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ENGINEERING MECHANICS

MECHANICAL ENGINEERING

Date of Test : 17/09/2023

ANSWER KEY >

- | | | | | |
|--------|---------|---------|---------|---------|
| 1. (b) | 7. (c) | 13. (c) | 19. (c) | 25. (b) |
| 2. (d) | 8. (b) | 14. (a) | 20. (d) | 26. (d) |
| 3. (d) | 9. (c) | 15. (d) | 21. (b) | 27. (c) |
| 4. (c) | 10. (a) | 16. (b) | 22. (c) | 28. (a) |
| 5. (c) | 11. (b) | 17. (a) | 23. (a) | 29. (c) |
| 6. (b) | 12. (c) | 18. (b) | 24. (b) | 30. (a) |

DETAILED EXPLANATIONS

1. (b)

As per given information,

$$h = 40 \text{ m}, u = 50 \text{ m/s}$$

Let the speed be ' v ' when it strikes to the ground

Apply law of conservation of energy,

$$mgh + \frac{1}{2}mu^2 = \frac{1}{2}mv^2$$

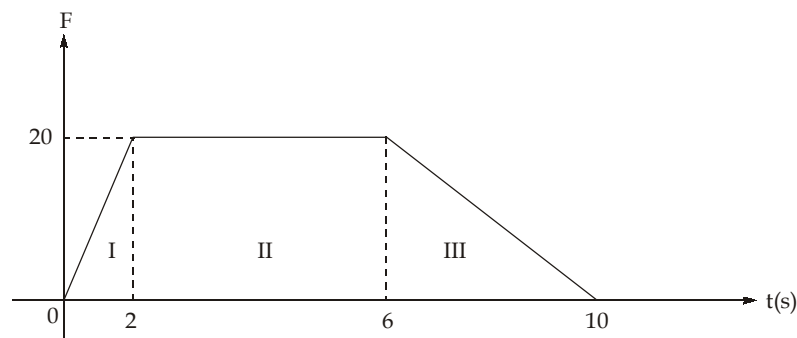
$$m \times 10 \times 40 + \frac{1}{2} \times m \times (50)^2 = \frac{1}{2} \times m \times v^2$$

$$400 + 1250 = \frac{v^2}{2}$$

$$v = 57.44 \text{ m/s}$$

2. (d)

As per given information,



$$\text{Impulse} = \int_{t_1}^{t_2} F(t) dt = \text{Area under (F - t) curve}$$

$$\text{Impulse} = \text{Area (I + II + III)}$$

$$= \frac{1}{2} \times 2 \times 20 + 4 \times 20 + \frac{1}{2} \times 4 \times 20$$

$$= 140 \text{ kg.m/s or Ns}$$

3. (d)

The block is displaced 2.5 m towards left,

Let the velocity of the body be v at mean position

As dissipative force, (friction = 0)

$$(\text{K.E.})_{\text{max}} = (\text{P.E.})_{\text{max}}$$

$$\frac{1}{2}mv_{\text{max}}^2 = \frac{1}{2} \times k_1 \times x^2 + \frac{1}{2} \times k_2 \times x^2$$

$$mv_{\text{max}}^2 = k_1 \times x^2 + k_2 x^2$$

Substituting the given value,

$$100 \times mv_{\text{max}}^2 = 40x^2 + 60x^2$$

$$100 m v_{\max}^2 = 100 x^2$$

$$v_{\max} = x = 2.5 \text{ m/s}$$

4. (c)

$$\begin{aligned} \text{Lagrangian, } L &= T - V \\ &= v^2 \dot{u}^2 + 2\dot{v}^2 - u^2 + v^2 \\ &= v^2(1 + \dot{u}^2) + (2\dot{v}^2 - u^2) \end{aligned}$$

The equation of motion, using langrangian (L) for $q = u$,

$$\Rightarrow \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{u}} \right) - \frac{\partial L}{\partial u} = 0$$

$$\Rightarrow \frac{d}{dt} (2v^2 \dot{u}) - (-2u) = 0$$

$$\Rightarrow 2[v^2 \ddot{u} + 2v\dot{v}\dot{u}] + 2u = 0$$

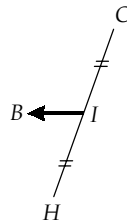
$$\Rightarrow 2v^2 \ddot{u} + 4v\dot{v}\dot{u} + 2u = 0$$

5. (c)

Force in member AH should be zero, as the AH is corner member with only two members connected to each other at 90° . Hence, in both members AH and GH force is zero.

