

**CLASS TEST**



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**CLASS TEST  
2019-20**

**MECHANICAL  
ENGINEERING**

**Subject : Thermodynamics**

**Date of test : 10/02/19**

*Answer Key*

- |        |         |         |         |         |
|--------|---------|---------|---------|---------|
| 1. (c) | 7. (b)  | 13. (d) | 19. (a) | 25. (d) |
| 2. (a) | 8. (c)  | 14. (b) | 20. (b) | 26. (c) |
| 3. (d) | 9. (a)  | 15. (c) | 21. (b) | 27. (d) |
| 4. (b) | 10. (d) | 16. (c) | 22. (d) | 28. (d) |
| 5. (a) | 11. (c) | 17. (a) | 23. (c) | 29. (a) |
| 6. (b) | 12. (d) | 18. (b) | 24. (a) | 30. (d) |

### Detailed Explanations

1. (c)

Quasi means almost. Every state in quasi-static process through which system passes remains almost in equilibrium.

4. (b)

The maximum inversion temperature of hydrogen and helium are 200 K and 24 K, so on throttling, the temperature of both will increase.

5. (a)

For charging of an evacuated adiabatic chamber,  $T_f = \gamma T_L$

where  $T_f$  = final temperature

$$\gamma = \frac{C_P}{C_V}$$

$T_L$  = supply line temperature

6. (b)

$$1 - \frac{T_2}{700} = 1 - \frac{448}{T_2}$$

$$T_2 = \sqrt{700 \times 448} = 560 \text{ K}$$

7. (b)

$$dW = \int_{V_1}^{V_2} p dV = p(V_f - V_i) = 0.1 \times (0.15 - 0.3)$$

$$= 0.1 \times (-0.15) = 0.015 \text{ MJ} = -15 \text{ kJ}$$

$$dQ = -35 \text{ kJ};$$

$$dQ = dU + dW$$

$$-35 = dU - 15$$

$$dU = -20 \text{ kJ}$$

8. (c)

$$\begin{aligned} W_{\text{net}} &= (T_1 - T_2)(s_1 - s_2) \\ &= (723 - 353)(2.5) \\ &= 925 \text{ kJ/kg} \end{aligned}$$

9. (a)

At constant entropy, the phase change of a saturated liquid does not take place by increasing the temperature. Thus, the resulting state will also be a liquid.

10. (d)

$$Tds = dh - vdp$$

At constant pressure  $dp = 0$

So,  $\left(\frac{dh}{ds}\right)_P = T$

13. (d)

Decrease in potential energy = Increase in heat energy

$$mgh = mc\Delta t$$

$$1 \times 9.81 \times 100 = 1 \times 4186 \times \Delta t$$

$$\Delta t = \frac{981}{4186} = 0.234^\circ\text{C}$$

14. (b)

$$\text{Loss of available energy} = Q_1 T_0 \frac{(T_1 - T_2)}{T_1 T_2} = \frac{2000 \times 300(600 - 400)}{600 \times 400} = 500 \text{ kJ}$$

18. (b)

$$\Delta PE = mg(h_2 - h_1) = 50 \times 9.81 \times 40 = 19620 \text{ J}$$

$$\Delta KE = \frac{1}{2}m(V_2^2 - V_1^2) = \frac{1}{2} \times 50 \times (30^2 - 10^2) = 20,000 \text{ J}$$

$$Q_{1-2} = 30,000 \text{ J}$$

$$W_{1-2} = \text{Work delivered} - \text{Work received} \\ = 0.002 \times 1000 \times 3600 - 4500 = 2700 \text{ J}$$

For non-flow process,

$$Q_{1-2} = W_{1-2} + dE$$

$$Q_{1-2} = W_{1-2} + dU + \Delta KE + \Delta PE$$

$$dU = 30,000 - 2700 - 20,000 - 19620 \\ = -12320 \text{ J}$$

19. (a)

From the relation,

$$Tds = du + pdv$$

$$ds = C_v \frac{dT}{T} + \frac{P}{T} dv = C_v \frac{dT}{T} + R \frac{dv}{V} \rightarrow 0$$

Upon integration,

$$s_2 - s_1 = \int C_v \frac{dT}{T} \\ = \int_{273}^{800} \left( \frac{0.65}{T} + 0.0002 \right) dT \\ = 0.65 \ln \frac{800}{273} + 0.0002(800 - 273) \\ = 0.6988 + 0.1054 = 0.8042 \text{ kJ/kgK}$$

20. (b)

Revolutions made by the workpiece in 10 min,

$$N = 180 \times 10 = 1800$$

$$\text{Torque, } T = \text{Force} \times \text{radius} = 100 \times \frac{0.3}{2} = 15 \text{ Nm}$$

