245.22	25.22	2511.9	2512.7
7	0.0010021	0.00100014	310.129	29.43	0.10637	246.91	29.43	2513.7	2514.5
8	0.0010730	0.00100020	314.130	33.63	0.12133	248.66	33.63	2515.6	2516.3
9	0.0011483	0.00100026	318.641	37.82	0.13624	250.47	37.82	2517.4	2518.1
10	0.0012282	0.00100035	323.663	42.02	0.15109	252.34	42.02	2519.2	2519.9
11	0.0013130	0.00100041	329.199	46.22	0.16587	254.28	46.22	2521.0	2521.8
12	0.0014028	0.00100055	335.259	50.41	0.18061	256.28	50.41	2522.9	2523.7
13	0.0014981	0.00100067	341.954	54.60	0.19528	258.34	54.60	2524.7	2525.5
14	0.0015990	0.00100080	349.311	58.79	0.20990	260.46	58.79	2526.5	2527.3
15	0.0017058	0.00100094	357.356	62.98	0.22446	262.64	62.98	2528.3	2529.1
16	0.0018188	0.00100110	366.111	67.17	0.23897	264.88	67.17	2530.2	2530.9
17	0.0019381	0.00100127	375.599	71.36	0.25343	267.18	71.36	2532.0	2532.7
18	0.0020637	0.00100145	385.841	75.54	0.26784	269.54	75.54	2533.8	2534.5
19	0.0021958	0.00100164	396.867	79.73	0.28221	271.96	79.73	2535.6	2536.2
20	0.0023343	0.00100184	408.707	83.91	0.29655	274.44	83.91	2537.4	2537.9
21	0.0024792	0.00100205	421.383	88.10	0.31087	276.98	88.10	2539.3	2539.5
22	0.0026306	0.00100228	434.838	92.28	0.32518	279.58	92.28	2541.1	2541.7
23	0.0027885	0.00100254	449.114	96.46	0.33948	282.24	96.46	2542.9	2543.5
24	0.0029529	0.00100283	464.251	100.65	0.35378	284.96	100.65	2544.7	2545.3
25	0.0031239	0.00100314	480.289	104.83	0.36808	287.74	104.83	2546.5	2547.1
26	0.0033016	0.00100347	497.277	109.01	0.38238	290.58	109.01	2548.3	2548.9
27	0.0034861	0.00100382	515.256	113.19	0.39668	293.48	113.19	2550.1	2550.7
28	0.0036775	0.00100419	534.277	117.37	0.41098	296.44	117.37	2551.9	2552.5
29	0.0038759	0.00100458	554.391	121.55	0.42528	299.46	121.55	2553.7	2554.3
30	0.0040814	0.00100500	575.657	125.73	0.43958	302.54	125.73	2555.5	2556.1
31	0.0042941	0.00100544	598.127	129.91	0.45388	305.68	129.91	2557.3	2557.9
32	0.0045141	0.00100591	621.851	134.09	0.46818	308.88	134.09	2559.2	2559.8
33	0.0047414	0.00100641	646.881	138.26	0.48248	312.14	138.26	2561.0	2561.6
34	0.0049761	0.00100693	673.277	142.43	0.49678	315.46	142.43	2562.8	2563.4
35	0.0052183	0.00100748	701.100	146.60	0.51108	318.84	146.60	2564.6	2565.2
36	0.0054691	0.00100806	730.411	150.77	0.52538	322.28	150.77	2566.4	2567.0
37	0.0057285	0.00100867	761.277	154.94	0.53968	325.78	154.94	2568.2	2568.8
38	0.0060066	0.00100931	793.756	159.11	0.55398	329.34	159.11	2570.0	2570.6
39	0.0062944	0.00101000	827.907	163.28	0.56828	332.96	163.28	2571.8	2572.4
40	0.0065920	0.00101071	863.701	167.45	0.58258	336.64	167.45	2573.6	2574.2

Saturated Water and Steam (Temperature-based), Contd.

T °C	P _{sat} MPa	Volume, m ³ /kg v _f	Energy, kJ/kg u _f	Enthalpy, kJ/kg h _f	Entropy, kJ/(kg K) s _f	Volume, m ³ /kg v _g	Energy, kJ/kg u _g	Enthalpy, kJ/kg h _g	Entropy, kJ/(kg K) s _g
41	0.0069005	0.00101144	901.200	171.64	0.59688	340.40	171.64	2575.4	2576.0
42	0.0072149	0.00101219	940.351	175.81	0.61098	343.98	175.81	2577.2	2577.8
43	0.0075459	0.00101297	981.700	180.00	0.62508	347.62	180.00	2579.0	2579.6
44	0.0078936	0.00101378	1025.300	184.21	0.63918	351.32	184.21	2580.8	2581.4
45	0.0082581	0.00101461	1071.200	188.44	0.65328	355.08	188.44	2582.6	2583.2
46	0.0086396	0.00101546	1119.450	192.69	0.66738	358.90	192.69	2584.4	2585.0
47	0.0090383	0.00101633	1170.100	196.96	0.68148	362.78	196.96	2586.2	2587.0
48	0.0094544	0.00101722	1223.200	201.25	0.69558	366.72	201.25	2588.0	2589.0
49	0.0098881	0.00101813	1278.800	205.56	0.70968	370.72	205.56	2590.0	2591.0
50	0.0103396	0.00101906	1336.950	209.89	0.72378	374.78	209.89	2592.0	2593.0
51	0.0108091	0.00102001	1397.700	214.24	0.73788	378.90	214.24	2594.0	2595.0
52	0.0112968	0.00102100	1461.100	218.61	0.75198	383.08	218.61	2596.0	2597.0
53	0.0118029	0.00102201	1527.200	223.00	0.76608	387.32	223.00	2598.0	2600.0
54	0.0123276	0.00102304	1596.050	227.41	0.78018	391.62	227.41	2600.0	2603.0
55	0.0128711	0.00102409	1667.700	231.84	0.79428	395.98	231.84	2602.0	2606.0
56	0.0134336	0.00102516	1742.200	236.29	0.80838	400.40	236.29	2604.0	2609.0
57	0.0140163	0.00102624	1819.600	240.76	0.82248	404.88	240.76	2606.0	2612.0
58	0.0146194	0.00102733	1900.000	245.25	0.83658	409.42	245.25	2608.0	2615.0
59	0.0152441	0.00102844	1983.450	249.76	0.85068	414.02	249.76	2610.0	2618.0
60	0.0158906	0.00102956	2070.000	254.29	0.86478	418.68	254.29	2612.0	2621.0
61	0.0165591	0.00103070	2160.700	258.84	0.87888	423.40	258.84	2614.0	2624.0
62	0.0172506	0.00103186	2255.600	263.41	0.89298	428.18	263.41	2616.0	2627.0
63	0.0179651	0.00103303	2354.800	268.00	0.90708	433.02	268.00	2618.0	2630.0
64	0.0187036	0.00103421	2458.350	272.61	0.92118	437.92	272.61	2620.0	2633.0
65	0.0194661	0.00103541	2566.300	277.24	0.93528	442.88	277.24	2622.0	2636.0
66	0.0202526	0.00103662	2678.700	281.89	0.94938	447.90	281.89	2624.0	2639.0
67	0.0210641	0.00103784	2795.600	286.56	0.96348	452.98	286.56	2626.0	2642.0
68	0.0219006	0.00103907	2917.100	291.25	0.97758	458.12	291.25	2628.0	2645.0
69	0.0227631	0.00104031	3043.300	295.96	0.99168	463.32	295.96	2630.0	2648.0
70	0.0236516	0.00104156	3174.300	300.69	1.00578	468.58	300.69	2632.0	2651.0
71	0.0245661	0.00104281	3310.100	305.44	1.01988	473.90	305.44	2634.0	2654.0
72	0.0255076	0.00104407	3450.700	310.21	1.03398	479.28	310.21	2636.0	2657.0
73	0.0264761	0.00104534	3596.200	315.00	1.04808	484.72	315.00	2638.0	2660.0
74	0.0274716	0.00104662	3746.700	319.81	1.06218	490.22	319.81	2640.0	2663.0
75	0.0284941	0.00104791	3902.300	324.64	1.07628	495.78	324.64	2642.0	2666.0
76	0.0295436	0.00104921	4063.100	329.49	1.09038	501.40	329.49	2644.0	2669.0
77	0.0306191	0.00105052	4229.200	334.36	1.10448	507.08	334.36	2646.0	2672.0
78	0.0317206	0.00105184	4400.700	339.25	1.11858	512.82	339.25	2648.0	2675.0
79	0.0328481	0.00105317	4577.800	344.16	1.13268	518.62	344.16	2650.0	2678.0
80	0.0339926	0.00105451	4760.500	349.09	1.14678	524.48	349.09	2652.0	2681.0

Saturated Water and Steam (Temperature-based), Contd.

Table with 15 columns: T (C), P_sat (MPa), v_f (m^3/kg), v_g (m^3/kg), u_f (kJ/kg), u_g (kJ/kg), h_f (kJ/kg), h_g (kJ/kg), s_f (kJ/kg K), s_g (kJ/kg K). Rows 80-110.

Saturated Water and Steam (Temperature-based), Contd.

Table with 15 columns: T (C), P_sat (MPa), v_f (m^3/kg), v_g (m^3/kg), u_f (kJ/kg), u_g (kJ/kg), h_f (kJ/kg), h_g (kJ/kg), s_f (kJ/kg K), s_g (kJ/kg K). Rows 111-320.

Saturated Water and Steam (Pressure-based), Contd.

Table with 12 columns: P (MPa), T_sat (C), v_f, v_g, u_f, u_g, h_f, h_g, s_f, s_g, Entropy, kJ/(kg K), Enthalpy, kJ/kg, Energy, kJ/kg, Volume, m³/kg. Rows 0.050 to 0.20.

Saturated Water and Steam (Pressure-based), Contd.

Table with 12 columns: P (MPa), T_sat (C), v_f, v_g, u_f, u_g, h_f, h_g, s_f, s_g, Entropy, kJ/(kg K), Enthalpy, kJ/kg, Energy, kJ/kg, Volume, m³/kg. Rows 0.10 to 1.50.

Saturated Water and Steam (Pressure-based), Contd.

p MPa	T _{sat} °C	Volume, m ³ /kg		Energy, kJ/kg		Enthalpy, kJ/kg		Entropy, kJ/kg K	
		v _f	v _g	u _f	u _g	h _f	h _g	s _f	s _g
1.50	198.287	0.00115387	0.13171	812.83	2593.1	811.56	2791.0	19.161	2.3113
1.55	198.818	0.00115808	0.12760	819.80	2593.1	851.59	2791.9	19.403	2.3291
1.60	201.370	0.00115808	0.12374	856.61	2591.8	858.46	2792.8	19.414	2.3435
1.65	202.856	0.00116103	0.12010	863.25	2595.5	865.17	2793.7	19.255	2.3575
1.70	204.307	0.00116336	0.11667	869.76	2596.2	871.74	2794.5	19.225	2.3711
1.75	205.725	0.00116565	0.11343	876.13	2596.7	878.17	2795.2	19.170	2.3845
1.80	207.112	0.00116792	0.11037	882.37	2597.2	884.47	2795.9	19.114	2.3975
1.85	208.469	0.00117016	0.10746	888.49	2597.8	890.65	2796.6	19.059	2.4102
1.90	209.798	0.00117238	0.10470	894.48	2598.3	896.73	2797.2	19.005	2.4227
1.95	211.101	0.00117458	0.10208	900.37	2598.7	902.66	2797.8	18.951	2.4348
2.0	212.377	0.00117675	0.099585	906.15	2599.1	908.50	2798.3	18.898	2.4468
2.1	213.631	0.00117890	0.097193	911.83	2599.5	914.25	2798.8	18.845	2.4586
2.2	214.865	0.00118103	0.094938	917.39	2599.9	919.91	2799.3	18.791	2.4699
2.3	216.079	0.00118314	0.092815	922.84	2600.3	925.48	2799.8	18.737	2.4807
2.4	217.273	0.00118523	0.090815	928.19	2600.7	930.97	2800.1	18.683	2.4911
2.5	218.447	0.00118730	0.088931	933.44	2601.1	936.38	2800.4	18.629	2.5011
2.6	219.599	0.00118936	0.087163	938.59	2601.4	941.73	2800.8	18.575	2.5107
2.7	220.731	0.00119138	0.085511	943.64	2601.7	947.03	2801.1	18.521	2.5200
2.8	221.843	0.00119338	0.083974	948.59	2602.0	952.28	2801.4	18.467	2.5289
2.9	222.935	0.00119533	0.082551	953.44	2602.3	957.48	2801.7	18.413	2.5375
3.0	224.007	0.00119723	0.081241	958.19	2602.6	962.63	2802.0	18.359	2.5458
3.1	225.059	0.00119908	0.080043	962.84	2602.9	967.73	2802.3	18.305	2.5538
3.2	226.091	0.00120088	0.078956	967.39	2603.2	972.78	2802.6	18.251	2.5615
3.3	227.103	0.00120263	0.077979	971.84	2603.5	977.78	2802.9	18.197	2.5689
3.4	228.095	0.00120433	0.077111	976.19	2603.8	982.73	2803.2	18.143	2.5760
3.5	229.067	0.00120598	0.076351	980.44	2604.1	987.63	2803.5	18.089	2.5828
3.6	230.019	0.00120758	0.075698	984.59	2604.4	992.48	2803.8	18.035	2.5893
3.7	230.951	0.00120913	0.075151	988.64	2604.7	997.28	2804.1	17.981	2.5955
3.8	231.863	0.00121063	0.074609	992.59	2605.0	1002.03	2804.4	17.927	2.6015
3.9	232.755	0.00121208	0.074171	996.44	2605.3	1006.73	2804.7	17.873	2.6072
4.0	233.627	0.00121348	0.073837	1000.19	2605.6	1011.38	2805.0	17.819	2.6127
4.1	234.479	0.00121483	0.073507	1003.84	2605.9	1015.98	2805.3	17.765	2.6180
4.2	235.311	0.00121613	0.073281	1007.39	2606.2	1020.53	2805.6	17.711	2.6231
4.3	236.123	0.00121738	0.073059	1010.84	2606.5	1025.03	2805.9	17.657	2.6280
4.4	236.915	0.00121858	0.072841	1014.19	2606.8	1029.48	2806.2	17.603	2.6327
4.5	237.687	0.00121973	0.072627	1017.44	2607.1	1033.88	2806.5	17.549	2.6372
4.6	238.439	0.00122083	0.072417	1020.59	2607.4	1038.23	2806.8	17.495	2.6415
4.7	239.171	0.00122188	0.072211	1023.64	2607.7	1042.53	2807.1	17.441	2.6456
4.8	239.883	0.00122288	0.072009	1026.59	2608.0	1046.78	2807.4	17.387	2.6495
4.9	240.575	0.00122383	0.071811	1029.44	2608.3	1050.98	2807.7	17.333	2.6532
5.0	241.247	0.00122473	0.071617	1032.19	2608.6	1055.13	2808.0	17.279	2.6567