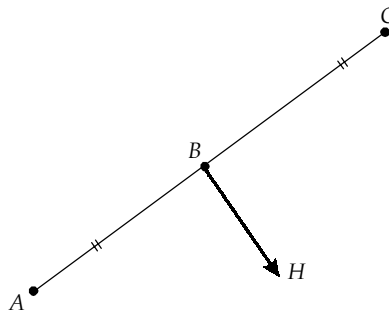
6. (b)

As in the given truss,



$$F_{CI} = F_{IH}$$

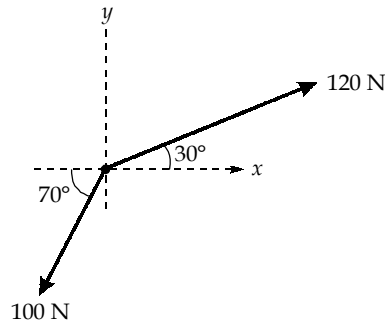
Therefore, $F_{BI} = 0$ (zero force member)



$$F_{CB} = F_{AB}$$

Therefore, $F_{BH} = 0$ (also zero force member)

7. (c)



$$\Sigma F_x = 120 \cos 30^\circ - 100 \cos 70^\circ = 69.72 \text{ N}$$

$$\Sigma F_y = 120 \sin 30^\circ - 100 \sin 70^\circ = -33.969 \text{ N}$$

$$\begin{aligned} \text{Resultant force, } R &= \sqrt{(F_x)^2 + (F_y)^2} \\ &= \sqrt{(69.72)^2 + (-33.969)^2} = 77.55 \text{ N} \approx 78 \text{ N} \end{aligned}$$

8. (b)

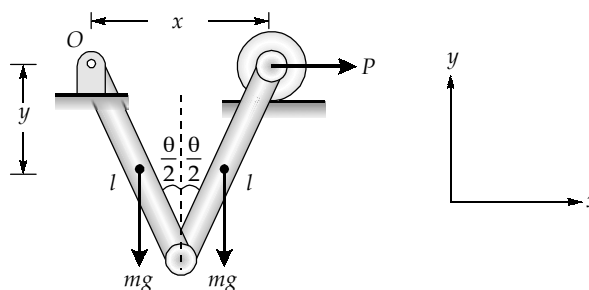
Conservation of linear momentum,

$$(m_A v_A)_i + (m_B v_B)_i = (m_A + m_B) v_f$$

$$15000 \times 1.5 + (-12000 \times 0.75) = 27000 \times v_f$$

$$v_f = 0.5 \text{ m/s}$$

9. (c)



$$x = 2l \sin \frac{\theta}{2}$$

$$\partial x = l \cos \frac{\theta}{2} \partial \theta$$

$$y = -\frac{l}{2} \cos \frac{\theta}{2}$$

$$\partial y = +\frac{l}{4} \sin \frac{\theta}{2} \partial \theta$$

$$+P(\partial x) + (-2mg) \cdot \partial y = 0$$

$$P \left(l \cos \frac{\theta}{2} \partial \theta \right) - 2mg \left(\frac{l}{4} \sin \frac{\theta}{2} \partial \theta \right) = 0$$

$$Pl \cos \frac{\theta}{2} \partial\theta = 2mg \times \frac{l}{4} \sin \frac{\theta}{2} \partial\theta$$

$$\tan \frac{\theta}{2} = \frac{2P}{mg}$$

$$\theta = 2 \tan^{-1} \left(\frac{2P}{mg} \right)$$

10. (a)

Using energy conservation

$$(PE)_A + (KE)_A = (PE)_B + (KE)_B \quad [\text{No energy loss due to smooth surface}]$$

$$mg \times 12 + \frac{1}{2}m \times 0^2 = mg \times 8 + \frac{1}{2}mV_B^2$$

$$\frac{mg \times 4 \times 2}{m} = V_B^2$$

$$V_B = \sqrt{8 \times 9.81} = 8.859 \text{ m/s}$$

11. (b)

As per given information,

$$m = 30 \text{ kg}; \quad r = 0.2 \text{ m}$$

$$\omega = 20 \text{ rad/s}; \quad T = 5 \text{ Nm}$$

$$F = 10 \text{ N}$$

$$I = \frac{1}{2}mr^2 = \frac{1}{2} \times 30 \times 0.2^2 = 0.6 \text{ kg.m}^2$$

Let the disk rotate an angle of θ rad.