Work done on the work piece

$$\delta W = 2\pi NT = 2\pi \times 1800 \times 15 = 169646 \text{ Nm (or J)}$$

(work will be -ve sign since it is being done on the work piece)

$$\delta Q = \delta W + \delta U$$

$$\delta Q = -169.64 + 125 = -44.64 \text{ kJ}$$

21. (b)

$$\text{Internal energy } U = f(T)$$

= Constant for isothermal process

$$\therefore U_1 = U_2$$

Change in Gibbs free energy, for an isothermal process is

$$\Delta G = RT \ln\left(\frac{P_2}{P_1}\right)$$

$$\text{For } P_2 > P_1, \Delta G = G_2 - G_1 > 0$$

$$\therefore G_2 > G_1$$

22. (d)

$$\text{Gas constant, } R = \frac{R_{\text{mol}}}{\text{Molecular mass}} = \frac{8314}{28} = 296.93 \text{ J/kg.K}$$

$$m = \frac{P_1 V_1}{RT_1} = \frac{1 \times 10^5 \times 0.03}{296.93 \times 290} = 0.0348 \text{ kg}$$

$$\text{Change in entropy, } (S_2 - S_1) = m \left[ C_p \ln \frac{T_2}{T_1} - R \ln \frac{P_2}{P_1} \right] \quad [\text{isothermal process} \rightarrow T_1 = T_2]$$

$$(S_2 - S_1) = -mR \ln \frac{P_2}{P_1}$$

$$= -0.0348 \times 296.93 \ln \frac{4}{1}$$

$$= -14.324 \text{ J/K}$$

23. (c)

$$\text{Work done in 2.5 hrs} = VIT$$

$$= \frac{12 \times 8 \times 2.5}{1000} \times 3600 = 864 \text{ kJ}$$

Non-flow energy equation,

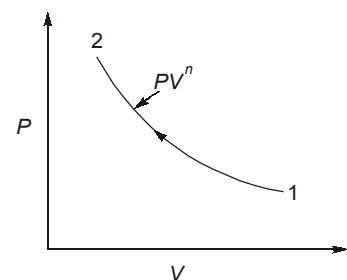
$$\delta Q = \delta W + dU$$

$$\delta Q = 864 - 1250 = -386 \text{ kJ}$$

24. (a)

$$\left(\frac{P_2}{P_1}\right) = \left(\frac{T_2}{T_1}\right)^{\frac{n}{n-1}}$$

$$\frac{T_2}{T_1} = \left(\frac{P_2}{P_1}\right)^{\frac{n-1}{n}}$$



$$\ln\left(\frac{T_2}{T_1}\right) = \frac{n-1}{n} \ln\left(\frac{P_2}{P_1}\right)$$

$$\frac{n-1}{n} = \frac{\ln(T_2/T_1)}{\ln(P_2/P_1)}$$

$$\frac{n-1}{n} = \frac{\ln(393/293)}{\ln(6/1.5)} = 0.2118$$

So,  $n = 1.2687$

25. (d)

We know efficiency of Carnot engine operating between temperature limits  $T_H$  and  $T_L$  is

$$\eta = 1 - \frac{T_L}{T_H}$$

$$\therefore 2\left(1 - \frac{T_L}{T_H}\right) = 1 - \frac{T_L}{T_H'}$$

$$\therefore \text{On solving, } T_H' = \frac{T_L T_H}{2T_L - T_H}$$

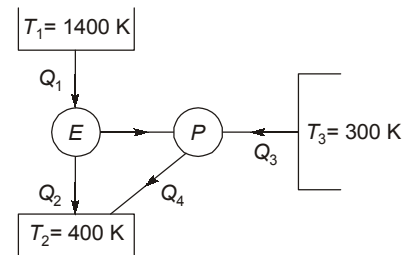
26. (c)

For the reversible engine:

$$\frac{Q_1}{Q_2} = \frac{T_1}{T_2}$$

For the reversible pump:

$$\frac{Q_3}{Q_4} = \frac{T_3}{T_4}$$



Since, the entire output of the heat engine is utilized to drive the pump,

$$Q_1 - Q_2 = Q_4 - Q_3$$

$$Q_1 - \frac{T_2}{T_1} Q_1 = Q_4 - \frac{T_3}{T_4} Q_4$$

$$\frac{Q_1}{Q_4} = \frac{T_1}{T_4} \left( \frac{T_4 - T_3}{T_1 - T_2} \right)$$

$$\frac{Q_1}{Q_4} = \frac{1400}{400} \left( \frac{400 - 300}{1400 - 400} \right) = 0.35$$

Heat supplied to the reservoir at 400 K,

$$100 = Q_2 + Q_4$$

$$100 = Q_1 \frac{T_2}{T_1} + \frac{Q_1}{0.35}$$

$$100 = Q_1 \frac{400}{1400} + \frac{Q_1}{0.35} = 3.14 Q_1$$

$\therefore$  Energy taken from the reservoir at 1400 K,

$$Q_1 = \frac{100}{3.14} = 31.82 \text{ kJ/s}$$

27. (d)