Water/Steam at p = 0.10 MPa (T_{sat} = 99.606°C)

T °C	v m ³ /kg	u kJ/kg	h kJ/kg	s kJ/kg K
270	2.4993	276.15	301.43	8.1091
280	2.5159	2779.8	3034.4	8.1459
290	2.5924	2795.2	3051.1	8.1818
300	2.6388	2810.6	3071.5	8.2172
310	2.6833	2826.2	3091.4	8.2520
320	2.7217	2841.7	3111.9	8.2864
330	2.7782	2857.3	3135.1	8.3202
340	2.8216	2873.0	3155.5	8.3536
350	2.8710	2888.7	3175.8	8.3866
360	2.9173	2904.4	3196.3	8.4191
370	2.9637	2920.3	3216.7	8.4512
380	3.0100	2936.3	3237.3	8.4829
390	3.0564	2952.3	3257.9	8.5142
400	3.1027	2968.3	3278.6	8.5452
410	3.1490	2984.4	3299.3	8.5757
420	3.1953	3000.6	3320.1	8.6059
430	3.2416	3016.7	3340.9	8.6358
440	3.2879	3033.1	3361.9	8.6653
450	3.3342	3049.4	3382.8	8.6946
460	3.3805	3065.8	3403.9	8.7235
470	3.4267	3082.3	3425.0	8.7521
480	3.4730	3098.9	3446.2	8.7804
490	3.5193	3115.5	3467.4	8.8084
500	3.5655	3132.1	3488.7	8.8361
520	3.6580	3165.8	3531.6	8.8908
540	3.7505	3199.6	3574.7	8.9415
560	3.8430	3233.7	3618.0	8.9972
580	3.9354	3268.2	3661.7	9.0489
600	4.0279	3302.8	3705.6	9.0998
620	4.1203	3337.8	3749.8	9.1499
640	4.2127	3373.0	3794.3	9.1991
660	4.3052	3408.5	3839.0	9.2476
680	4.3976	3444.2	3884.0	9.2954
700	4.4900	3480.1	3929.4	9.3424
720	4.5824	3516.8	3975.0	9.3888
740	4.6747	3553.3	4020.9	9.4345
760	4.7671	3590.3	4067.0	9.4797
780	4.8595	3627.6	4113.5	9.5242
800	4.9519	3665.0	4160.2	9.5681
820	5.0443	3702.8	4207.2	9.6114
840	5.1366	3740.8	4254.5	9.6544
860	5.2290	3779.2	4302.1	9.6968
880	5.3213	3817.8	4349.9	9.7386
900	5.4137	3856.6	4398.0	9.7800
920	5.5061	3895.8	4446.4	9.8209
940	5.5984	3935.2	4495.0	9.8613
960	5.6908	3974.8	4543.9	9.9013
980	5.7831	4014.8	4593.1	9.9408
1000	5.8754	4055.1	4642.6	9.9800

Water/Steam at $p = 0.30 \text{ MPa}$ ($T_{\text{sat}} = 133.522^\circ\text{C}$)

T $^\circ\text{C}$	v m^3/kg	u kJ/kg	h kJ/kg	s $\text{kJ}/\text{kg K}$	T $^\circ\text{C}$	v m^3/kg	u kJ/kg	h kJ/kg	s $\text{kJ}/\text{kg K}$
0	0.001000066	-0.01	0.26	-0.00013	270	0.82810	2700.1	3008.5	7.5913
5	0.000999991	21.02	21.32	0.07625	280	0.81888	2775.6	3028.8	7.6314
10	0.001000020	42.01	42.31	0.15106	290	0.80962	2851.2	3049.2	7.6678
15	0.001000081	62.97	63.27	0.22412	300	0.80034	2926.7	3069.6	7.7037
20	0.001001070	83.89	84.19	0.29642	310	0.79104	2992.2	3090.0	7.7398
25	0.00100287	104.80	105.10	0.36715	320	0.78172	3057.7	3110.4	7.7758
30	0.00100428	125.70	126.00	0.43666	330	0.77237	3123.2	3130.8	7.8080
35	0.00100591	146.60	146.90	0.50503	340	0.76301	3188.7	3151.4	7.8417
40	0.00100776	167.49	167.79	0.57229	350	0.75363	3254.2	3172.0	7.8750
45	0.00100988	188.39	188.69	0.63849	360	0.74424	3319.7	3192.6	7.9078
50	0.00101202	209.29	209.59	0.70368	370	0.73483	3385.2	3213.2	7.9401
55	0.0010143	230.20	230.50	0.76788	380	0.72541	3450.7	3233.8	7.9721
60	0.00101700	251.11	251.42	0.83111	390	0.71600	3516.2	3254.4	8.0036
65	0.00102017	272.03	272.34	0.89350	400	0.70658	3581.7	3275.0	8.0347
70	0.00102385	292.98	293.29	0.95497	410	0.69716	3647.2	3295.6	8.0655
75	0.00102771	313.93	314.24	1.0156	420	0.68774	3712.7	3316.2	8.0959
80	0.00103183	334.90	335.21	1.0754	430	0.67832	3778.2	3336.8	8.1259
85	0.00103621	355.89	356.20	1.1344	440	0.66890	3843.7	3357.4	8.1556
90	0.00104084	376.91	377.22	1.1927	450	0.65948	3909.2	3378.0	8.1849
95	0.00104572	397.94	398.25	1.2502	460	0.65006	3974.7	3398.6	8.2140
100	0.00105086	418.97	419.28	1.3071	470	0.64064	4040.2	3419.2	8.2427
105	0.00105625	439.99	440.30	1.3632	480	0.63122	4105.7	3439.8	8.2711
110	0.00106189	461.01	461.32	1.4187	490	0.62180	4171.2	3460.4	8.2992
115	0.00106768	482.03	482.34	1.4736	500	0.61238	4236.7	3481.0	8.3271
120	0.00107362	503.06	503.37	1.5278	510	0.60296	4302.2	3501.6	8.3547
125	0.00107971	524.08	524.39	1.5815	520	0.59354	4367.7	3522.2	8.3821
130	0.00108595	545.11	545.42	1.6346	530	0.58412	4433.2	3542.8	8.4092
133.522	0.00107317	561.11	561.43	1.6717	540	0.57470	4498.7	3563.4	8.4359
135	0.00107322	605.70	606.02	1.6916	550	0.56528	4564.2	3584.0	8.4622
140	0.00106833	2545.7	2728.2	6.9998	600	0.43114	3301.6	3704.0	8.5914
145	0.00106343	2545.3	2739.4	7.0269	620	0.3723	3336.6	3748.3	8.6416
150	0.00105853	2544.9	2750.6	7.0543	640	0.3135	3371.6	3792.6	8.6909
155	0.00105363	2544.5	2761.8	7.0791	660	0.2547	3406.6	3837.0	8.7395
160	0.00104873	2544.1	2773.0	7.1011	680	0.1959	3441.6	3881.4	8.7873
165	0.00104383	2543.7	2784.2	7.1211	700	0.1371	3476.6	3925.8	8.8344
170	0.00103893	2543.3	2795.4	7.1391	720	0.0783	3511.6	3970.2	8.8809
175	0.00103403	2542.9	2806.6	7.1551	740	0.0195	3546.6	4014.6	8.9267
180	0.00102913	2542.5	2817.8	7.1691	760	0.0007	3581.6	4059.0	8.9719
185	0.00102423	2542.1	2829.0	7.1821	780	0.0000	3616.6	4103.4	9.0164
190	0.00101933	2541.7	2840.2	7.1941	800	0.0000	3651.6	4147.8	9.0601
195	0.00101443	2541.3	2851.4	7.2051	820	0.0000	3686.6	4192.2	9.1039
200	0.00100953	2540.9	2862.6	7.2151	840	0.0000	3721.6	4236.6	9.1468
210	0.00100463	2540.5	2884.2	7.2331	860	0.0000	3756.6	4281.0	9.1892
220	0.00100000	2540.1	2905.8	7.2491	880	0.0000	3791.6	4325.4	9.2314
230	0.00099566	2539.7	2927.4	7.2631	900	0.0000	3826.6	4369.8	9.2724
240	0.00099153	2539.3	2949.0	7.2761	920	0.0000	3861.6	4414.2	9.3133
250	0.00098761	2538.9	2970.6	7.2881	940	0.0000	3896.6	4458.6	9.3538
260	0.00098389	2538.5	2992.2	7.2991	960	0.0000	3931.6	4503.0	9.3938
270	0.00098037	2538.1	3013.8	7.3091	980	0.0000	3966.6	4547.4	9.4338
270	0.00098037	2538.1	3013.8	7.3091	1000	0.0000	3966.6	4547.4	9.4338

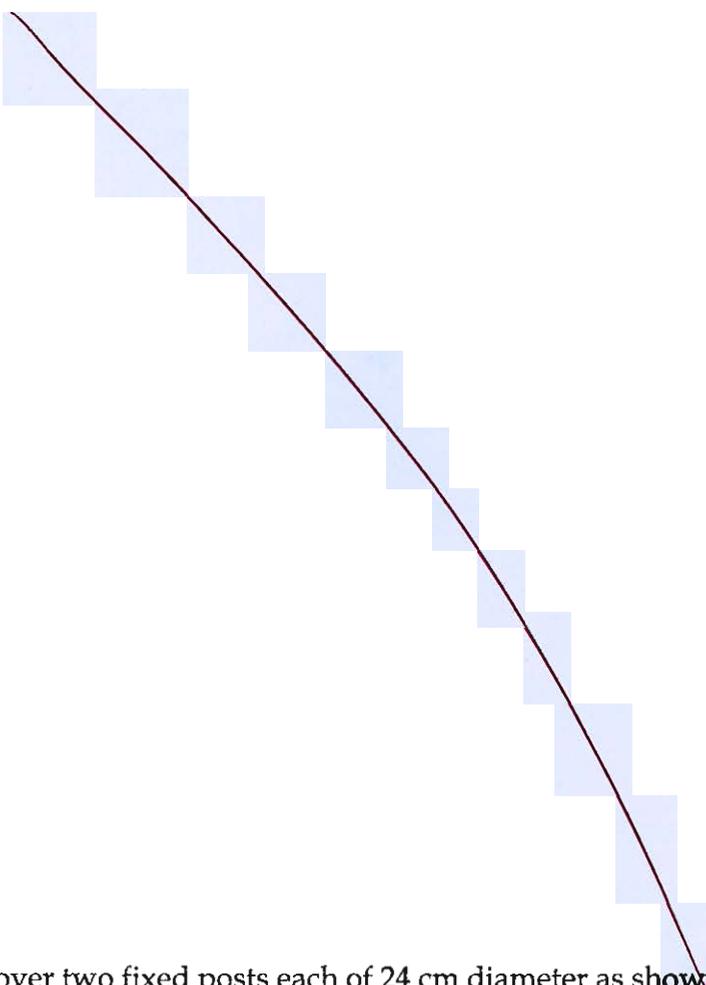
Water/Steam at $p = 4.0 \text{ MPa}$ ($T_{\text{sat}} = 250.354^\circ\text{C}$)

T $^\circ\text{C}$	v m^3/kg	u kJ/kg	h kJ/kg	s $\text{kJ}/\text{kg K}$	T $^\circ\text{C}$	v m^3/kg	u kJ/kg	h kJ/kg	s $\text{kJ}/\text{kg K}$
0	0.00099819	0.03	1.02	0.00009	270	0.0536930	2676.4	2871.2	6.2016
5	0.00099813	31.00	21.99	0.07617	280	0.0554970	2680.9	2902.9	6.2505
10	0.00099811	41.92	45.91	0.15072	290	0.0572170	2704.1	2933.0	6.3133
15	0.00099809	62.80	66.80	0.22385	300	0.0588760	2726.2	2963.1	6.3639
20	0.00100000	83.67	87.67	0.29564	310	0.0604680	2747.5	2989.4	6.4118
25	0.00100120	104.52	108.52	0.36619	320	0.0620210	2768.8	3016.5	6.4576
30	0.00100263	125.36	129.37	0.43553	330	0.0635360	2788.4	3044.2	6.5011
35	0.00100427	146.20	150.22	0.50374	340	0.0650190	2808.0	3072.1	6.5435
40	0.00100612	167.05	171.07	0.57085	350	0.0664730	2827.1	3099.3	6.5843
45	0.00100816	187.89	191.92	0.63691	360	0.0679030	2845.6	3126.8	6.6238
50	0.00101038	208.74	212.78	0.70196	370	0.0693110	2863.4	3154.6	6.6621
55	0.00101277	229.59	233.64	0.76604	380	0.0707010	2881.0	3182.8	6.6994
60	0.00101531	250.46	254.52	0.82918	390	0.0720730	2902.1	3211.5	6.7358
65	0.00101797	271.34	275.41	0.89141	400	0.0734310	2920.8	3241.5	6.7714
70	0.00102075	292.23	296.31	0.95277	410	0.0747760	2939.0	3271.8	6.8061
75	0.00102369	313.13	317.23	1.0133	420	0.0761080	2957.1	3302.5	6.8402
80	0.00102673	334.05	338.16	1.0730	430	0.0774290	2975.1	3333.8	6.8736
85	0.00102986	354.99	359.11	1.1319	440	0.0787410	2993.0	3365.0	6.9064
90	0.00103308	375.94	380.08	1.1900	450	0.0800430	3011.0	3396.2	6.9386
95	0.00103638	396.89	401.08	1.2475	460	0.0813370	3028.9	3427.4	6.9703
100	0.00103974	417.83	422.10	1.3042	470	0.0826230	3046.7	3458.7	7.0015
105	0.00104316	438.77	443.15	1.3602	480	0.0839020	3064.6	3490.2	7.0321
110	0.00104663	459.72	464.22	1.4156	490	0.0851750	3082.4	3521.8	7.0622
115	0.00105015	480.67	485.31	1.4703	500	0.0864420	3100.2	3553.4	7.0916
120	0.00105372	501.62	506.40	1.5245	510	0.0877050	3117.9	3585.1	7.1206
125	0.00105733	522.57	527.48	1.5780	520	0.0889650	3135.6	3616.8	7.1490
130	0.00106100	543.52	548.51	1.6310	530	0.0902220	3153.3	3648.5	7.1771
135	0.00106472	564.47	569.49	1.6835	540	0.0914770	3171.0	3680.2	7.2049
140	0.00106849	585.42	590.46	1.7354	550	0.0927300	3188.7	3711.9	7.2324
145	0.00107231	606.37	611.43	1.7869	560	0.0939810	3206.4	3743.6	7.2597
150	0.00107618	627.32	632.40	1.8379	570	0.0952300	3224.1	3775.3	7.2868
155	0.00108009	648.27	653.37	1.8884	580	0.0964780	3241.8	3807.0	7.3137
160	0.00108404	669.22	674.34	1.9385	590	0.0977240	3259.5	3838.7	7.3404
165	0.00108802	690.17	695.31	1.9882	600	0.0989690	3277.2	3870.4	7.3669
170	0.00109203	711.12	716.28	2.0376	610	0.1002130	3294.9	3902.1	7.3932
175	0.00109607	732.07	737.25	2.0865	620	0.1014560	3312.6	3933.8	7.4193
180	0.00110014	753.02	758.32	2.1352	630	0.1026980	3330.3	3965.5	7.4452
185	0.00110423	773.97	779.39	2.1835	640	0.1039390	3348.0	3997.2	7.4709
190	0.00110834	794.92	800.46	2.2315	650	0.1051790	3365.7	4028.9	7.4964
195	0.00111247	815.87	821.53	2.2792	660	0.1064180	3383.4	4060.6	7.5217
200	0.00111662	836.82	842.60	2.3267	670	0.1076560	3401.1	4092.3	7.5468
210	0.00112079	857.77	863.67	2.3742	680	0.1088930	3418.8	4124.0	7.5717
220	0.00112500	878.72	884.74	2.4215	690	0.1101290	3436.5	4155.7	7.5964
230	0.00112923	899.67	905.81	2.4688	700	0.1113640	3454.2	4187.4	7.6209
240	0.00113350	920.62	926.88	2.5161	710	0.1125980	3471.9	4219.1	7.6452
250	0.00113781	941.57	947.95	2.5634	720	0.1138310	3489.6	4250.8	7.6693
260	0.00114216	962.52	969.02	2.6107	730	0.1150630	3507.3	4282.5	7.6932
270	0.00114654	983.47	990.09	2.6579	740	0.1162940	3525.0	4314.2	7.7169
270	0.00114654	983.47	990.09	2.6579	750	0.1175240	3542.7	4345.9	7.7404