From work energy principle

$$T \cdot \theta + F \times r \cdot \theta = \frac{1}{2} \times I \times \omega^2 \quad [\because \text{Workdone} = \text{change in energy}]$$

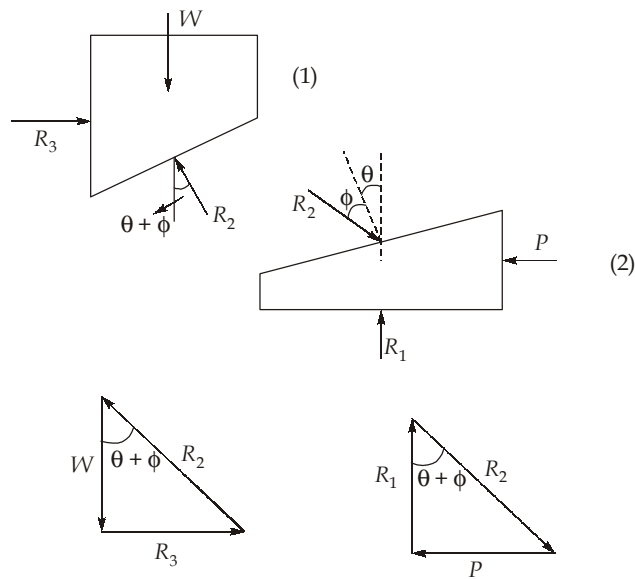
$$5 \cdot \theta + 10 \times 0.2 \times \theta = \frac{1}{2} \times 0.6 \times (20)^2$$

$$7 \cdot \theta = 120$$

$$\theta = 17.14 \text{ rad}$$

$$\text{Number of revolutions} = \frac{\theta}{2\pi} = \frac{17.14}{2\pi} = 2.73 \text{ rev}$$

12. (c)



$$\cos(\theta + \phi) = \frac{W}{R_2}, \quad \sin(\theta + \phi) = \frac{P}{R_2}$$

$$\frac{P}{W} = \tan(\theta + \phi)$$

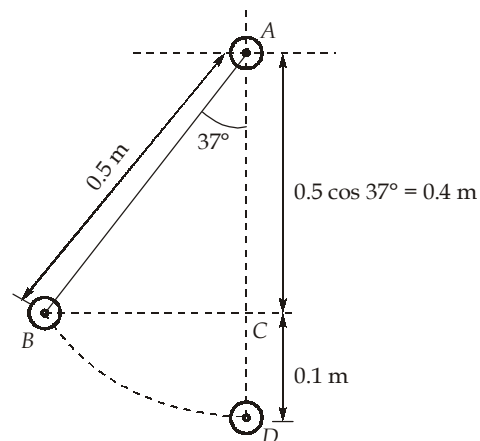
$$P = W \tan(\theta + \phi)$$

$$= 500 \tan(\theta + \phi)$$

$$\phi = \tan^{-1} 0.20 = 11.309$$

$$P = 500 \tan(15 + 11.309) = 247.21 \text{ N}$$

13. (c)



$$CD = AD - AC$$

$$= 0.5 - 0.5 \cos 37^\circ$$

$$= 0.1 \text{ m}$$

Applying energy conservation between B and D

$$mg \times CD = \frac{1}{2}mv^2 \quad \therefore V = \text{Velocity at D}$$

Let

$$g = 10 \text{ m/s}^2$$

$$10 \times 0.1 = 0.5v^2$$

$$v^2 = 2$$



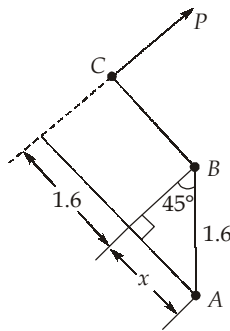
At point 'D'

$$\text{Tension, } T = \frac{mv^2}{r} + mg$$

$$T = m \left(\frac{v^2}{r} + g \right) = 0.1 \left(\frac{2}{0.5} + 10 \right)$$

$$T = 1.4 \text{ N}$$

14. (a)



$$\text{Moment of P about A} = P \times (1.6 + x)$$

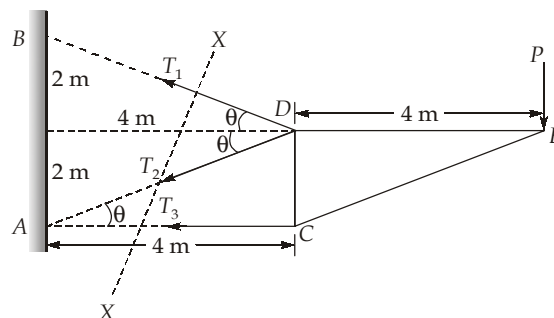
$$\sin 45^\circ = \frac{x}{1.6}$$

$$x = 1.6 \sin 45^\circ = 1.1314 \text{ m}$$

$$M_A = 30 \times (1.6 + 1.1314) = 81.9 \text{ N-m}$$

15. (d)