From the principle of energy conservation,

heat given by hot gases = heat gained by water

$$(mC_p \Delta t)_{\text{gas}} = (mC_p \Delta t)_{\text{water}} = \text{heat transfer } (Q)$$

$$m_g \times 1.08 \times (180 - 80) = 0.25 \times 4.186 \times (60 - 30) = Q$$

$$\therefore m_g = 0.2907 \text{ kg and } Q = 31.395 \text{ kJ/s}$$

$$(ds)_{\text{gases}} = m_g C_{pg} \ln \frac{T_{2g}}{T_{1g}}$$

$$= 0.2907 \times 1.08 \times \ln \left( \frac{80 + 273}{180 + 273} \right) = -0.078 \text{ kJ/K}$$

$$(ds)_{\text{water}} = m_w C_{pw} \ln \frac{T_{2w}}{T_{1w}} = 0.25 \times 4.186 \times \ln \left( \frac{60 + 273}{30 + 273} \right)$$

$$= 0.098 \text{ kJ/K}$$

∴ Entropy production as a result of heat transfer is

$$\begin{aligned} (ds)_{\text{net}} &= (ds)_{\text{gases}} + (ds)_{\text{water}} \\ &= -0.078 + 0.098 = 0.02 \text{ kJ/K} \end{aligned}$$

Rate of increase in unavailable energy (or loss of available energy)

$$\begin{aligned} &= T_0 (ds)_{\text{net}} \\ &= (273 + 27) \times 0.02 \\ &= 6 \text{ kJ/s} \end{aligned}$$

28. (d)

$$\begin{aligned} (\text{COP})_{\text{heat pump}} &= \frac{T_H}{T_H - T_L} \\ &= \frac{22 + 273}{20} = 14.75 \end{aligned}$$

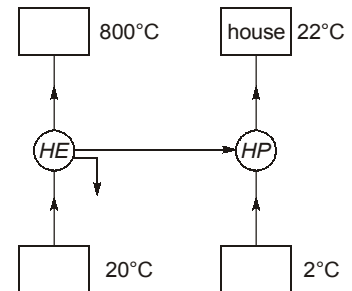
$$\begin{aligned} \text{Power input to the heat pump} &= \frac{Q_H}{\text{COP}} \\ &= \frac{62000}{14.75} = 4203 \text{ kJ/h} \end{aligned}$$

$$(W_{\text{input}})_{\text{HP}} = \frac{1}{2} W_{\text{HE}}$$

$$W_{\text{HE}} = 8406 \text{ kJ/h}$$

$$\eta_{\text{HE}} = 1 - \frac{T_L}{T_H} = 1 - \frac{293}{1073} = 0.727$$

$$Q_S \text{ for the heat engine} = \frac{W_{\text{HE}}}{\eta_{\text{th}}} = \frac{8406}{0.727} = 11560 \text{ kJ/h}$$



$$\text{Heat supply to the heat engine} = \frac{11560}{3600} = 3.21 \text{ kW}$$

29. (a)

$$\begin{aligned} (\Delta S)_{\text{universe}} &= (\Delta S)_{\text{system}} + (\Delta S)_{\text{surrounding}} \\ &= mC \ln \frac{T_2}{T_1} - \frac{Q}{T} \\ &= 1 \times 4.18 \ln \frac{363}{273} - \frac{1 \times 4.18(90 - 0)}{363} \\ &= 1.191 - 1.036 \\ &= 0.155 \text{ kJ/K} \end{aligned}$$

30. (d)

In the absence of any other information regarding  $v_g$  and  $v_f$ , the Clausius Clapeyron equation can be used to determine the saturation temperature corresponding to the given pressure.

$$\begin{aligned} \left( \frac{\partial P}{\partial T} \right)_{\text{sat}} &= \frac{h_{fg} \cdot P}{RT^2} \\ \Rightarrow \quad [\ln P]_{P_1}^{P_2} &= \frac{h_{fg}}{R} \left[ -\frac{1}{T} \right]_{T_1}^{T_2} \\ \Rightarrow \quad \ln \left( \frac{P_2}{P_1} \right) &= \frac{h_{fg}}{R} \left[ \frac{1}{T_1} - \frac{1}{T_2} \right] \\ \Rightarrow \quad \ln \frac{250}{101.325} &= \frac{2257}{8.314} \left[ \frac{1}{373} - \frac{1}{T_2} \right] \\ \Rightarrow \quad 0.903 &= 4886.45 \left[ \frac{1}{373} - \frac{1}{T_2} \right] \\ T_2 &= 400.6 \text{ K} \\ t_2 &= 127.6^\circ\text{C} \end{aligned}$$

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