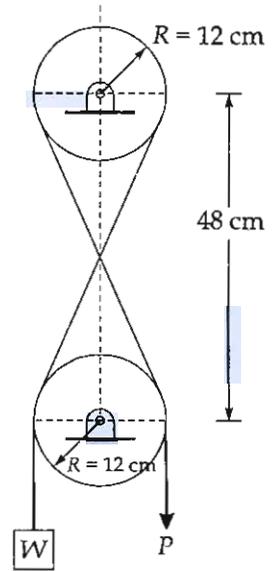
Section : B

- Q.5 (a) A hollow shaft of outside diameter 60 mm and inside diameter 0.4 times the outside diameter, is subjected to a torque of T (Nm) and a bending moment of $0.5T$ (Nm). The tensile yield stress of the shaft material is 260 MPa. For the given application, taking factor of safety to be 2, what will be the maximum permissible value of T to avoid failure according to Tresca's failure theory?

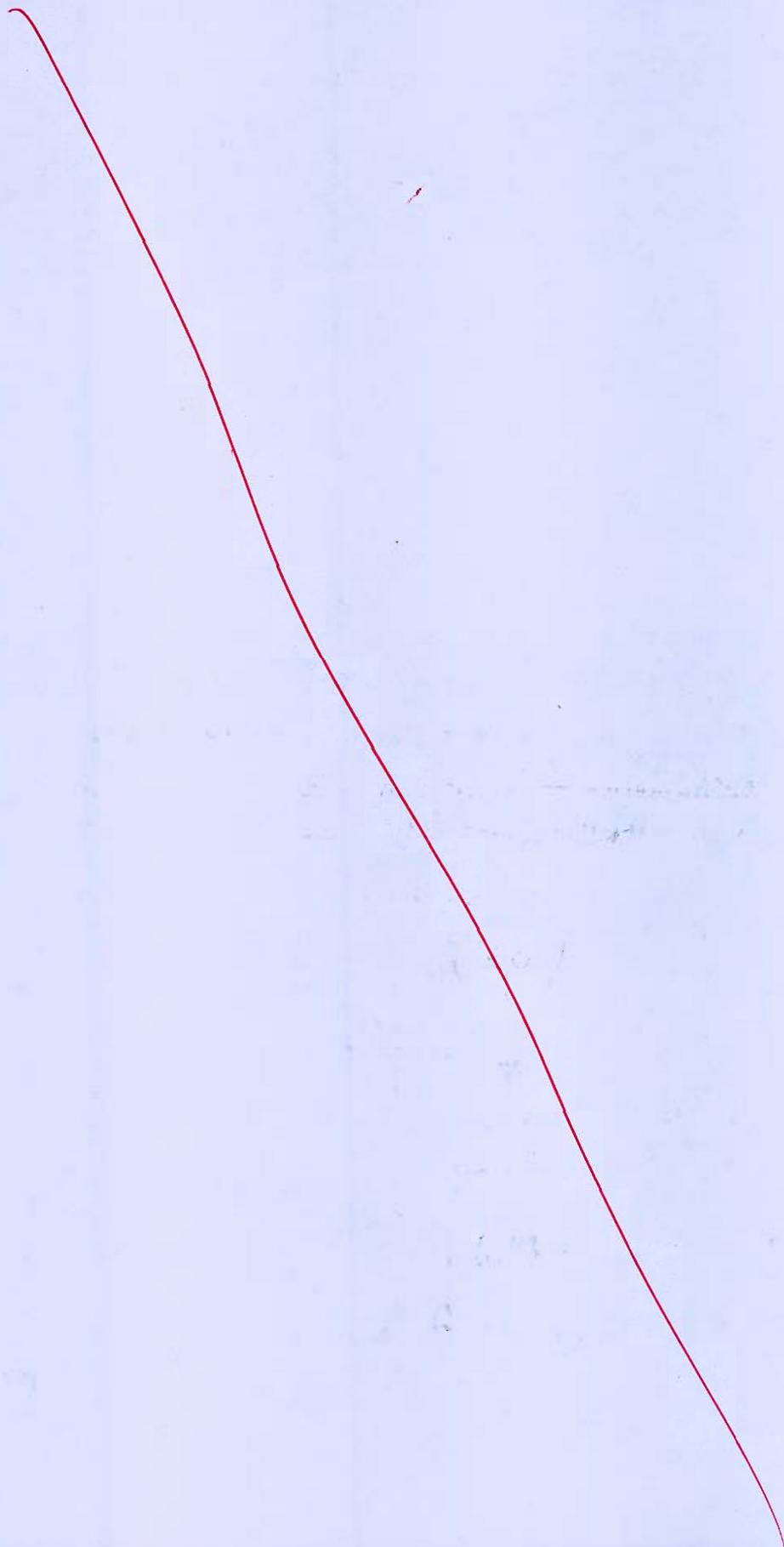
[12 marks]



Q.5 (b) A rope is looped over two fixed posts each of 24 cm diameter as shown in figure below. If the coefficient of friction $\mu = \frac{7}{8\pi}$, then determine the maximum and minimum values of P that will prevent the motion of the load $W = 7$ kN.



[12 marks]



- Q.5 (c) Two pieces of materials A and B have the same bulk modulus, but the Young's modulus of B is 2% greater than that for A. Obtain the value of modulus of rigidity for the material B in terms of Young's modulus and modulus of rigidity for material A.

[12 marks]

for Material A,
Properties $\rightarrow E_A, k_A, G_A$

$$E_B = 1.02 E_A$$

$$k_B = k_A$$

Using

$$E = 3k(1 - 2\nu)$$

$$E_A = 3k_A(1 - 2\nu_A)$$

$$\text{also, } E_B = 3k_B(1 - 2\nu_B)$$

$$\therefore, \quad \cancel{3k_B} (1 - \cancel{2\mu_B}) = 1.02 \cancel{3k_A} (1 - 2\mu_A)$$

$$1 - 2\mu_B = 1.02 (1 - 2\mu_A)$$

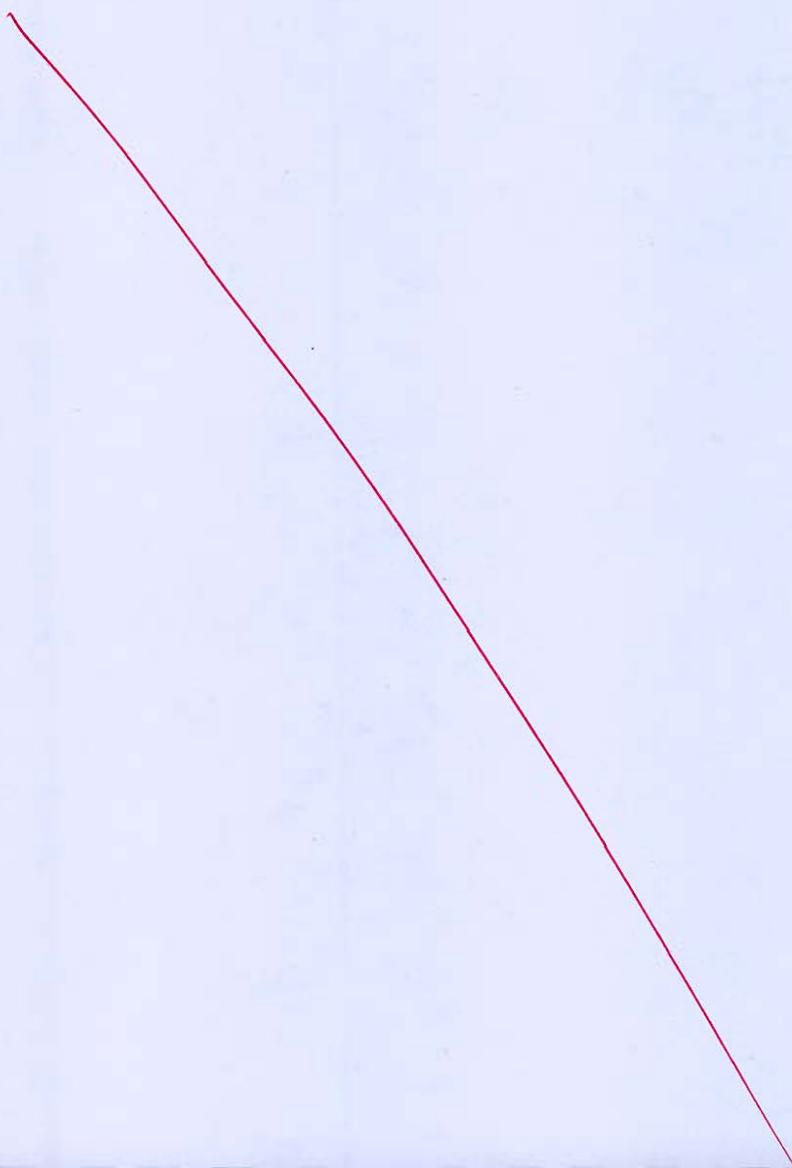
$$1 - 2\mu_B = 1.02 - 2.04\mu_A$$

~~2.04 μ_A~~

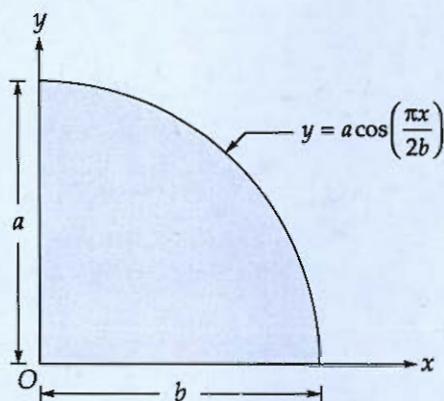
$2.04\mu_A - 2\mu_B = .02$

 ①

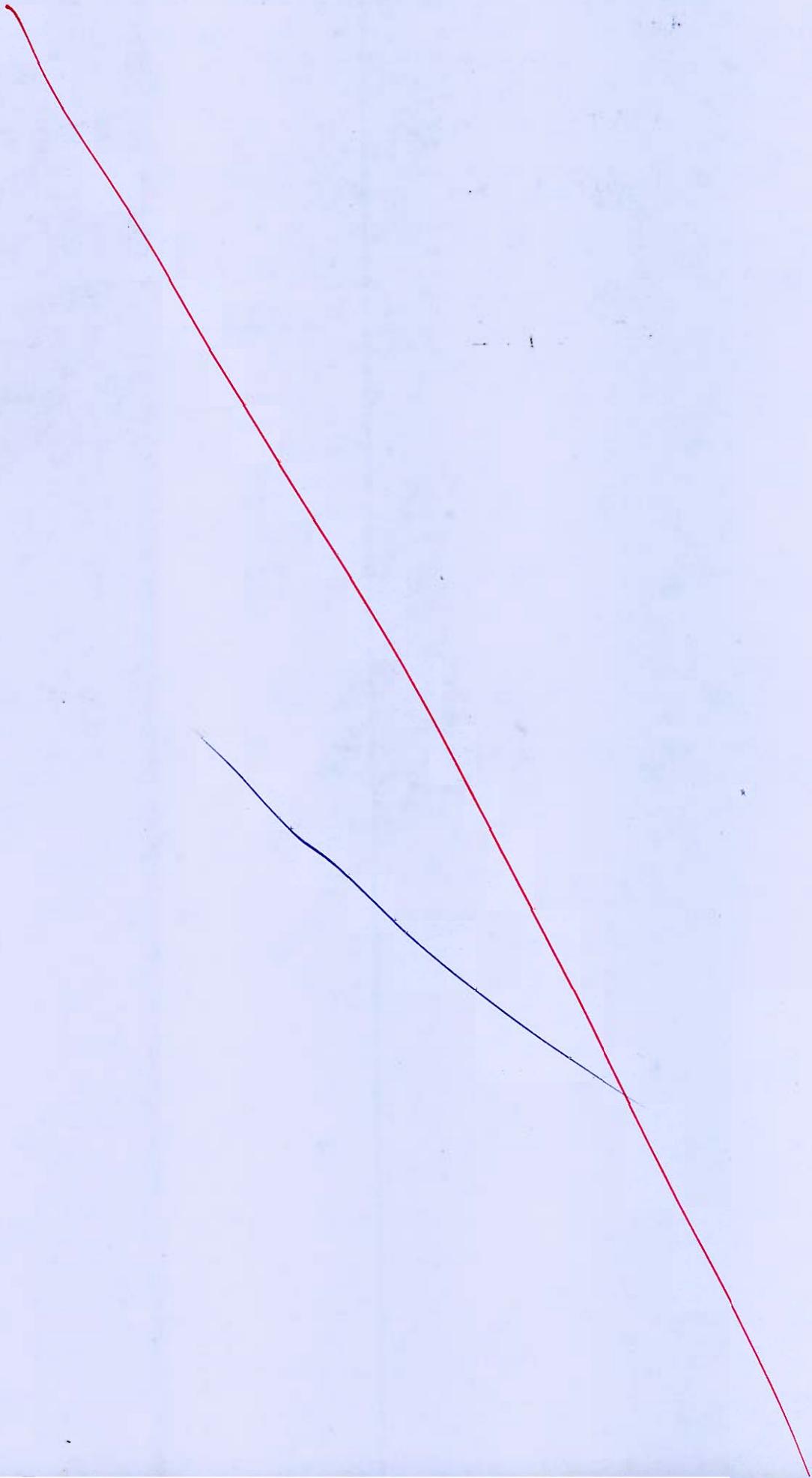
①



- Q.5 (d) Determine the centroid of the area under the curve $y = a \cos\left(\frac{\pi x}{2b}\right)$ between $x = 0$ and $x = b$ as shown in figure below.