At the section X-X through 3 members of truss as shown below:



Let force along bar BD is T_1

Now taking moment balance about A.

$$\Rightarrow \Sigma M_A = 0$$

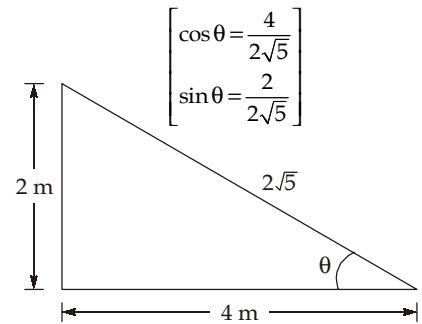
$$\Rightarrow$$

$$P \times 8 = T_1 \cos\theta \times 2 + T_1 \sin\theta \times 4$$

$$= T_1 \left[\frac{4 \times 2}{2\sqrt{5}} + \frac{2 \times 4}{2\sqrt{5}} \right]$$

$$P \times 8 = T_1 \times \frac{8}{\sqrt{5}}$$

$$T_1 = P\sqrt{5} = 125 \times \sqrt{5} = 279.5 \text{ kN (T)}$$



16. (b)

Given: $W = 6000 \text{ N}$, $r = \frac{50}{2} = 25 \text{ mm}$, $p = 10 \text{ mm}$

$$\tan\theta = \frac{p}{\pi d} = \frac{10}{\pi \times 50} = 0.06366$$

$$\theta = 3.643^\circ$$

$$\tan\phi = \mu = 0.05$$

$$\phi = \tan^{-1}0.05 = 2.862^\circ$$

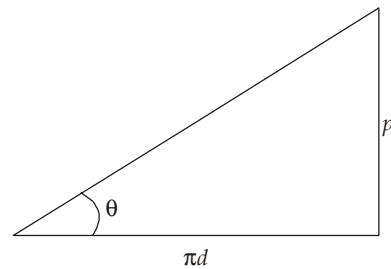
$$T = Wr \tan(\theta + \phi)$$

$$= 6000 \times 0.025 \tan(3.643 + 2.862)$$

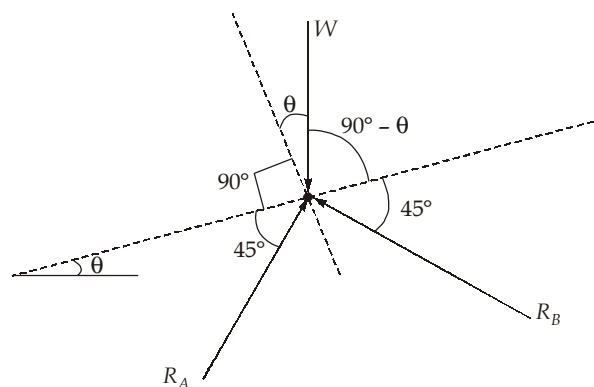
$$= 17.1036 \text{ Nm}$$

tangential force at end of lever effort

$$= \frac{17.1036}{0.3} = 57.01 \text{ N}$$



17. (a)



Apply Lami's theorem:

$$\frac{R_A}{\sin(135^\circ - \theta)} = \frac{R_B}{\sin(135^\circ + \theta)}$$

Given: $R_A = 2 R_B$

$$\Rightarrow 2 \sin(135^\circ + \theta) = \sin(135^\circ - \theta)$$

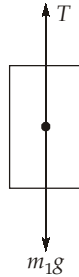
$$\Rightarrow 2[\sin 135^\circ \cos\theta + \cos 135^\circ \sin\theta] = [\sin 135^\circ \cos\theta - \cos 135^\circ \sin\theta]$$

$$\Rightarrow 3 \cos 135^\circ \sin\theta = -\sin 135^\circ \cos\theta$$

$$\Rightarrow \tan\theta = -\frac{1}{3}\tan 135^\circ$$

$$\theta = 18.43^\circ$$

18. (b)
Cylinder

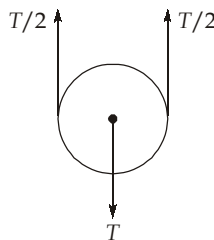


From Newton's first law,

$$m_1g - T = 0$$

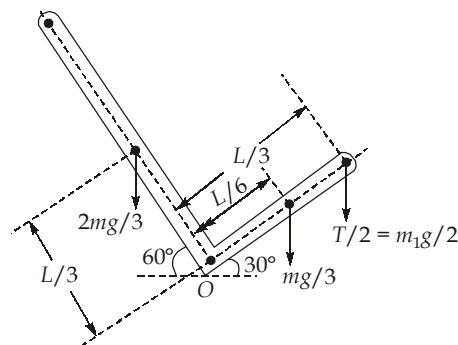
$$T = m_1g$$

Pulley



$$\frac{T}{2} = \frac{m_1g}{2}$$

To cause loss of contact at A, reaction at A will be zero.