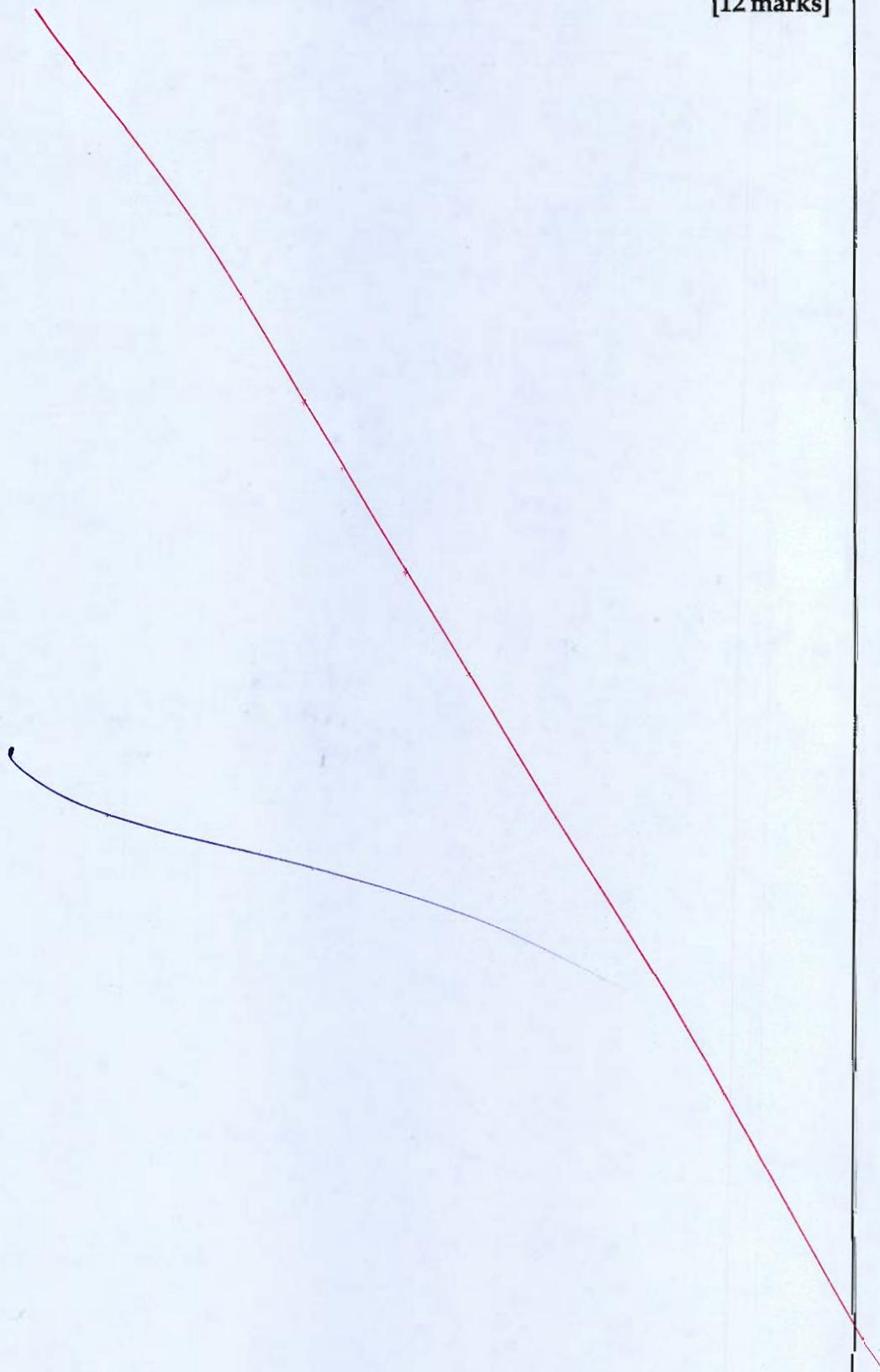


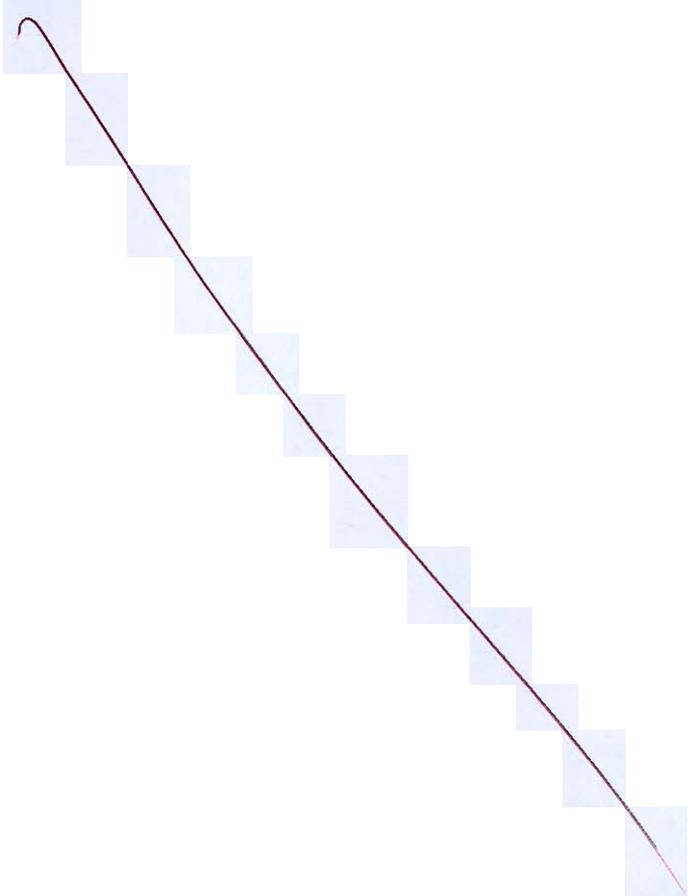
[12 marks]



- Q.5 (e) A copper cylinder 945 mm long, 420 mm internal diameter and 10 mm thickness with flat ends, is initially full of oil at atmospheric pressure. Calculate the volume of oil which must be pumped into the cylinder in order raise the pressure to 6 MPa above atmospheric pressure. For copper, take $E = 100$ GPa and $\mu = 0.3$. Take bulk modulus of oil as 2600 MPa. Neglect the deformation of the end plates.

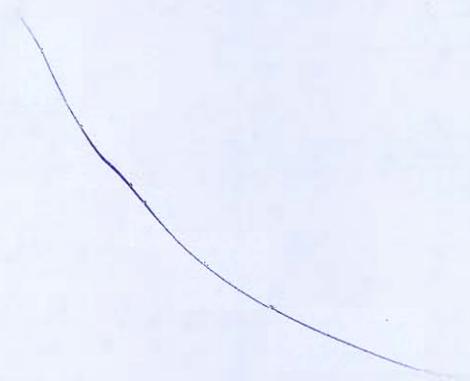
[12 marks]

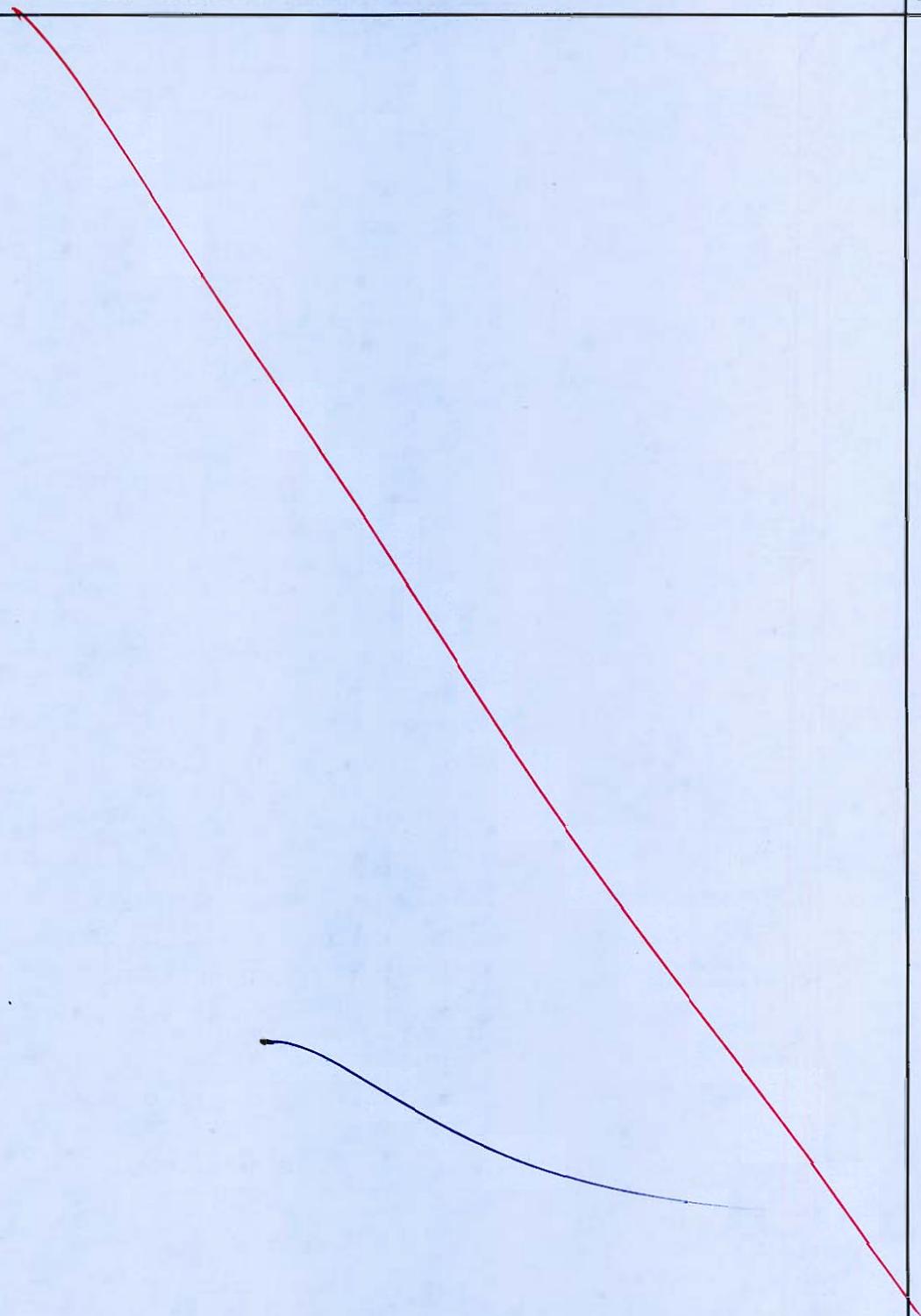


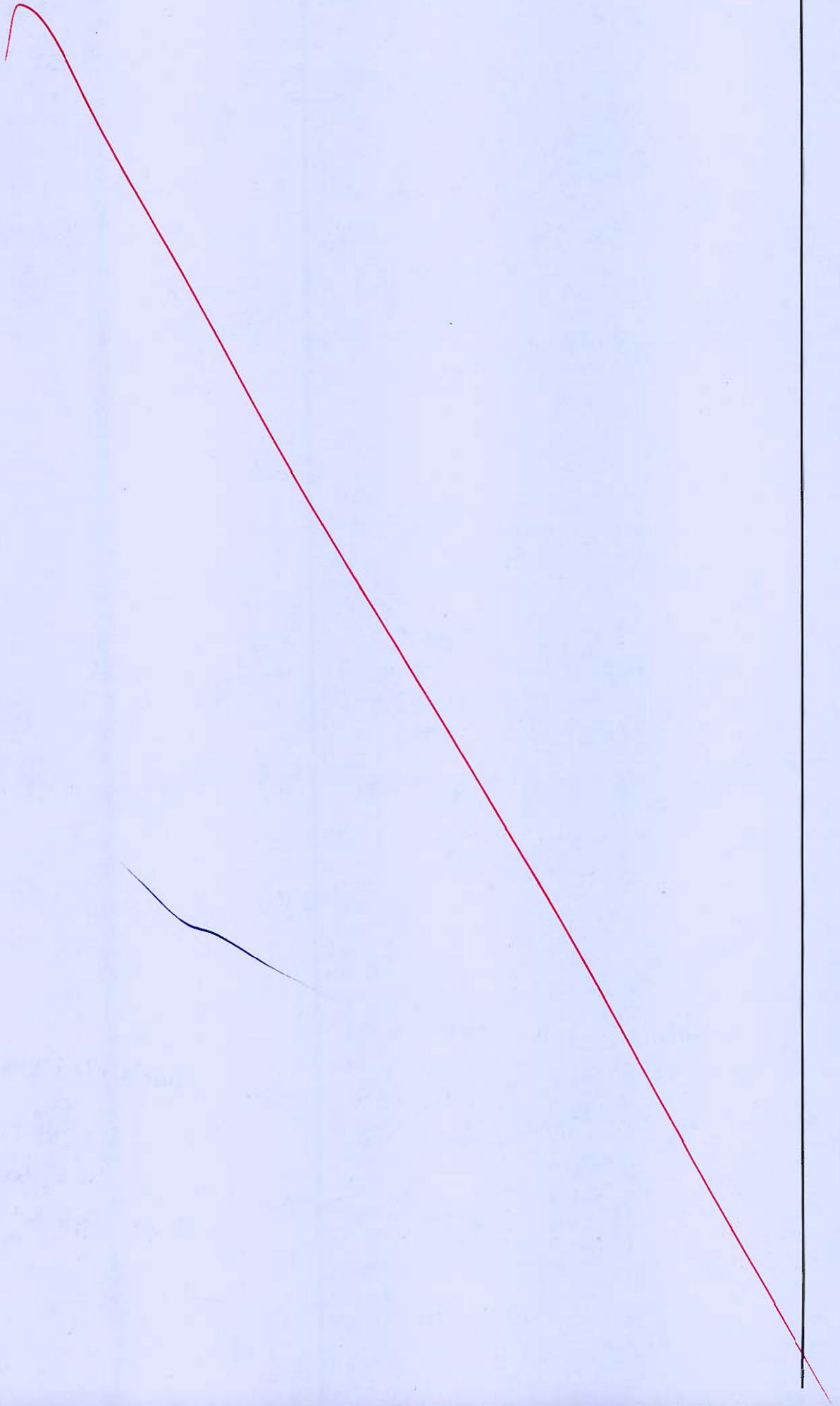


- Q.6 (a) (i) A seamless spherical shell is of 87.5 cm internal diameter and 6 mm thickness. It is filled with fluid under pressure until its volume increases by 64 cm^3 . Calculate the fluid pressure by taking $E = 205 \text{ GPa}$ and Poisson's ratio = 0.32.
- (ii) A body is under action of two principal stresses of 50 MPa and -80 MPa and the third principle stress being zero. The elastic limit in simple tension and compression is 210 MPa. Calculate the factor of safety based on the elastic limit according to
- (1) Maximum shear stress theory
 - (2) Maximum strain energy theory
 - (3) Maximum shear strain energy theory
- Take poisson's ratio as 0.25.

[8 + 12 marks]

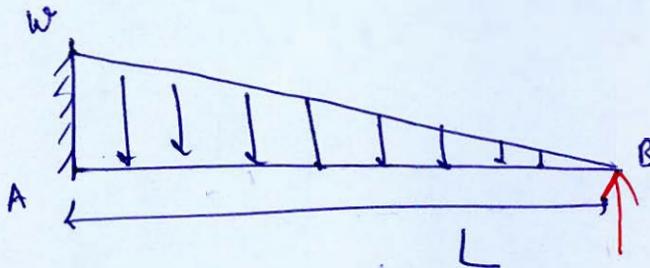






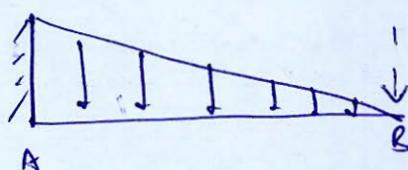
Q.6 (b) A cantilever beam of length 'L' is loaded by a uniformly varying load, the load at free end is zero and maximum load of w at the fixed end. The free end is propped to the level of fixed end. Assuming EI to be constant, calculate the reaction at prop and equation to the elastic curve along with the slope at the propped end.

[20 marks]



Calculation of Prop Reaction :-

① calculating deflection at B when no prop load is used,



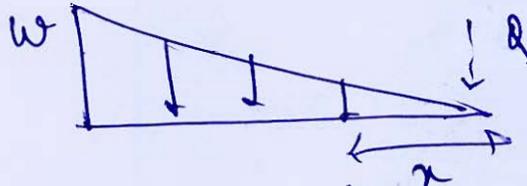
let Q be the dummy load at the free end.

~~Bending Moment at any section x from B $= Q$~~

Using Castiglione's theorem,

$$\delta_B = \frac{1}{EI} \int_0^L M \left| \frac{\partial M}{\partial Q} \right| dx$$

At a section x from B



δ_B at $L \rightarrow$ intensity of loading w .

$$L \rightarrow w$$

$$1 \rightarrow \frac{w}{L} ?$$

$$x \rightarrow \frac{xw}{L}$$

$$\delta_B M_x = -Qx - \left(\frac{1}{2}\right) \left(\frac{xw}{L}\right) (x) \left(\frac{x}{3}\right)$$

$$\frac{\partial M_x}{\partial Q} = -x$$

$$\delta_B \delta_B = \frac{1}{EI} \int_0^L \left(-Qx - \frac{wx^3}{6L} \right) (-x) dx$$

$$\delta_B = \frac{1}{EI} \int_0^L \frac{wx^4}{6L} dx$$

$$\delta_B = \frac{1}{EI} \frac{w}{6L} \frac{x^5}{5} \Big|_0^L$$

$$\delta_B = \frac{wL^4}{30EI}$$

(downward)

~~δ_B~~

When a prop is used, deflection caused by it at free end of cantilever is $\frac{PL^3}{3EI}$ (upward)

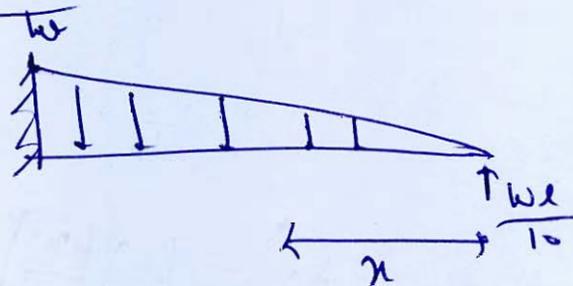


$$\therefore, \frac{PL^3}{3EI} = \frac{wl^4}{30EI} \quad (\text{for no deflection at free end})$$

$$P = \frac{wl}{10}$$

Equation of Elastic Curve

$M_x =$



$$M_x = \frac{wlx}{10} - \frac{wx^3}{6L}$$

$$\therefore, EI \frac{d^2y}{dx^2} = M_x$$

$$EI \frac{d^2y}{dx^2} = \frac{wlx}{10} - \frac{wx^3}{6L}$$

Slope at ~~free~~ Roped End

Integrating Elastic Curve Eqn,

$$EI \left(\frac{dy}{dx} \right) = \frac{wl}{10} \frac{x^2}{2} - \frac{wx^4}{24L} + C_1$$

⑧
Take time to think
before proceeding
for solution.

$$\frac{dy}{dx} = \frac{1}{EI} \left(\frac{wx^2}{20} - \frac{wx^4}{24L} + C_1 \right)$$

$$\text{at } x=L, \frac{dy}{dx} = 0 \Rightarrow 0 = \frac{wL^2}{20} - \frac{wL^3}{24} + C_1$$

$$C_1 = \frac{wL^3}{24} - \frac{wL^2}{20}$$

$$C_1 = \frac{5wL^3 - 6wL^2}{120} \Rightarrow$$

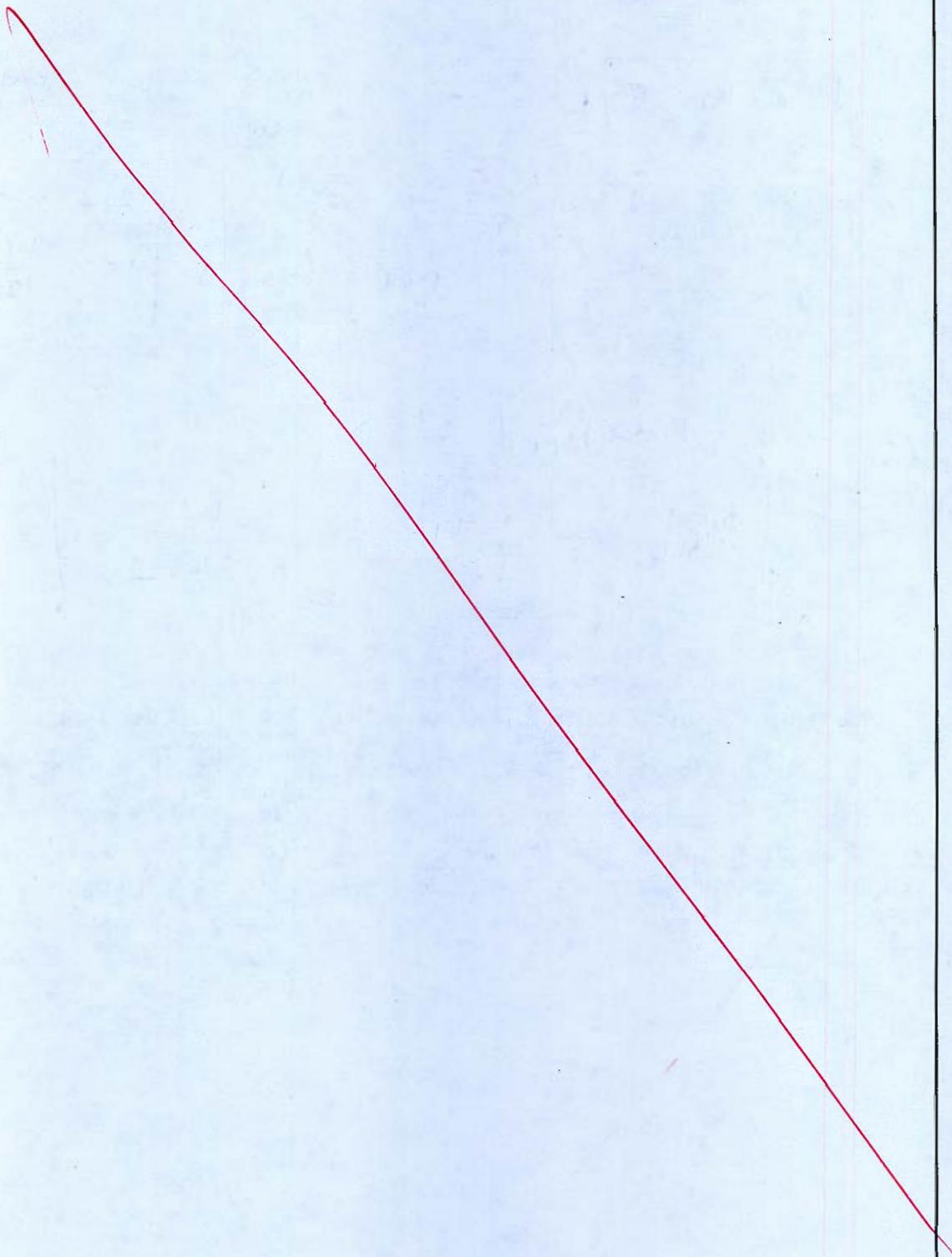
$$C_1 = -\frac{wL^2}{120}$$

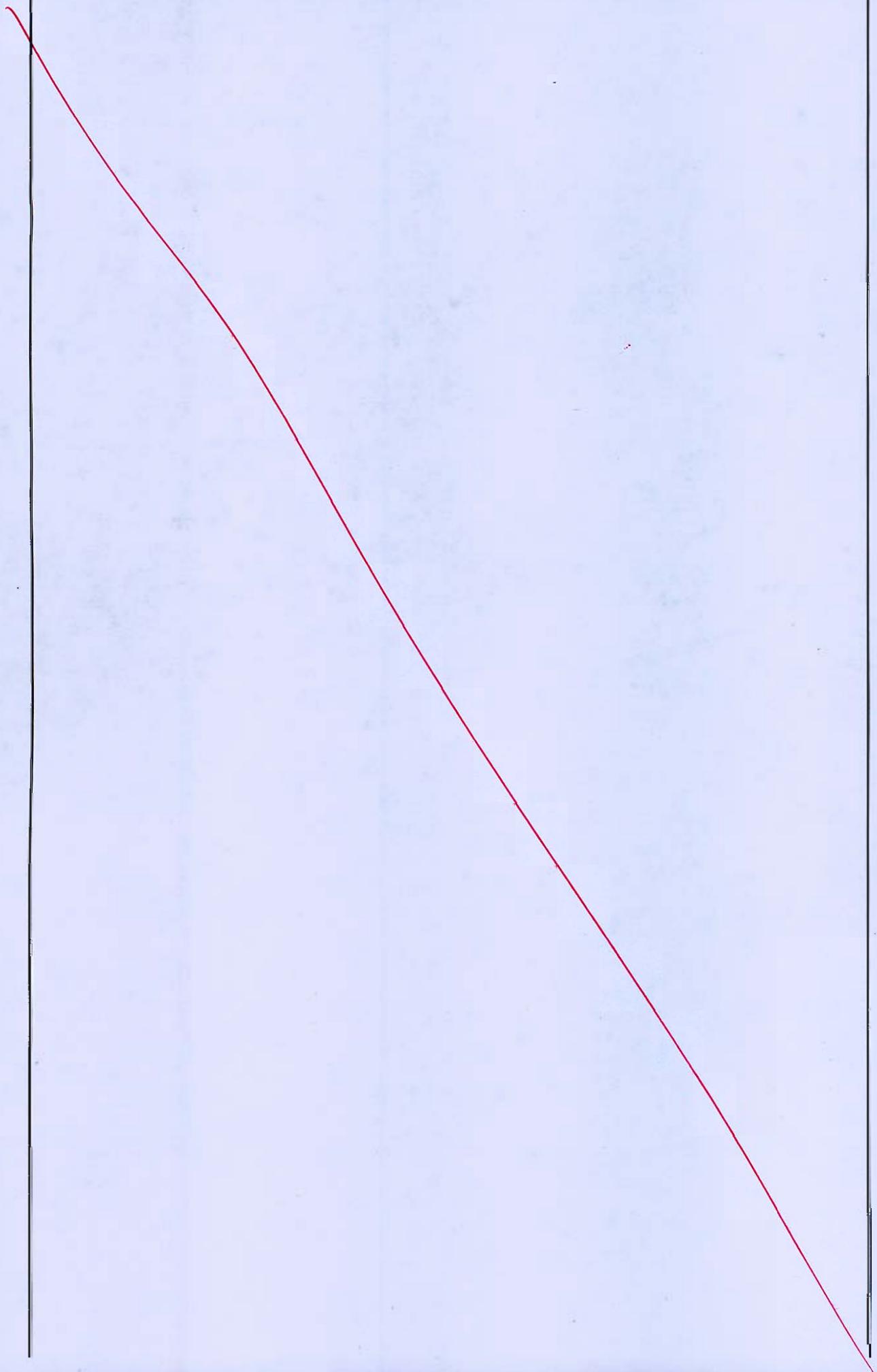
So, slope at free end, $x=0$

$$\left. \frac{dy}{dx} \Big|_{x=0} = \frac{1}{EI} C_1 = -\frac{wL^2}{120EI} \right.$$

- Q.6 (c) Two steel plugs fit freely into the ends of a steel tubular distance piece 420 mm long and drawn together by a steel bolt 520 mm long and nut, the nut being tight fit in the beginning. The nut is further tightened by $\frac{1}{5}$ turn to draw the pieces together, the pitch of the bolt thread being 2.5 mm. The pieces are then subjected to forces of 60 kN tending to pull them apart. The area of cross-section of the bolt is 700 mm² and that of the tube is 500 mm². Young's modulus of steel is 200 GPa. Calculate the stresses in the bolt and the tube.