$$\Sigma M_o = 0$$

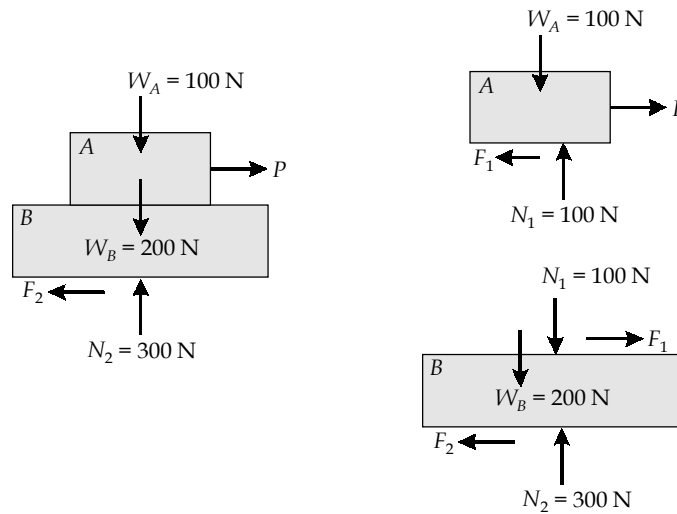
$$\frac{2mg}{3} \times \frac{L}{3} \cos 60^\circ - \frac{mg}{3} \times \frac{L}{6} \cos 30^\circ - \frac{T}{2} \times \frac{L}{3} \cos 30^\circ = 0$$

$$\Rightarrow \frac{2mg}{9} L \cos 60^\circ = \frac{m}{18} g L \cos 30^\circ + \frac{m_1g}{2} \times \frac{L}{3} \cos 30^\circ$$

$$\Rightarrow \frac{2m}{9} \cos 60^\circ = \frac{m}{18} \cos 30^\circ + \frac{m_1}{6} \cos 30^\circ$$

$$m_1 = 0.436 m$$

19. (c)
Given,



Assume impending sliding at surface, 1

$$F_1 = (F_1)_{\max} = (\mu_s)N_1 = 0.2 \times 100 = 20\text{ N}$$

From FBD of block 'A'

$$\begin{aligned} \sum F_x = 0, \quad P - F_1 &= 0 \\ P &= 20\text{ N} \end{aligned}$$

Assume impending sliding at surface force 2 only

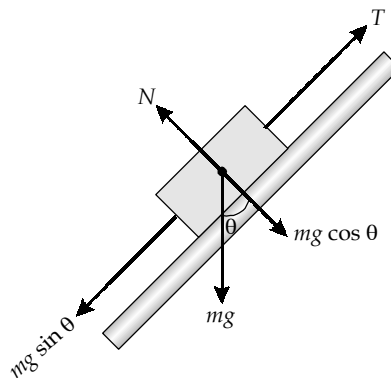
$$F_2 = (F_2)_{\max} = \mu_s N_2 = 0.1 \times 300 = 30\text{ N}$$

From FDB,

$$\begin{aligned} P - F_2 &= 0 \\ P &= 30\text{ N} \end{aligned}$$

$P = 20\text{ N}$ will cause motion to impend at surface 1 and that $P = 30\text{ N}$ will cause motion to impend at surface 2, therefore the largest force that can be applied without causing either block to move is $P = 20\text{ N}$.