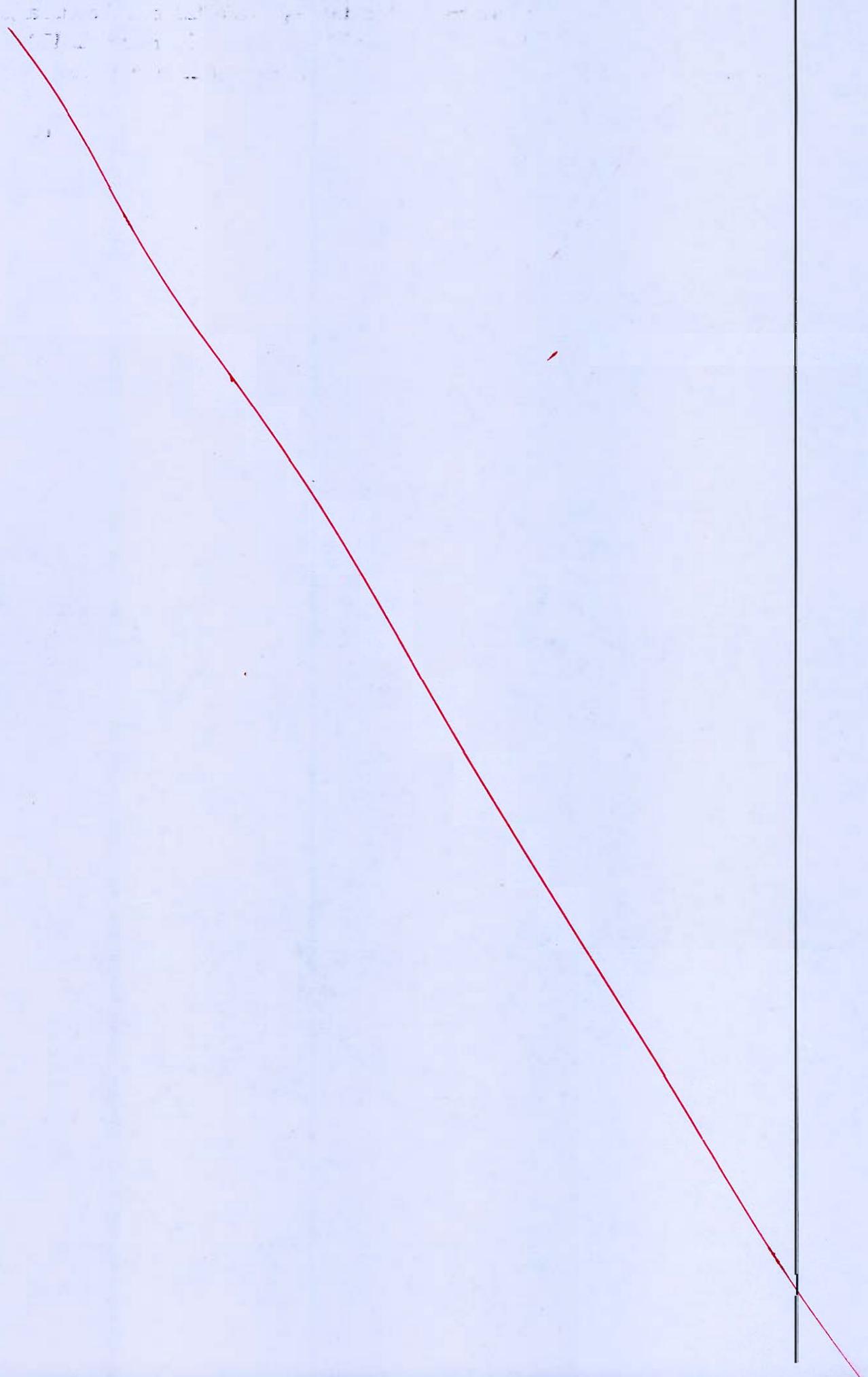
[20 marks]



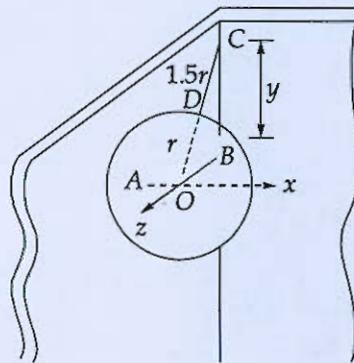


- Q.7 (a) A hollow steel shaft 4.5 m long is to transmit 150 kW of power at 120 rpm. The total angle of twist is not to exceed 1.8° in this length and the allowable shear stress is 45 MPa. Calculate the inside and outside diameters of the shaft. Assume modulus of elasticity as 200 GPa and Poisson's ratio as 0.25.

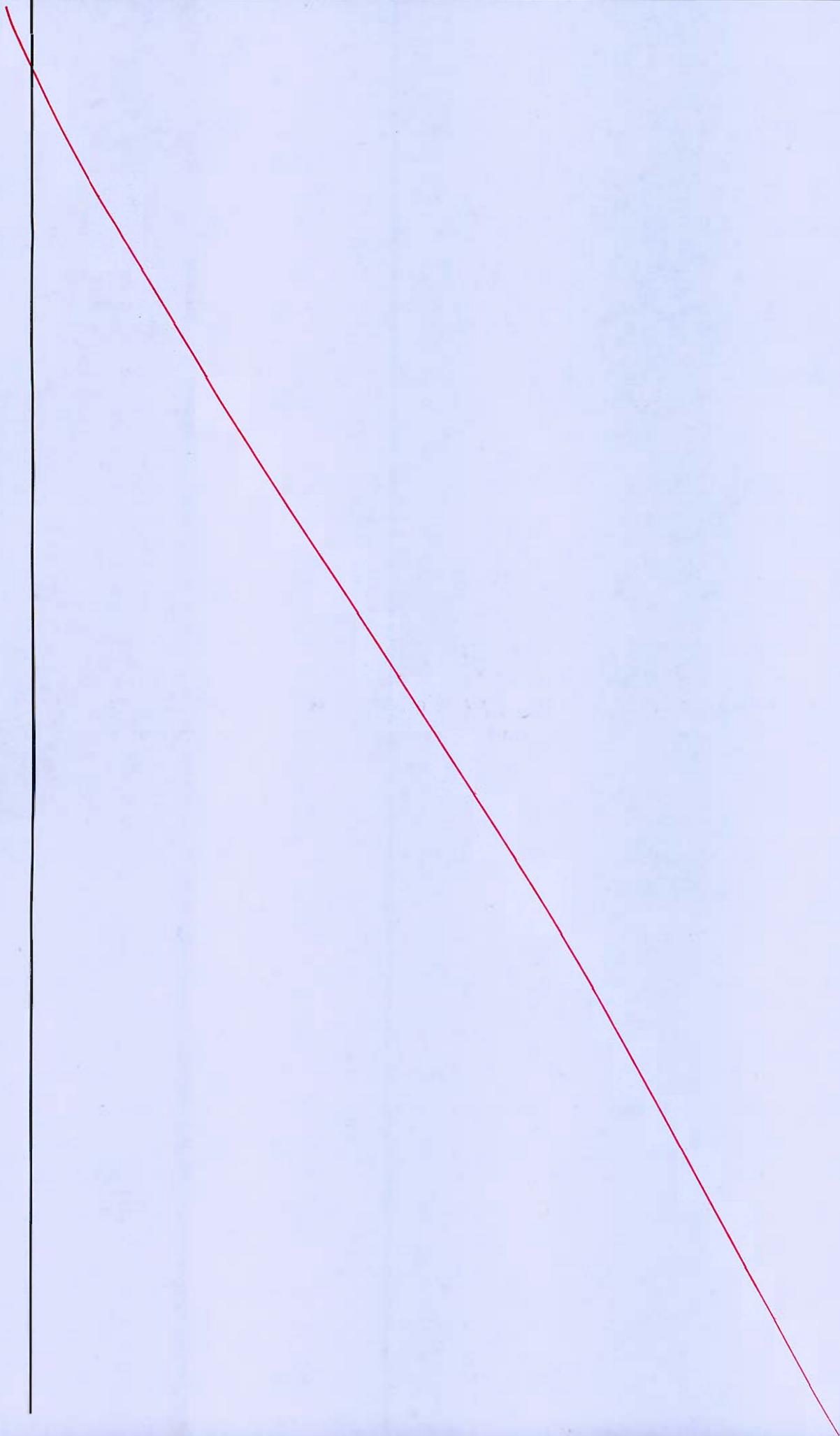
[20 marks]

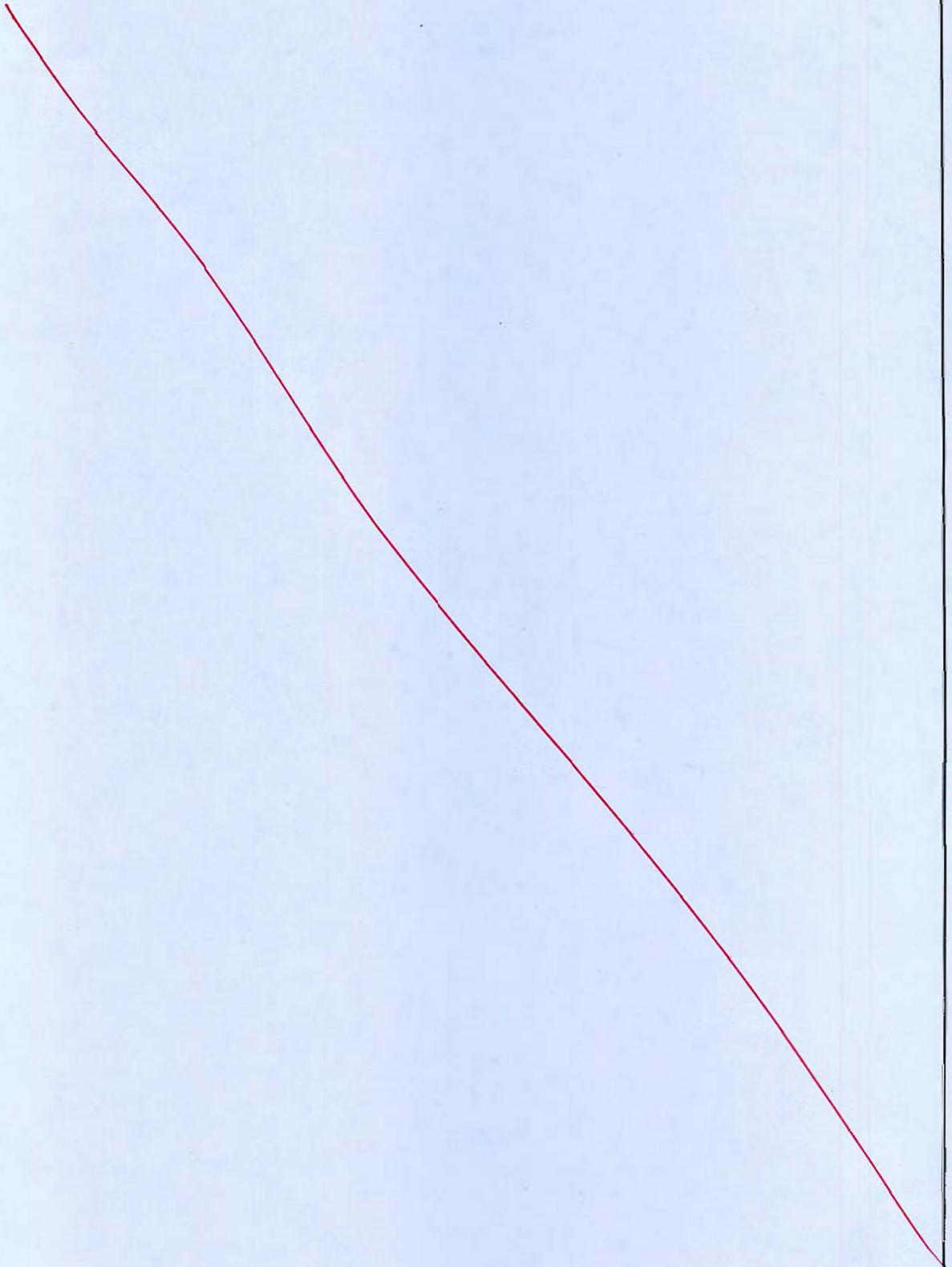


- Q.7 (b) A wire length 30 cm is tied to point D on the surface of a sphere of radius 20 cm. Then the wire is tied to point C which is the intersection of two smooth walls, the walls being at right angles to each other as shown in figure below. If the weight of the sphere is 500 N, then determine the tension in the wire and the reactions from the wall. Assume all contact surface are smooth.



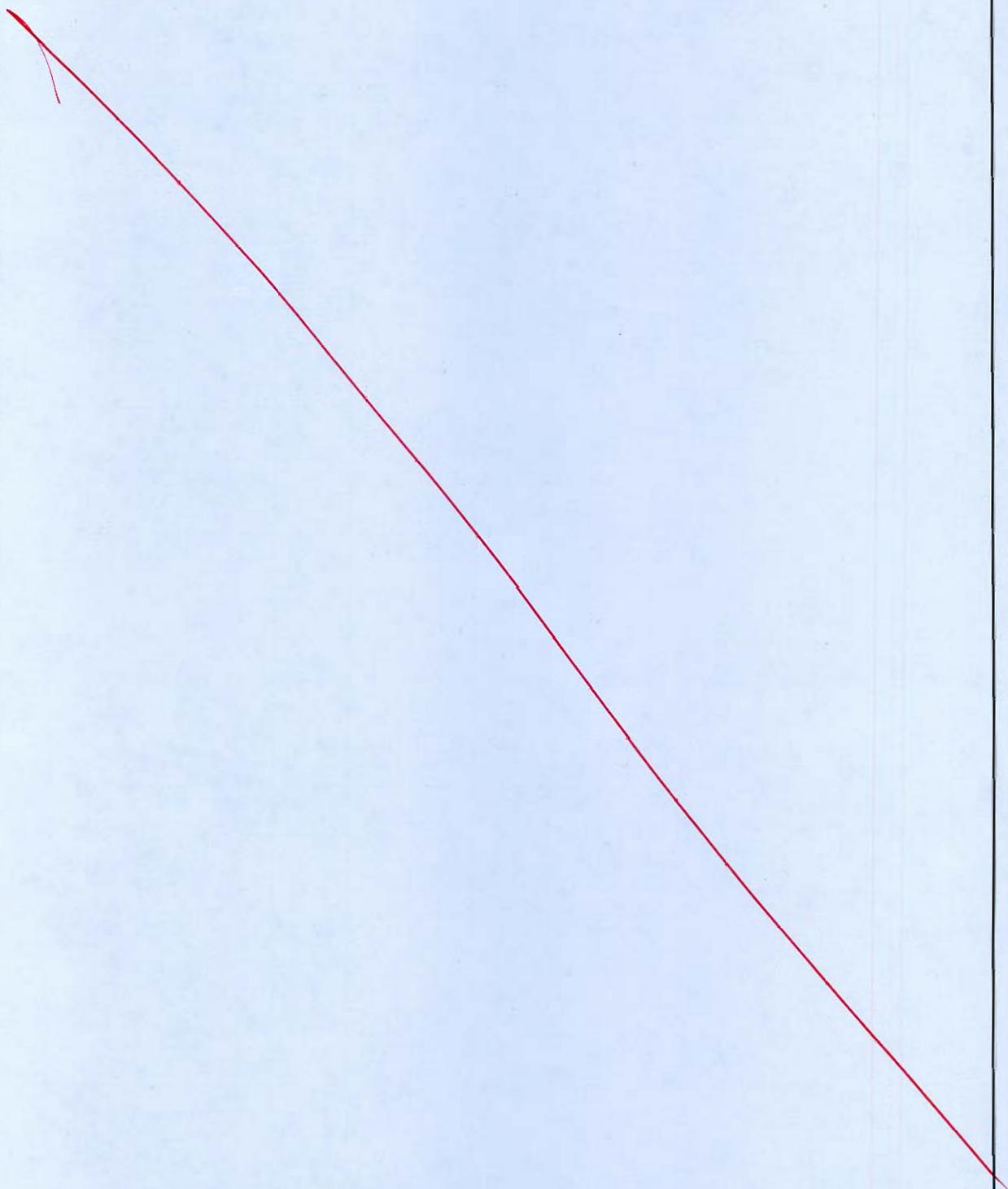
[20 marks]





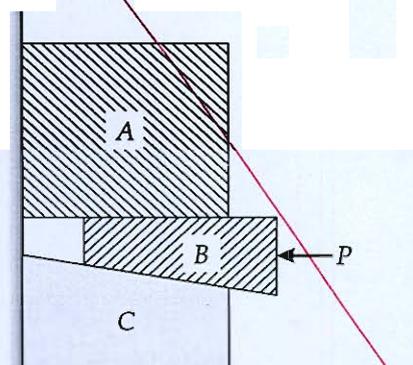
- 2.7 (c) The compression flange of a cast iron girder is 140 mm and 20 mm deep, the tension flange 260 mm wide and 40 mm and the web 300 mm \times 20 mm. Calculate the load per metre run which may be carried over a 5 m span by a beam simply supported at the ends, if maximum permissible stresses are 95 MPa in compression and 35 MPa in tension.

[20 marks]

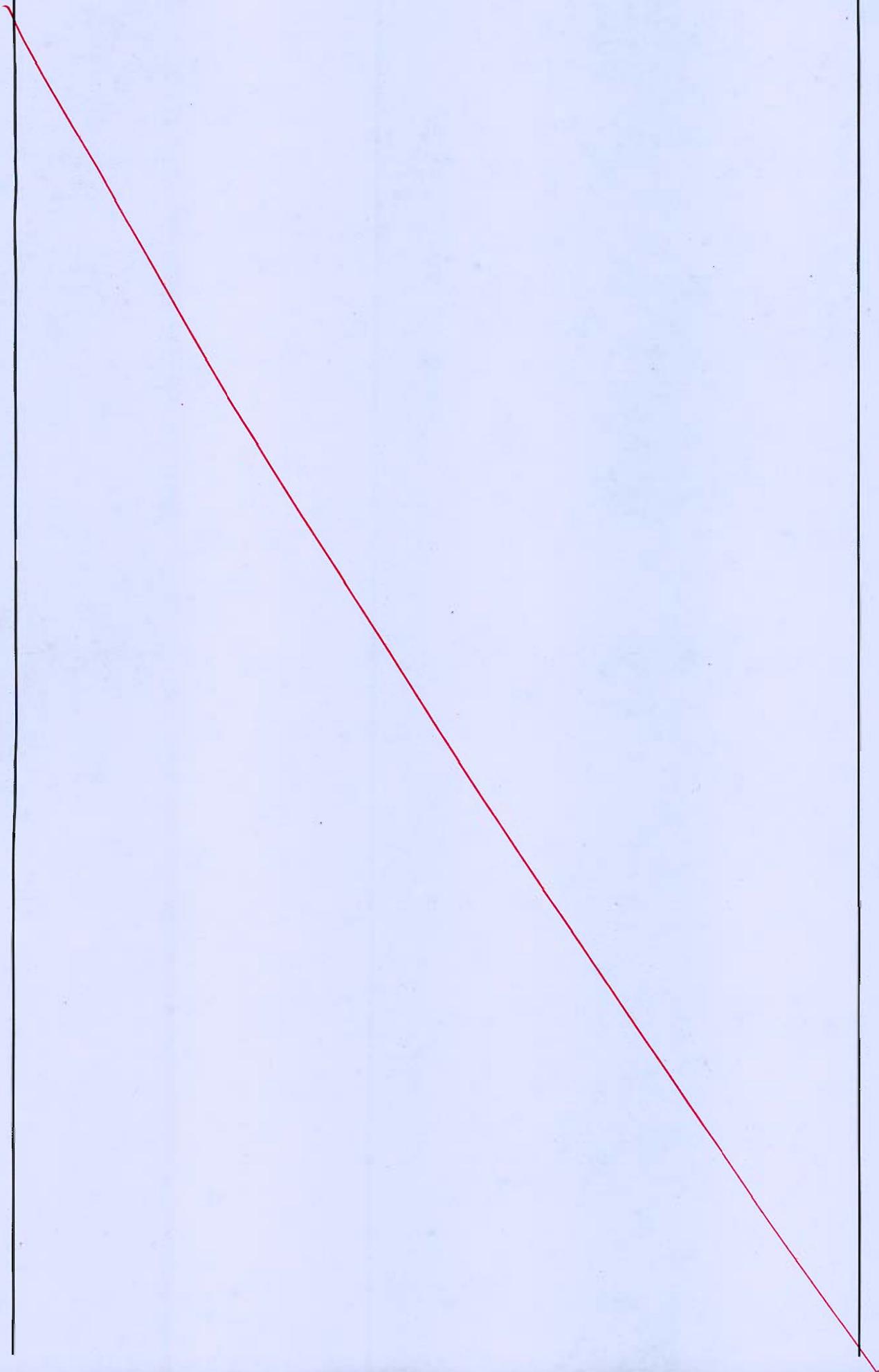


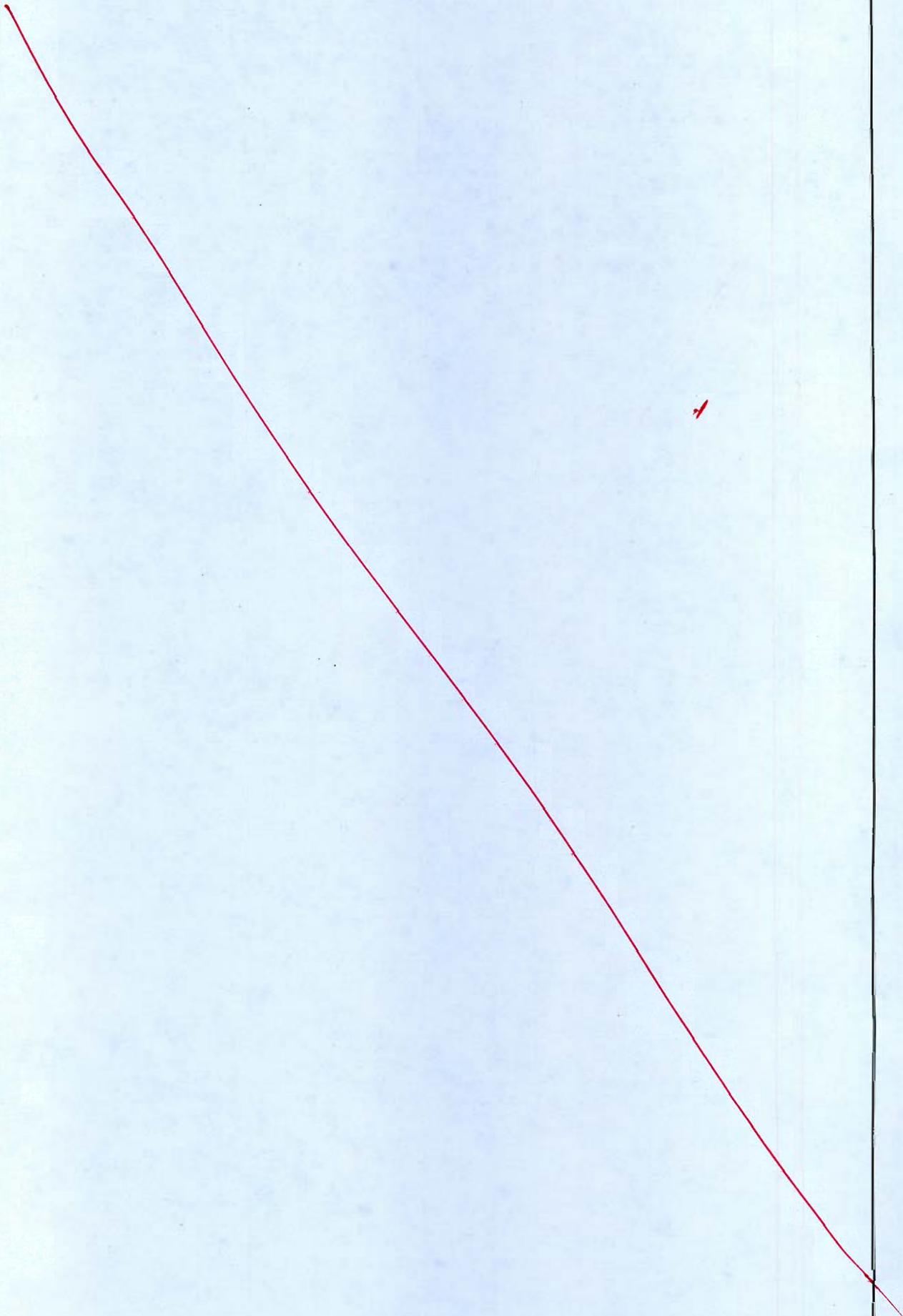


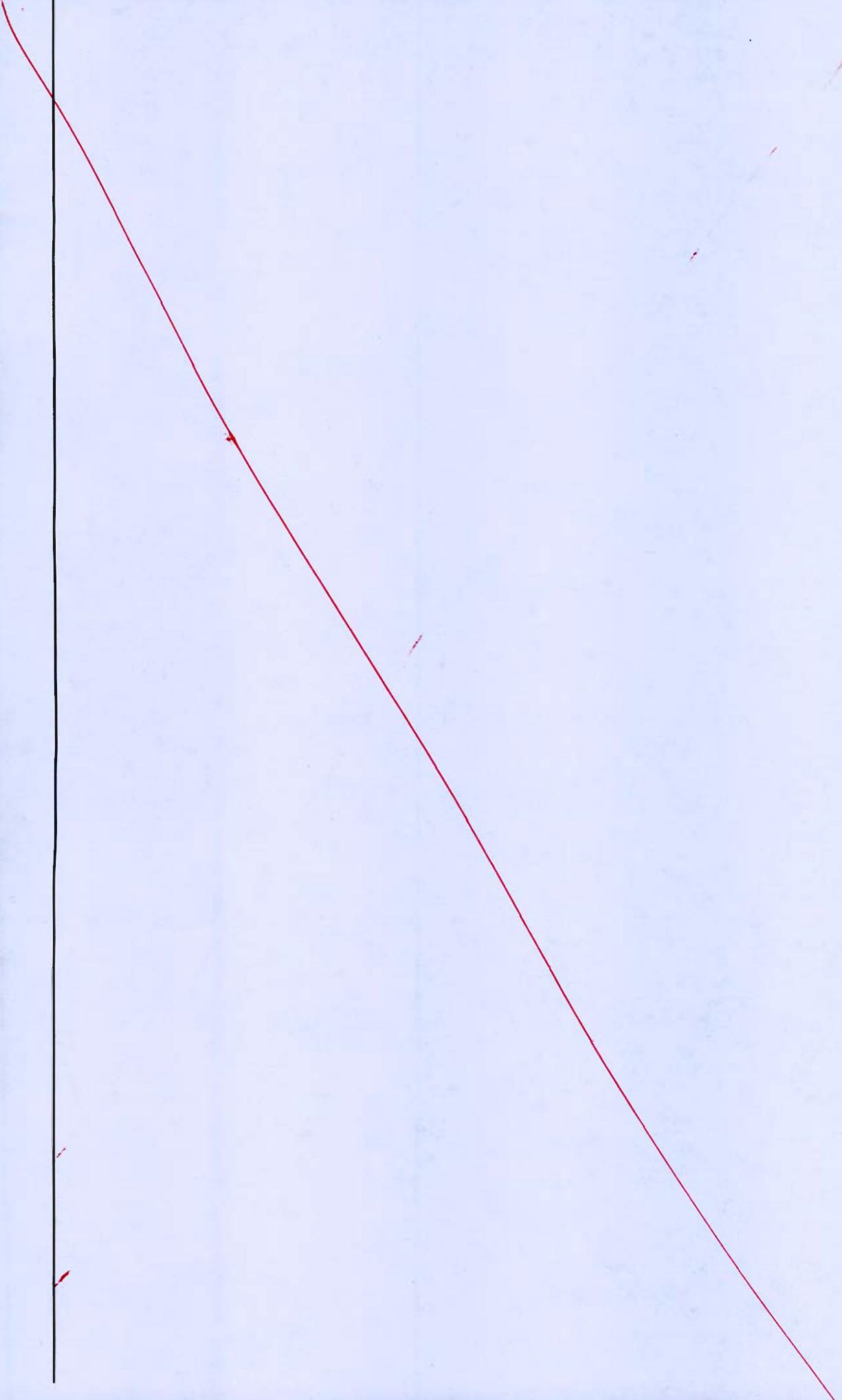
- Q.8 (a)** (i) A block 'A' weighing 840 N is raised up with the help of two 8° wedge 'B' and 'C' of negligible weights as shown in figure. If the coefficient of static friction is 0.25 for all surfaces of contact, then evaluate the smallest force P to be applied to raise the block A
- (ii) A vehicle of mass 680 kg, is moving with a velocity 34 m/s. A force of 255 N acts on it for 3 minutes. Find the final velocity of the vehicle:
- (1) When the force acts in the direction of motion, and
 - (2) When the force acts in the opposite direction of the motion.
- Comment your view on final result.