20. (d)



$$\begin{aligned} T &= \mu mg \cos \theta + mg \sin \theta \\ &= 0.2 \times 10 \times \frac{1}{\sqrt{2}} + \frac{10}{\sqrt{2}} \\ &= 1.2 \times \frac{10}{\sqrt{2}} = 8.485\text{ N} \\ P_{\min} &= 8T = 8 \times 4.855 = 67.88 \end{aligned}$$

21. (b)

$$\text{Velocity, } v = (3t^2 - 6t) \text{ m/s}$$

$$\frac{ds}{dt} = v$$

$$s = (t^3 - 3t^2) \text{ m}$$

At $t = 0$

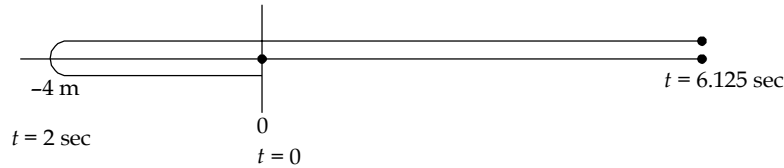
$$s = 0$$

$t = 2$

$$s = -4 \text{ m}$$

$t = 3.5$

$$s = 6.125$$



So, total distance travelled by the particle

$$s = 4 + 4 + 6.125 = 14.125 \text{ m}$$

22. (c)

Moment about the point 'C'

$$\text{(Vector method), } M_C = \vec{r} \times \vec{F}$$

$$\text{Force vector, } \vec{F} = 500 \frac{\overline{AB}}{|\overline{AB}|}$$

$$= 500 \left(\frac{2\hat{i} - 4\hat{j} + 3\hat{k}}{\sqrt{(2)^2 + (-4)^2 + (3)^2}} \right)$$

$$= 92.847(2\hat{i} - 4\hat{j} + 3\hat{k})$$

$$\text{Position vector, } r_{CA} = -2\hat{i} - 0\hat{j} + 0\hat{k}$$

$$M_C = \vec{r}_{CA} \times \vec{F}$$

$$= (-2\hat{i})[92.847(2\hat{i} - 4\hat{j} + 3\hat{k})]$$

$$M_C = 92.847 \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ -2 & 0 & 0 \\ 2 & -4 & 3 \end{vmatrix}$$

$$= 92.847(6\hat{j} + 8\hat{k})$$

$$= 557.086\hat{j} + 742.776\hat{k}$$

$$\text{Magnitude, } M_C = \sqrt{(557.086)^2 + (742.776)^2}$$

$$M_C = 928.47 \text{ Nm}$$

23. (a)

Let momentum, $P = at^2 + bt + c$

$$\text{force, } F = \frac{dP}{dt} \text{ at } t = 0 \text{ is } 80 \text{ N}$$

$$F = 2a \times 0 + b$$

$$80 = b$$

 \Rightarrow

$$b = 80$$

Now,

$$\text{at } t = 2 \text{ sec}$$

$$F = 2a \times 2 + 80$$

$$\frac{480 - 80}{4} = a$$

$$a = 100$$

$$F = 2at + 80$$

Now,

$$F = 200t + 80$$

$$\text{Acceleration} = \frac{F}{m} = \frac{200t + 80}{5} = 40t + 16$$

$$\text{Acceleration} = \frac{dv}{dt}$$

$$dv = (40t + 16)dt$$

$$\int_0^v dv = \int_0^5 (40t + 16)dt$$

$$v - 0 = \left[40 \times \frac{t^2}{2} + 16t \right]_0^5$$

$$v = 40 \times \frac{5^2}{2} + 16 \times 5$$

$$v = 25 \times 20 + 80$$

$$v = 580 \text{ m/s}$$

24. (b)

$$\sin\theta = \frac{r}{L}$$

(a) Tension T along the string(b) The weight mg vertically downwards.

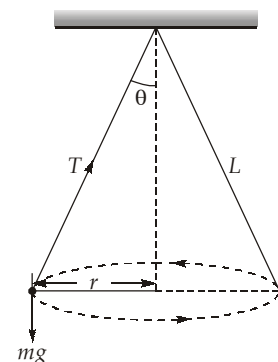
In radial direction,

$$T \sin\theta = \frac{mv^2}{r}$$

In vertical direction, $T \cos\theta = mg$

Equation (i) and (ii)

$$\tan\theta = \frac{v^2}{rg}$$



$$v = \sqrt{(rg \tan \theta)} = \sqrt{(rg) \times \left(\frac{r}{\sqrt{L^2 - r^2}} \right)}$$

$$v = \frac{r\sqrt{g}}{(L^2 - r^2)^{1/4}}$$

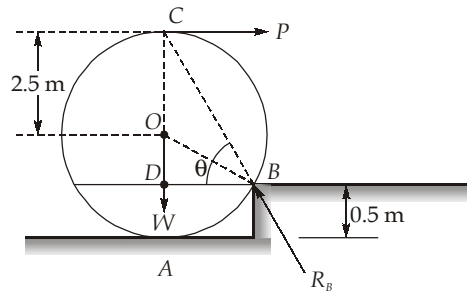
By equation (ii),

$$T = \frac{mg}{\cos \theta} = \frac{mgL}{(\sqrt{L^2 - r^2})}$$

$$T = \frac{mgL}{(L^2 - r^2)^{1/2}}$$

25. (b)

For a body under three forces to be in equilibrium, these forces must be coplanar and concurrent. So, force P , weight of cylinder W and reaction at $B(R_B)$ passes through the same point that's the upper most point of cylinder.