[14 +6 = 20 marks]





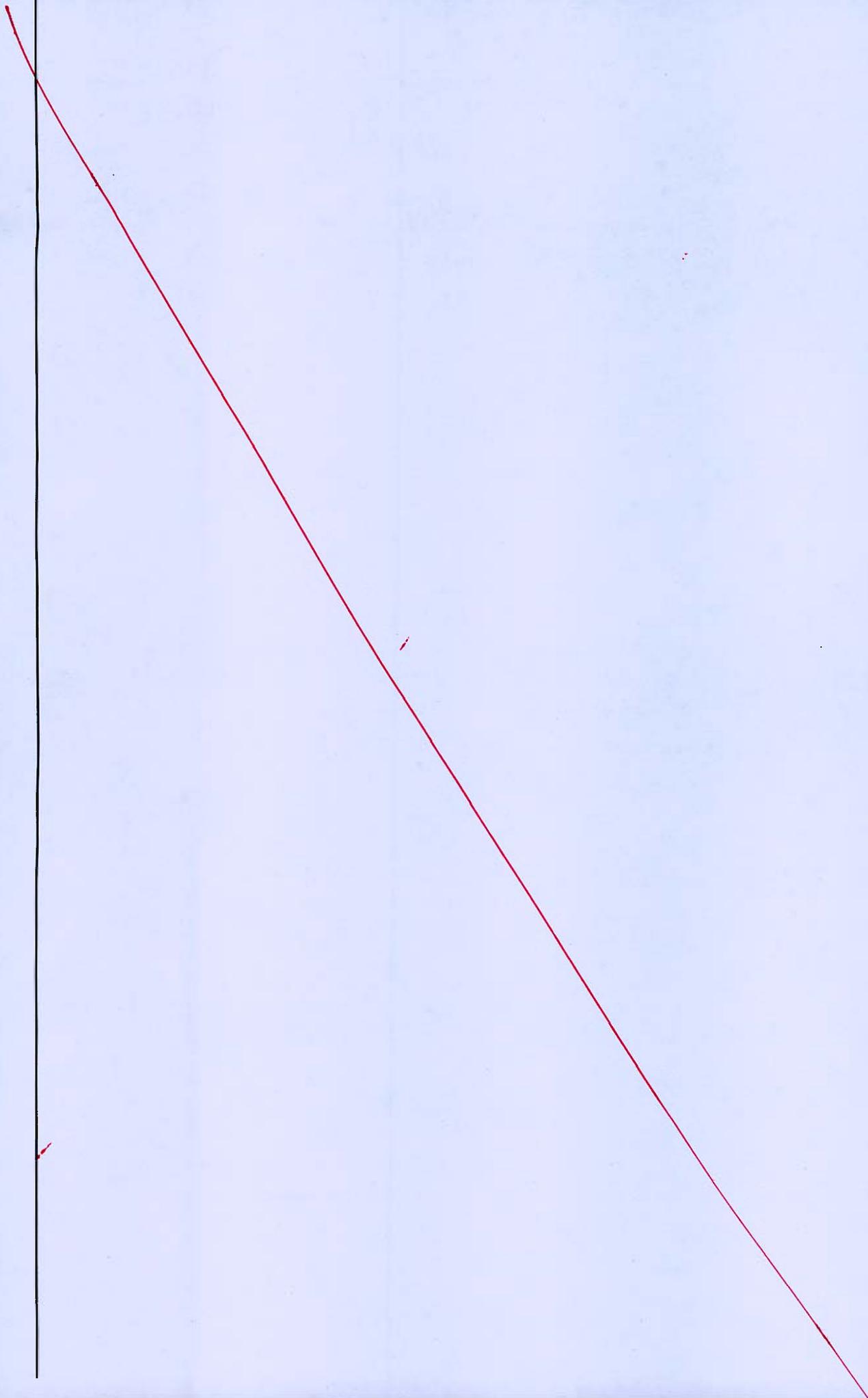


Q.8 (b) A gun metal bar of 30 mm diameter is completely enclosed in a steel tube, 30 mm internal diameter and 45 mm external diameter. A pin, 12 mm in diameter is fitted transversely to the axis of the bar near each end, to secure the bar to the tube. Calculate the intensity of stresses induced in bar, tube and pin, when the temperature of the whole assembly is raised by 42°C .

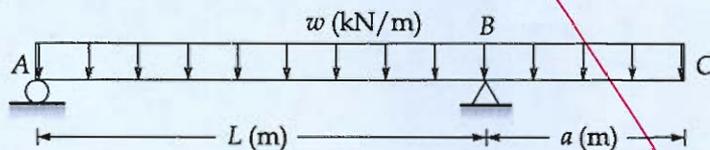
Take : For gun metal : $\alpha_g = 20 \times 10^{-6}/^{\circ}\text{C}$
 $E_g = 0.91 \times 10^5 \text{ N/mm}^2$

For Steel : $\alpha_s = 12 \times 10^{-6}/^{\circ}\text{C}$
 $E_s = 2 \times 10^5 \text{ N/mm}^2$

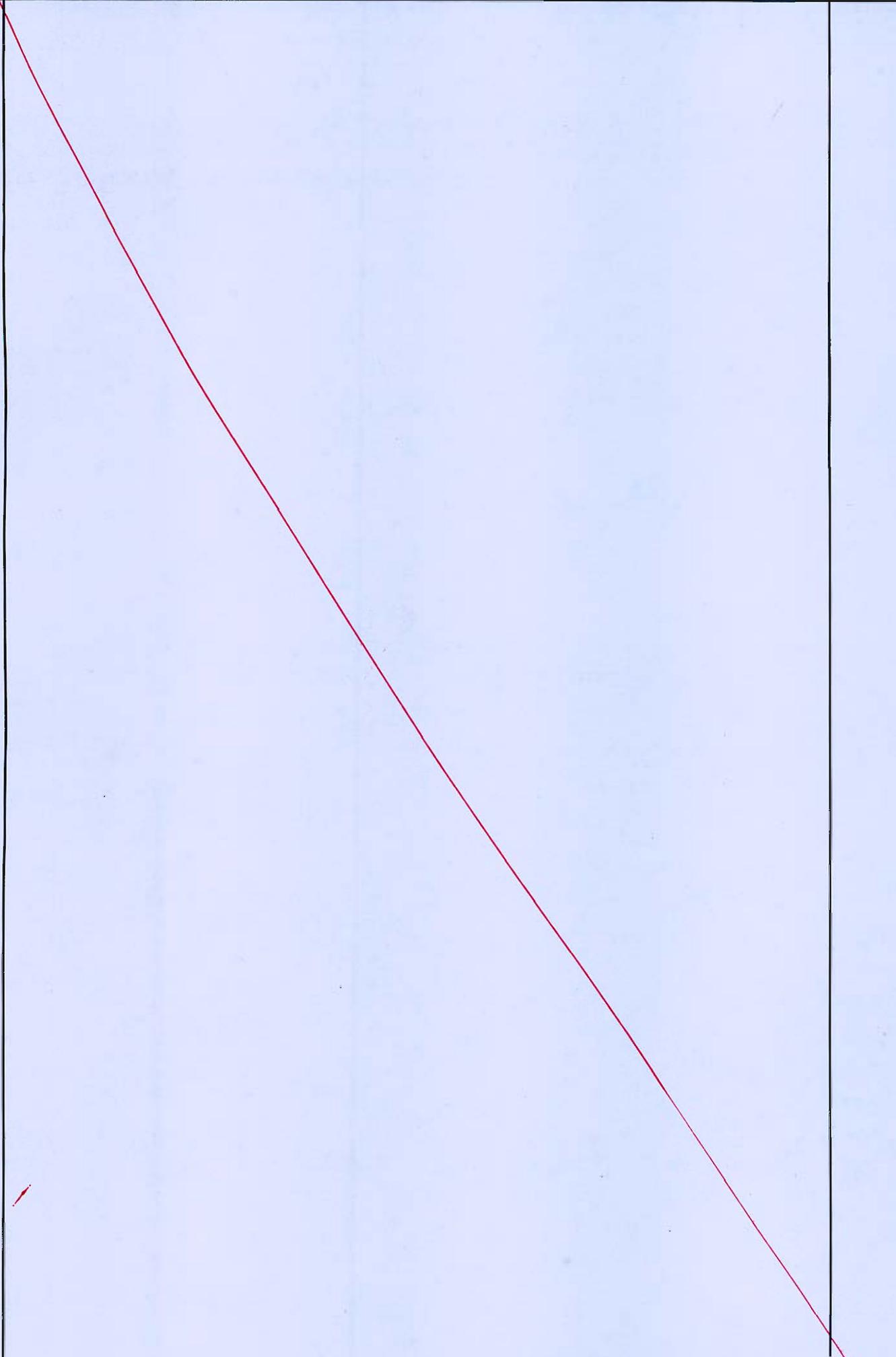
[20 marks]

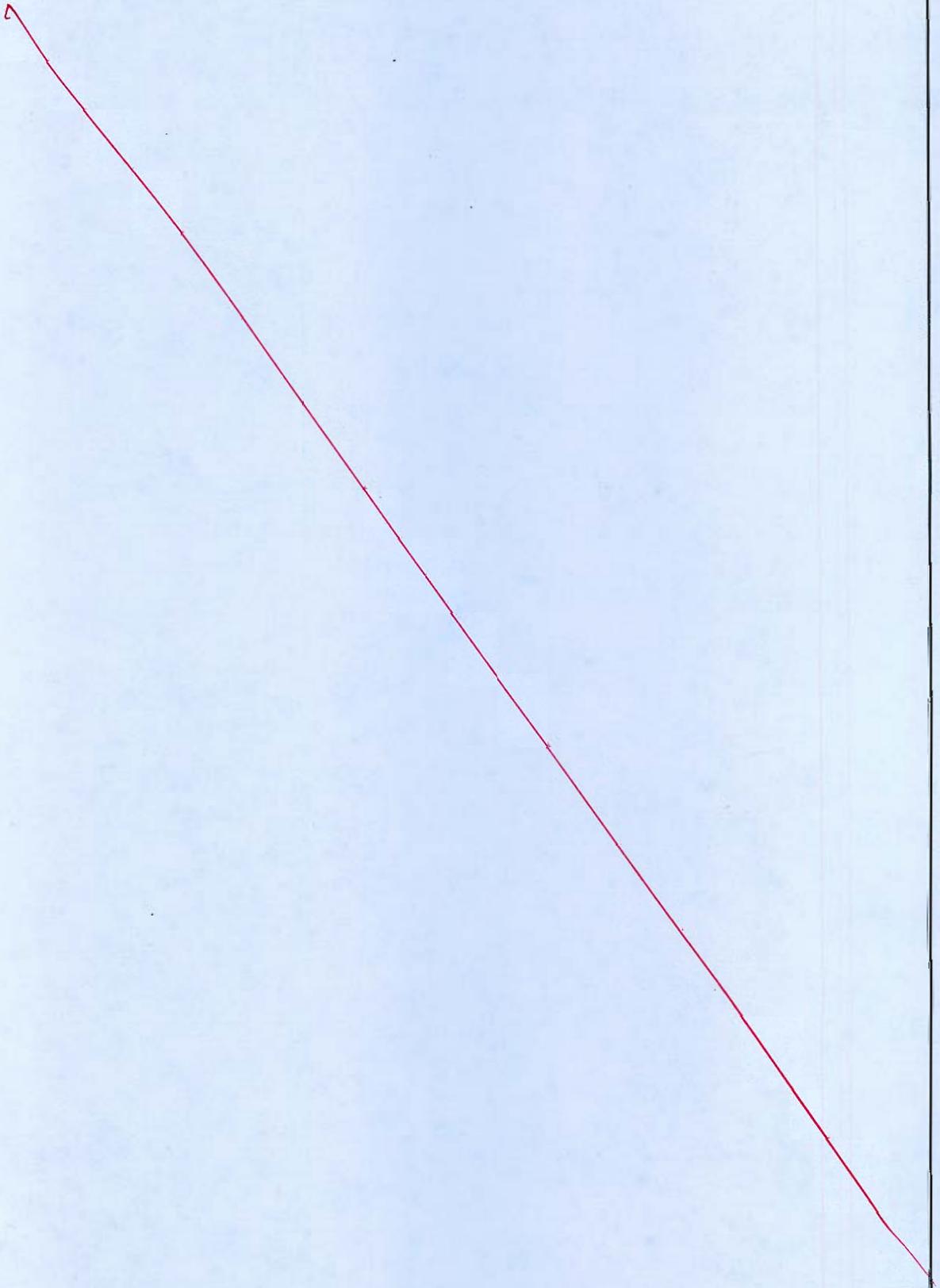


- Q.8 (c) A simply supported beam with one side overhang is loaded with uniformly distributed load as shown in figure below. Comment your result on the variation of reaction forces at A and B , if ' L ' is less than, equal to and greater than ' a '. Draw shear force and bending moment diagram for $w = 4 \text{ kN/m}$, $L = 8 \text{ m}$ and $a = 4 \text{ m}$.

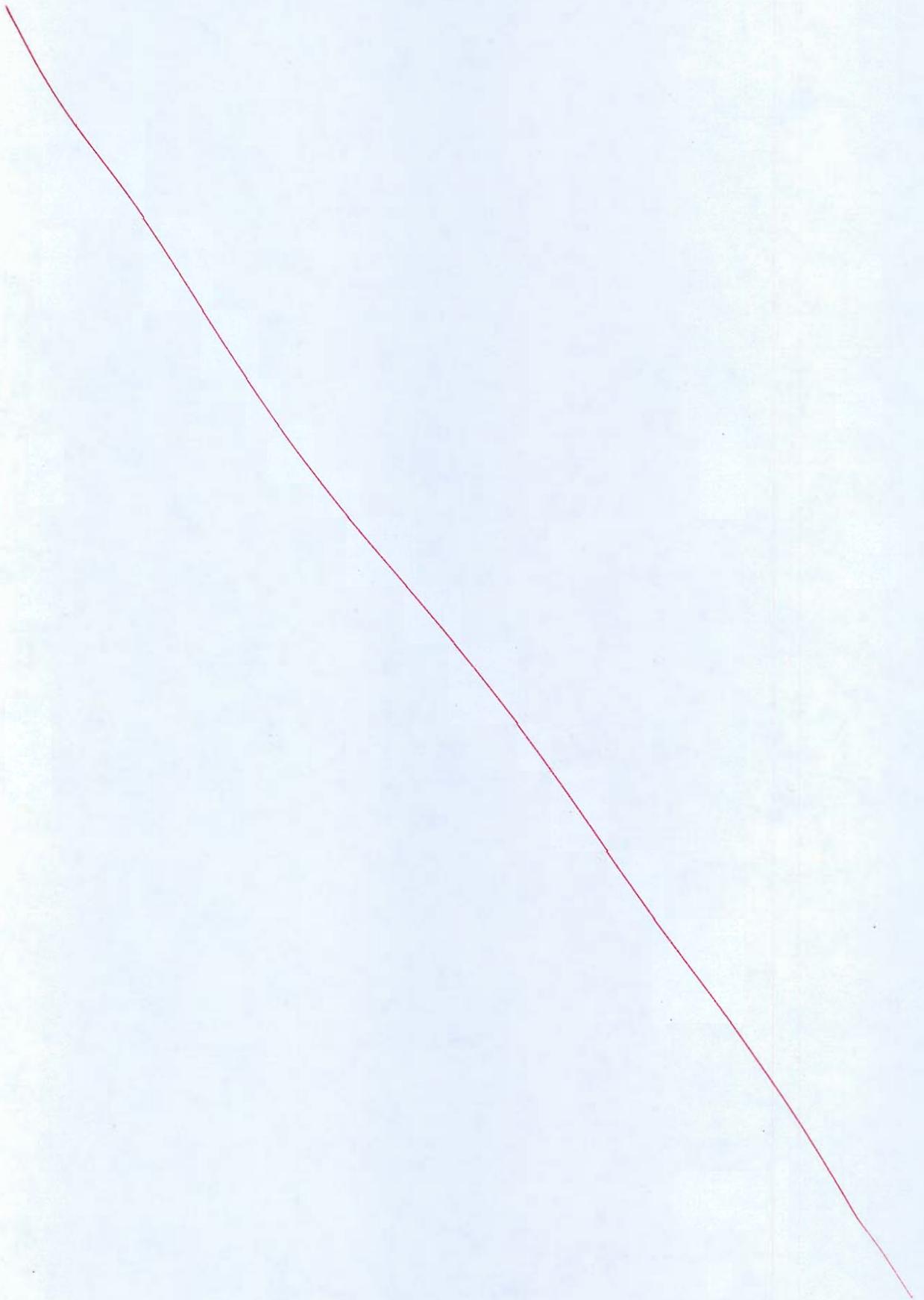


[20 marks]





Space for Rough Work



Space for Rough Work

Space for Rough Work