In $\triangle OBD$,

$$\begin{aligned} OB^2 &= OD^2 + DB^2 \\ 2.5^2 &= (2.5 - 0.5)^2 + DB^2 \\ DB^2 &= 2.25 \\ DB &= 1.5 \text{ m} \end{aligned}$$

In $\triangle BCD$,

$$\begin{aligned} \tan \theta &= \frac{2.5 + 2}{1.5} = \frac{4.5}{1.5} = 3 \\ \theta &= 71.565^\circ \end{aligned}$$

$$\frac{W}{P} = \tan \theta$$

$$\Rightarrow \tan 71.565^\circ = \frac{800}{P}$$

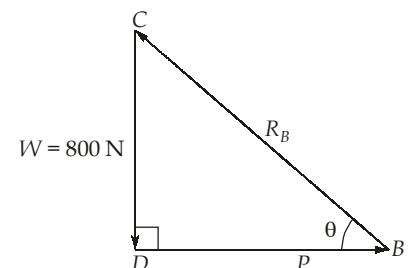
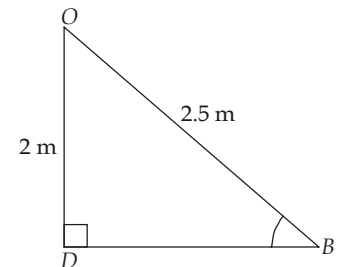
$$\Rightarrow P = \frac{800}{3} = 266.67 \text{ N}$$

Alternate: Taking moment about B ,

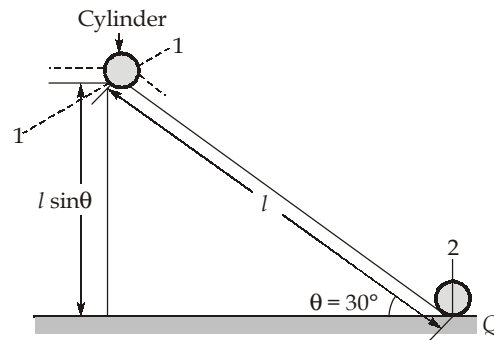
$$P \times CD = W \times DB$$

$$P = \frac{800 \times 1.5}{4.5}$$

$$P = \frac{800}{3} = 266.67 \text{ N}$$



26. (d)



Cylinder rolls without slipping with angular speed $\omega = \frac{v}{r}$ about its axis.

By energy conservation,

$$(KE)_{1-1} + (PE)_{1-1} = (KE)_{2-2} + (PE)_{2-2}$$

$$0 + mgl\sin\theta = \frac{1}{2}mv^2 + \frac{1}{2}I\omega^2 + 0$$

$$mgl\sin\theta = \frac{1}{2}mv^2 + \frac{1}{2}\left(\frac{1}{2}mR^2\right)\omega^2 = \frac{1}{2}mv^2 + \frac{1}{4}mv^2$$

$$mgl\sin\theta = \frac{3}{4}mv^2$$

$$v = \sqrt{\frac{4}{3}gl\sin\theta} = \sqrt{\frac{4}{3} \times 10 \times 2 \times \sin 30^\circ}$$

$$v = 3.65 \text{ m/s}$$

27. (c)

$$F \propto v^3$$

$$F = kv^3$$

$$a = \frac{F}{m} = \frac{kv^3}{m}$$

We know that,

$$a = \frac{dv}{dt}$$

$$\frac{dv}{dt} = \frac{kv^3}{m}$$

$$\frac{1}{v^3}dv = \left(\frac{k}{m}\right)dt$$

$$\left[\frac{v^{-2}}{-2}\right]_u^v = \left(\frac{k}{m}\right)\int_0^t dt$$

$$-\frac{1}{2}\left[\frac{1}{v^2}\right]_u^v = \left(\frac{k}{m}\right)[t]_0^t$$

$$-\frac{1}{2}\left[\frac{u^2 - v^2}{v^2u^2}\right] = \left(\frac{k}{m}\right)t$$

$$t \propto \left(\frac{u^2 - v^2}{v^2u^2}\right)$$

28. (a)

Given data: $m = 8.4 \text{ kg}$, $\omega = 6.9 \text{ rad/s}$, $F = 6.6 \text{ N}$, $M = 59 \text{ Nm}$, $L = 4 \text{ m}$, $\omega_{\theta=90^\circ} = ?$

Moment of Inertia of rod about hinge O ,

$$I_O = \frac{mL^2}{12} + m \times \left(\frac{L}{2}\right)^2 = \frac{mL^2}{3} = \frac{8.4 \times 4 \times 4}{3} = 44.8 \text{ kg m}^2.$$

By conservation of energy:

$$mgh_{cm} + (M + F \times L)\Delta\theta = \frac{1}{2}I_0(\omega_1^2 - \omega_0^2)$$

$$\text{For } \theta = 90^\circ, h_{cm} = 2 \text{ m}$$

$$(8.4 \times 9.81 \times 2) + (59 + 6.6 \times 4) \times \frac{\pi}{2} = \frac{1}{2} \times (44.8) [\omega_1^2 - 6.9^2]$$

$$\frac{298.954 \times 2}{44.8} = \omega_1^2 - 6.9^2$$

$$\omega_1^2 = 60.9562$$

$$\omega_1 = 7.807 \text{ rad/s}$$

29. (c)

Normal reaction, $N = mg \cos 30^\circ$

friction force, $f = \mu N$

$$f = \mu mg \cos 30^\circ$$

Now, resultant force in downward direction will be $= (mg \sin 30^\circ - \mu mg \cos 30^\circ)$

We know that,

$$a = \frac{F}{m} = \frac{m(g \sin 30^\circ - \mu g \cos 30^\circ)}{m}$$

$$\frac{dv}{dt} = g(0.5 - 0.866 \times 3x)$$

$$v \frac{dv}{dx} = g(0.5 - 0.866 \times 3x)$$

Given, $v_i = 0, v_f = 0$

Integrating, $\int_0^0 v dv = g \int_0^x (0.5 - 0.866 \times 3x) dx$

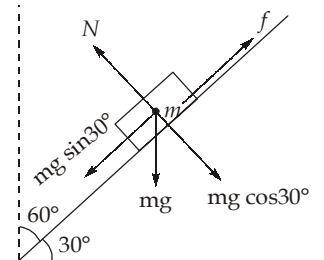
$$\left[\frac{v^2}{2} \right]_0^0 = g \left[0.5x - \frac{0.866}{2} \times 3x^2 \right]_0^x$$

$$0 = 0.5x - 0.433 \times 3x^2$$

$$0.433 \times 3x^2 = 0.5x$$

$$x = \frac{0.5}{0.433 \times 3}$$

$$x = 0.3849 \text{ m}$$



30. (a)

Given that, $m = 20 \text{ kg}$, $r_i = 0.1 \text{ m} = R$, $r_o = 0.3 \text{ m} = 3R$

Cross-section area = $\pi[(3R)^2 - R^2] = 8\pi R^2$

Mass of small strip of thickness ' dr ' at a distance ' r ' from centre.

$$dm = \frac{m \times 2\pi r dr}{8\pi R^2}$$

We know that,

$$I = \int_R^{3R} dm \cdot r^2$$

$$I = \int_R^{3R} \frac{m \times 2\pi r dr \times r^2}{8\pi R^2} = \frac{m}{4R^2} \times \left[\frac{r^4}{4} \right]_R^{3R}$$

$$= \frac{m}{4 \times 4R^2} \times (81R^4 - R^4) = \frac{20}{4} mR^2 = 5 mR^2$$

$$(\text{K.E.})_{\text{total}} = (\text{K.E.})_{\text{rotational}} + (\text{K.E.})_{\text{translation}}$$

$$\text{K.E.} = \frac{1}{2} m v^2 + \frac{1}{2} I \omega^2$$

$$= \frac{1}{2} m \times 9R^2 \omega^2 + \frac{1}{2} I \omega^2 = \frac{9}{2} mR^2 \omega^2 + \frac{5}{2} mR^2 \omega^2$$

$$\text{K.E.} = 7 mR^2 \omega^2$$

$$62.5 = 7 \times 20 \times (0.1)^2 \times \omega^2$$

$$\omega = 6.6815 \text{ rad/s}$$

$$v = 3R\omega = 3 \times 0.1 \times 6.6815$$

Velocity of centre of mass, $v = 2 \text{ m/s}$

Note: for rolling without slipping, $v = r\omega$, $v = 3R\omega$

